

Horizontal Air Bearing Experiment Number 1

T.L. Clauson

August 31, 1999

U.S. Department of Energy

Lawrence
Livermore
National
Laboratory

DISCLAIMER

This document was prepared as an account of work sponsored by an agency of the United States Government. Neither the United States Government nor the University of California nor any of their employees, makes any warranty, express or implied, or assumes any legal liability or responsibility for the accuracy, completeness, or usefulness of any information, apparatus, product, or process disclosed, or represents that its use would not infringe privately owned rights. Reference herein to any specific commercial product, process, or service by trade name, trademark, manufacturer, or otherwise, does not necessarily constitute or imply its endorsement, recommendation, or favoring by the United States Government or the University of California. The views and opinions of authors expressed herein do not necessarily state or reflect those of the United States Government or the University of California, and shall not be used for advertising or product endorsement purposes.

Work performed under the auspices of the U. S. Department of Energy by the University of California Lawrence Livermore National Laboratory under Contract W-7405-Eng-48.

This report has been reproduced
directly from the best available copy.

Available to DOE and DOE contractors from the
Office of Scientific and Technical Information
P.O. Box 62, Oak Ridge, TN 37831
Prices available from (423) 576-8401
<http://apollo.osti.gov/bridge/>

Available to the public from the
National Technical Information Service
U.S. Department of Commerce
5285 Port Royal Rd.,
Springfield, VA 22161
<http://www.ntis.gov/>

OR

Lawrence Livermore National Laboratory
Technical Information Department's Digital Library
<http://www.llnl.gov/tid/Library.html>

Horizontal Air Bearing Experiment Number 1

Introduction

The Horizontal Air Bearing Experiment #1 is a series of tests intended to further the understanding of rotational dynamics. A simple experimental assembly is rotated using the Horizontal Air Bearing and allowed to spin freely as the internal rotational damping is measured. The low friction of the bearing effectively isolates the test assembly, allowing the internal damping of the test object to be evaluated.

The experimental assembly is composed of an aluminum ball within a spherical cavity. A flanged pipe section and an auxiliary adapter plate secure the assembly to the Air Bearing interface plate. Three aluminum balls are interchanged to vary test parameters. The aluminum balls are free to move independently as the entire assembly rotates. The aluminum balls vary in diameter and/or surface finish. While the diameter and surface finish is varied, the space between the ball and socket is dry. To examine the effect of viscosity, the space is filled with a lubricant while the ball diameter and surface finish is held constant.

Data Presentation and Reduction

The Horizontal Air Bearing has the capability to permit balancing the rotor and payload both statically and dynamically. In addition, it has a built in torsion pendulum so the moment of inertia of the rotating mass can be measured and used to calculate the decelerating torque and payload moment of inertia.

During a damping test, the rotor is spun up to a predetermined rotational speed. Once the desired speed is reached, spin up is stopped and time data is taken. The collected data consists of the direction of rotation, start speed, stop speed, moment of inertia, period and elapsed time. Damping torque is calculated from the relationship

$$\text{Torque} = (\text{MOI}) \times (\text{deceleration}).$$

Damping measurements are performed at least three times in all configurations. The test assembly is first spun as a rigid body by immobilizing the aluminum ball within the cavity. Next the aluminum ball is freed and the damping measurement performed. The aluminum ball is changed to permit variation in configuration. The cavity between the ball and socket is filled with a lubricant for the final damping measurements. The configurations examined are presented in Table 1.

