Model of Diffusers / Permeators for Hydrogen Processing

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MODEL OF DIFFUSERS / PERMEATORS FOR HYDROGEN PROCESSING

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Palladium-silver (Pd-Ag) diffusers are mainstays of hydrogen processing. Diffusers separate hydrogen from inert species such as nitrogen, argon or helium. The tubing becomes permeable to hydrogen when heated to more than 250º C and a differential pressure is created across the membrane. The hydrogen diffuses better at higher temperatures. Experimental or experiential results have been the basis for determining or predicting a diffuser’s performance. However, the process can be mathematically modeled, and comparison to experimental or other operating data can be utilized to improve the fit of the model. A reliable model-based diffuser system design is the goal which will have impacts on tritium and hydrogen processing.

A computer model has been developed to solve the differential equations for diffusion given the operating boundary conditions. The model was compared to operating data for a low pressure diffuser system. The modeling approach and the results are presented in this paper.

I. INTRODUCTION

Palladium (or palladium-silver) diffusers, also called “permeators”, have been employed in tritium processing operations for many years to remove hydrogen isotopes (i.e., protium, deuterium, and tritium) from inert gas streams. Several new proposed projects (e.g., the ITER project or the Tritium Facilities Modernization and Consolidation at the Savannah River Site) require operating conditions beyond the range of past experience. In most cases, elaborate and expensive testing would have to be conducted to verify the modified apparatus configuration achieves its functional design.

The objective of the current work was to develop a mathematical model that would reliably predict the performance of a palladium diffuser for hydrogen removal. The model involves the diffuser design geometry and the basic process parameters.

II. MODEL DEVELOPMENT

II.A Murav’ev Model for Hydrogen Permeators

Murav’ev\(^1\) constructed a mathematical model to calculate and optimize gas flow rates in the hydrogen-permeable palladium alloy capillaries of permeators. Fig. 1 schematically shows the modeled membrane apparatus in which hydrogen is recovered from a gas mixture by hydrogen permeation through the membrane into capillary channel.

![Fig. 1. Schematic of the membrane apparatus with the permeate flow inside the membrane tube](image)

Since hydrogen flow in the capillaries is laminar with Reynolds number less 1000, the pressure drop across an elementary length can be described by the Hagen-Poiseuille equation:

\[
\frac{d}{d} = \frac{\lambda}{d} \frac{p}{u^2} \frac{dz}{2}
\]

The rate of hydrogen flow through the membrane and the pressure drop across the membrane are related by the
Sieverts law:

\[ dQ = \frac{\gamma}{\delta} (p_v^r - p_r^r) dF \]  \hspace{1cm} (2)

The exponent \( r \) accounts for the dissociation of the gas. For a non-dissociated gas, \( r = 1 \); for a diatomic dissociating gas, \( r = 0.5 \). Experimental data indicated \( 0.5 \leq r \leq 1 \).

Eqs. (1) and (2) can be rewritten in the dimensionless form:

\[ \frac{dw}{d\omega} = m_v \frac{s}{w} \]  \hspace{1cm} (3)

\[ \frac{ds}{d\omega} = -m_s (X^r - bw^r) \]  \hspace{1cm} (4)

where

\[ m_v = \frac{128 \pi p_a}{\mu T_o} \frac{L}{d^4} \frac{Q_o}{p_a^2} \]  \hspace{1cm} (5)

\[ m_s = \frac{\pi d L \rho \gamma p^*}{\delta Q_o} \]  \hspace{1cm} (6)

\[ X = \frac{p_v}{p_{vo}} \]  \hspace{1cm} (7)

\[ b = \left( \frac{p_a}{p_{vo}} \right)^r \]  \hspace{1cm} (8)

Murav’ev used the following dimensionless variables to derive Eqs. (3) and (4):

\[ w = p/p_a \quad s = Q/Q_o \quad \omega = z/L \quad h = Q/Q_p \]

\[ \alpha = Q_p/Q_o \quad \tau = Q_P/Q_o \]

Eqs. (3) and (4) are subjected to the boundary conditions

\[ \omega = 0: \quad s = \alpha \quad w = 1 \]  \hspace{1cm} (9)

\[ \omega = 1: \quad s = 0 \quad \left( \frac{dw}{d\omega} = 0 \right) \]  \hspace{1cm} (10)

The total material and hydrogen balance equations are provided for different flow directions.