<i>Test I.D.</i>	<i>Experiment Variable</i>	<i>Environment</i>
Configuration 1a.	Ball, 0.008 gap - 125 surface finish	Shimmed, Dry
Configuration 1b.	Ball, 0.008 gap - 125 surface finish	Free, Dry
Configuration 2a.	Ball, 0.016 gap - 63 surface finish	Shimmed, Dry
Configuration 2b.	Ball, 0.016 gap - 63 surface finish	Free, Dry
Configuration 3a.	Ball, 0.008 gap - 63 surface finish	Shimmed, Dry
Configuration 3b.	Ball, 0.008 gap - 63 surface finish	Free, Dry
Configuration 3c.	Ball, 0.008 gap - 63 surface finish	Lubricant (Velocite #10)*
Configuration 3d.	Ball, 0.008 gap - 63 surface finish	Lubricant (Velocite #6)**

*Mobil Oil Company, viscosity=37.75 cP @25°C

**Mobil Oil Company, viscosity=12.5 cP @25°C

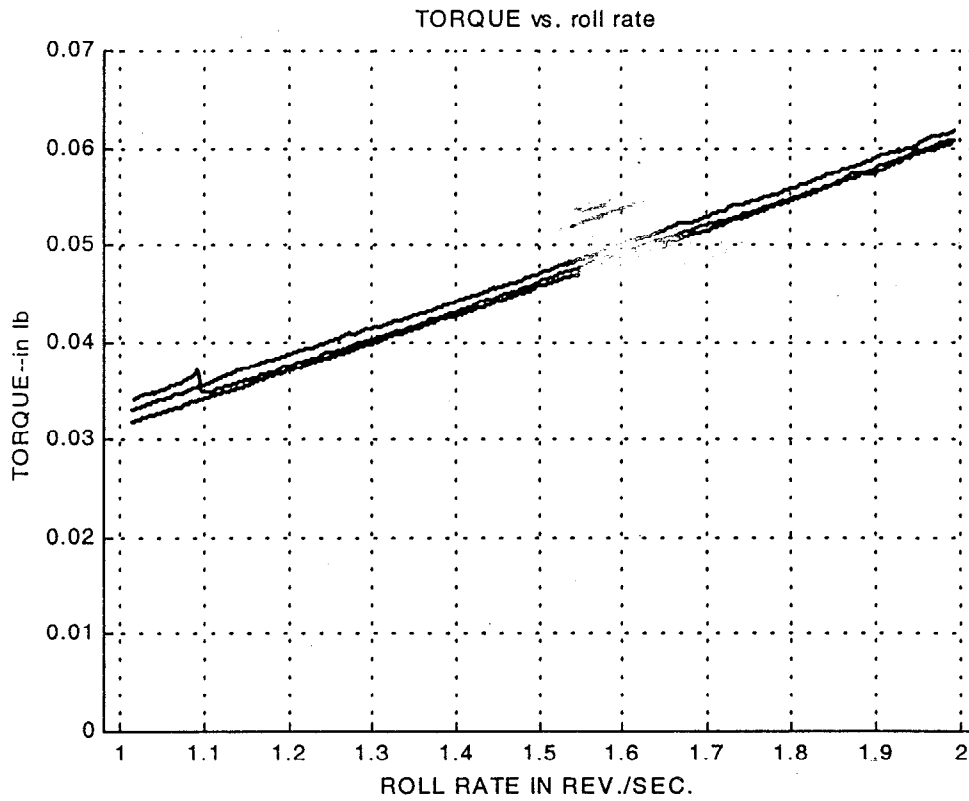


Figure 1. Configuration 1a. Spin-down tests 1, 2, and 3.

Description

Gap: .008"

Finish: 125 μ

Shimmed, Dry

Configuration 1a, spin-down tests 1, 2, and 3 were conducted consecutively, without disturbing the set-up between tests.

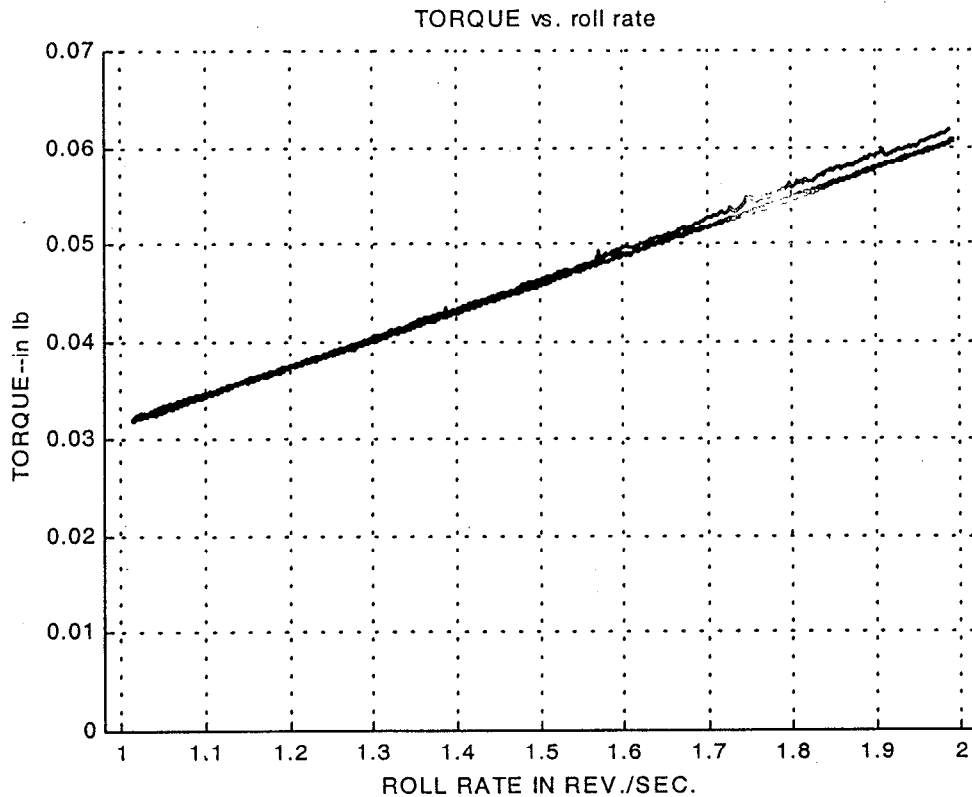


Figure 2. Configuration 1a, Spin down tests 4, 5, and 6.

Description

Gap: .008"

Finish: 125 μ

Shimmed, Dry

Configuration 1a, spin-down tests 1, 2, and 3 were conducted consecutively, without disturbing the set-up between tests. After spin-down test number three, the test assembly was taken apart, and the ball removed. The assembly was then reassembled. Without further changes, such as static balancing, POI or MOI measurement, spin-down test 4 was performed. The test assembly was again taken apart between tests 4 and 5, and 5 and 6.