For cocurrent flows:

\[ v + s = 1 \quad X = \frac{X_v - s}{1 - s} \]  \hspace{1cm} (11)

For countercurrent flows:

\[ v - s = 1 - \alpha \quad X = \frac{X_v - \alpha + s}{1 - \alpha + s} \]  \hspace{1cm} (12)

Murav’ev proposed an algorithm to numerically solving the set of Eqs. (3) and (4). The model capabilities to predict the performance of high-temperature hydrogen permeators were demonstrated, in which the effects of the flow resistance of the capillaries, the flow directions, and the variation of the hydrogen concentration in the gas feed were accounted for. The model enables an optimal design of the capillary geometry and the process parameters.

II.B ACM Software

The Aspen Custom Modeler (ACM) marketed by Aspen Technology, Inc. was selected as the modeling software in this work. The architecture of ACM makes it well suited to modeling and solving distributed parameters systems in which process variables vary with respect to spatial position as well as time described mathematically by partial differential equations (PDEs). ACM provides a number of built-in, well-known discretization methods that can be chosen for approximating partial derivatives and integrals. Complex ACM models have been successfully developed by researchers at the Savannah River National Laboratory (SRNL) in the past.\(^2\)

Murav’ev model was coded in ACM. The calculated results presented in Ref. 1 were accurately reproduced. In all calculations performed in the current paper, the length domain L was divided into 20 elements. The 4th-order central finite difference was the selected discretization method. This computational scheme was found adequate to handle the problems of interest.

II.C Modifications to Murav’ev Model

As shown in Fig. 1, the Murav’ev model was derived for hydrogen permeation through the membrane into the capillaries. Many hydrogen permeator systems in operation or being tested, however, have the reversed flow pattern. In those systems, the gas mixture flows in the membrane tubes, and hydrogen permeates through tubes into the shell side. A schematic of such membrane apparatuses is provided in Fig. 2.
Murav’ev model was thus modified to appropriately describe these membrane configurations. All governing equations, dimensionless variables and the boundary conditions are summarized below.

\[
\frac{dw}{d\omega} = m_v \frac{v_w}{w} \quad (13)
\]

\[
\frac{ds}{d\omega} = \mp m_o \left( w' \chi^r - b \right) \quad (14)
\]

where

\[
m_v = \frac{128}{\pi} \frac{p_o}{T_o} \mu \frac{L}{d^4} \frac{Q_o}{p_r} \quad (15)
\]

\[
m_s = \frac{\pi d L \rho p_f^r}{\delta Q_o} \quad (16)
\]

\[
X = \frac{p_r}{p} \quad (17)
\]

\[
b = \left( \frac{p_p}{p_r} \right)^r \quad (18)
\]

Dimensionless variables:

\[
w = p/p_r \quad s = Q/Q_o \quad \omega = z/L \quad h = Q/Q_p \quad \alpha = Q_p/Q_o \quad v = Q_v/Q_o
\]

Boundary conditions:

\[
\omega = 0: \quad s = \alpha \quad 0 \quad (\text{Cocurrent}) \quad 0 \quad (\text{Countercurrent})
\]

\[
w = 1
\]

\[
\omega = 1: \quad s = 0 \quad (\text{Cocurrent}) \quad \alpha \quad (\text{Countercurrent})
\]

The total material and hydrogen balance equations are the same as in Eqs. (11) and (12). Although having the gas feed in the tubes remarkably increases the Reynolds number of the tube flow, at the highest feed flow rates of interest the Reynolds number is less than 2000. Consequently, the Hagen-Poiseuille equation (Eq. 13) is still applicable.