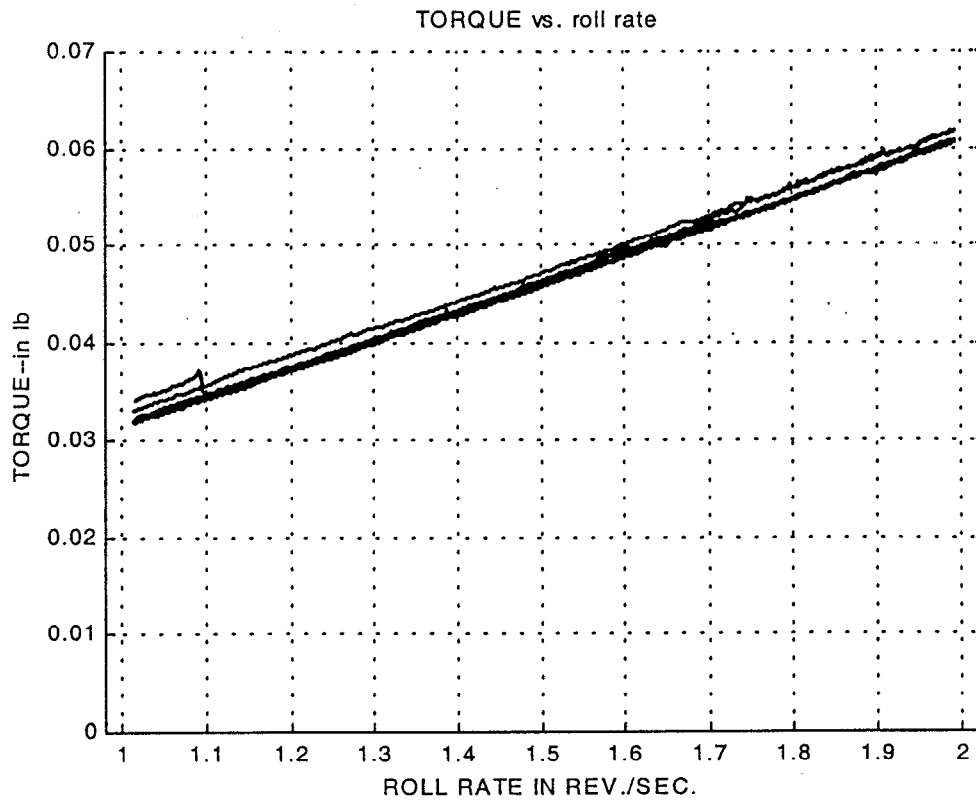


Figure 3. Configuration 1a. Spin-down tests 1 - 6.

Description

Gap: .008"
Finish: 125 μ
Shimmed, Dry

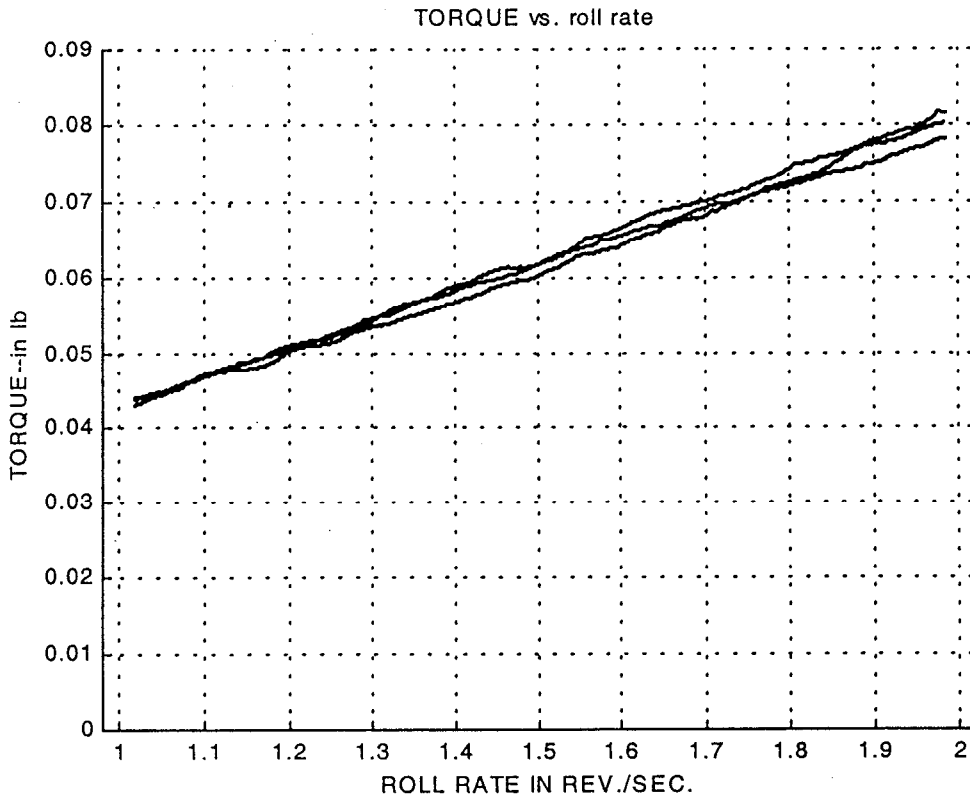


Figure 4. Configuration 1b. Spin-Down tests 1 – 3.

Description

Gap: .008"
 Finish: 125 μ
 Loose, Dry

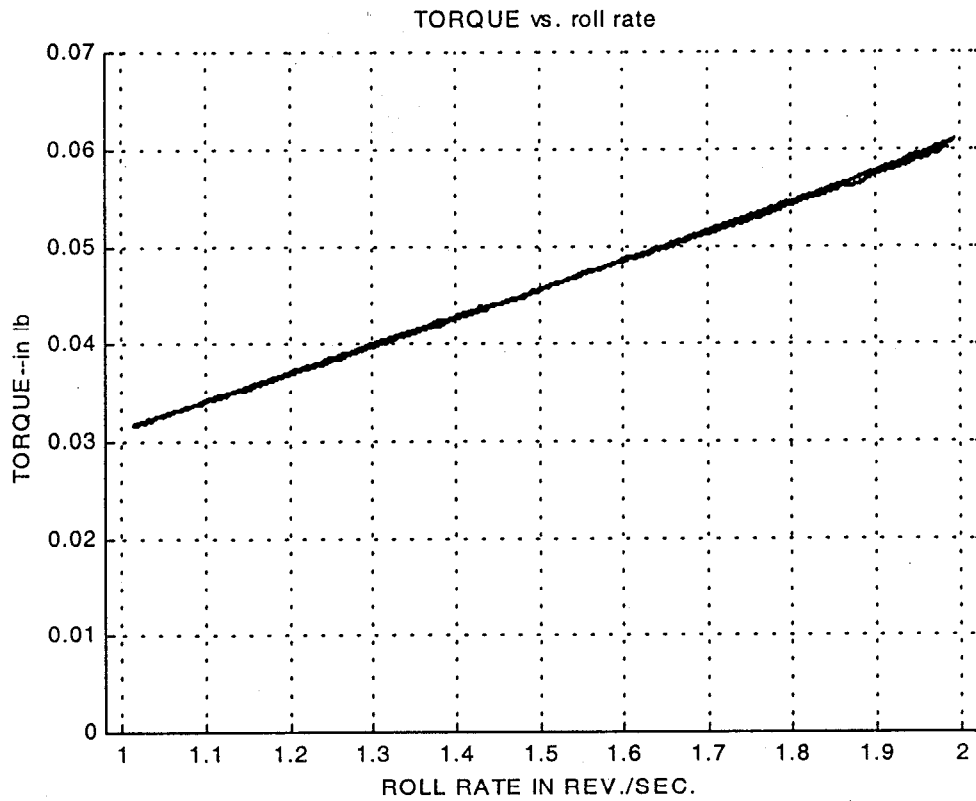


Figure 5. Configuration 2a, Spin-down tests 3, 6, and 7.