**II.C.1 Effect of r**

Calculations were performed with two selected values of r, 0.5 and 1. In each calculation, permeability \( \gamma \) was varied to achieve the best result in comparison to the data obtained from experimental testing of a low-pressure diffuser conducted at the Los Alamos National Laboratory (LANL). In these tests, hydrogen was removed from the feed, a mixture of nitrogen and hydrogen. The results in Fig. 3 confirm hydrogen permeation would be best modeled with \( r = 0.5 \). Note all flow rates are reduced to normal conditions (i.e., \( p_o = 1.01325 \text{ bar}, T_o = 273 \text{ K} \)).

**II.C.2 Permeability**

The LANL testing indicated permeability correlates well with the tube-side flow rate, thus suggesting a substantial mass transfer resistance in the gas phase.
Generally, the tube-side flow rate should not affect the permeation rate in the palladium or the mechanisms on the shell-side of the palladium. However, the flow in the tubes is laminar, so the boundary layer mass transfer resistance for the diffusion of hydrogen to the palladium surface can be appreciable. Such a laminar boundary layer mass transfer resistance would be expected to be strongly dependent on the gas velocity within the tubes.

Data at 0.00018 m$^3$/hr (3 cm$^3$/min) per-tube hydrogen feed flow were regressed to correlate permeability for two ranges of the per-tube nitrogen feed flow rates (i.e., below and above 0.02 m$^3$/hr). The correlations were then adjusted for all hydrogen flow rates.

### III. RESULTS

The modified Murav’ev model for countercurrent flows was applied to compute the low-pressure diffuser performance at various hydrogen feed flow rates. The calculated data cover a large range of feed flow rates. The comparison of the calculated hydrogen retentate composition (expressed in ppm) with test data is in Fig. 4. Overall, the modeling results are in agreement with experimental data. The agreement is less favorable at high hydrogen and nitrogen flow rates. However, the testing exhibited considerable experimental error in measured retentate partial pressures (therewith, hydrogen retentate composition) at high nitrogen flow rates which would have an adverse impact on data comparison.

![Fig. 4. Comparison of calculated hydrogen retentate composition with test data](image)

### IV. CONCLUSIONS

A hydrogen permeator model developed by Murav’ev was successfully modified to adequately account for the flow configurations of a low-pressure diffuser. Correlations for permeability were provided for a large range of hydrogen and nitrogen feed flow rates. The diffuser performance data produced by the modified model are in agreement with test data except at high feed flow rates when large experimental error would make any comparison difficult.

### NOMENCLATURE

- \( d \) Inner diameter of the tube
- \( F \) Inner surface area of the tube
- \( L \) Tube length
- \( p \) Pressure in the tube
- \( p_a \) Hydrogen pressure at the open end of tube
- \( p_o \) Pressure at normal condition
- \( p_p \) Permeate pressure
- \( p_r \) Retentate pressure
- \( p_v \) Partial pressure of hydrogen in the gas mixture
- \( p_{vo} \) Pressure of the gas feed
- \( Q \) Flow rate of hydrogen
- \( Q_o \) Feed flow rate
- \( Q_p \) Permeate flow rate
- \( Q_r \) Retentate flow rate
- \( Q_v \) Flow rate of gas mixture
- \( T \) Temperature
- \( T_o \) Temperature at normal condition
- \( u \) Cross-section averaged flow velocity in the tube
- \( X \) Hydrogen mole fraction in the gas mixture
- \( X_r \) Hydrogen mole fraction in the retentate
- \( X_v \) Hydrogen mole fraction in feed
- \( \gamma \) Permeability
- \( \delta \) Membrane thickness
- \( \lambda \) Flow resistant
- \( \mu \) Dynamic viscosity
- \( \rho \) Gas density in the tube

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