Description

Gap: .016"
Finish: 63 μ
Shimmed, Dry

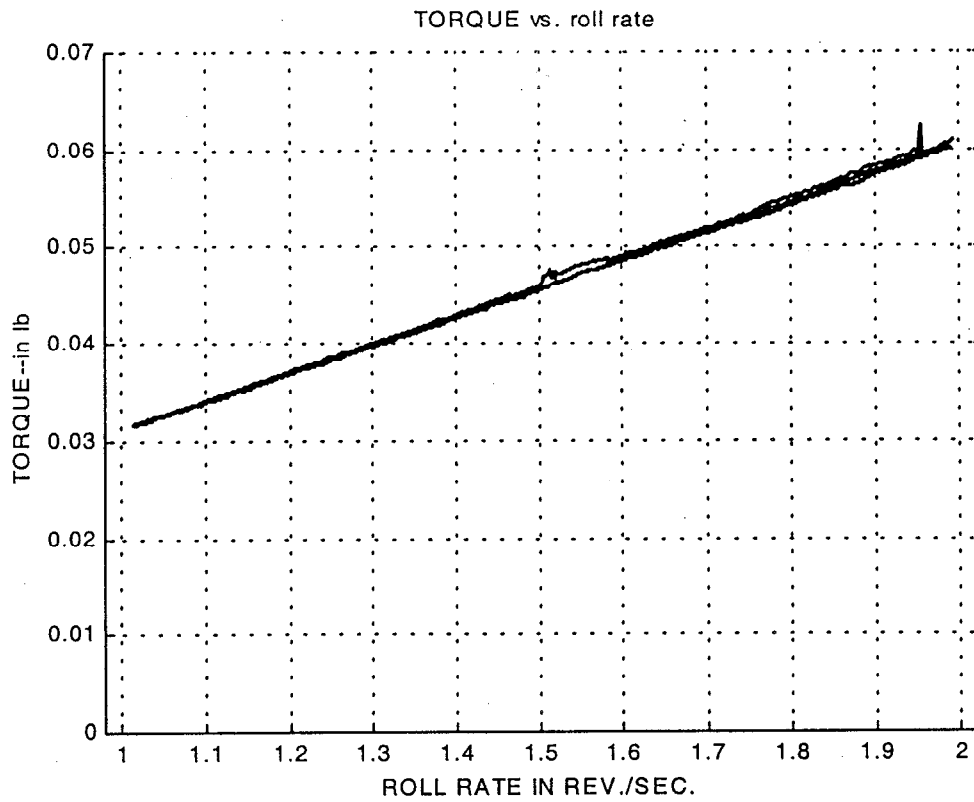


Figure 6. Configuration 2a. Spin-Down tests 5, 6, and 7.

Description

Gap: .016"

Finish: 63 μ

Shimmed, Dry

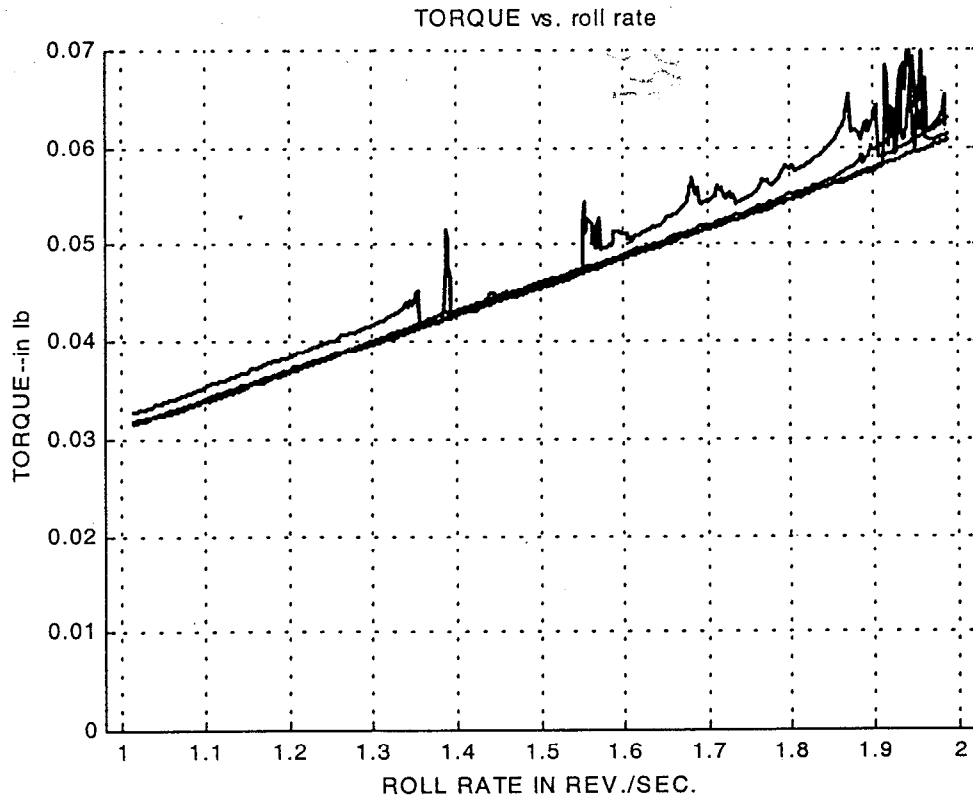


Figure 7. Configuration 2a. Spin-down tests 1 - 4.

Description

Gap: .016"

Finish: 63 μ

Shimmed, Dry

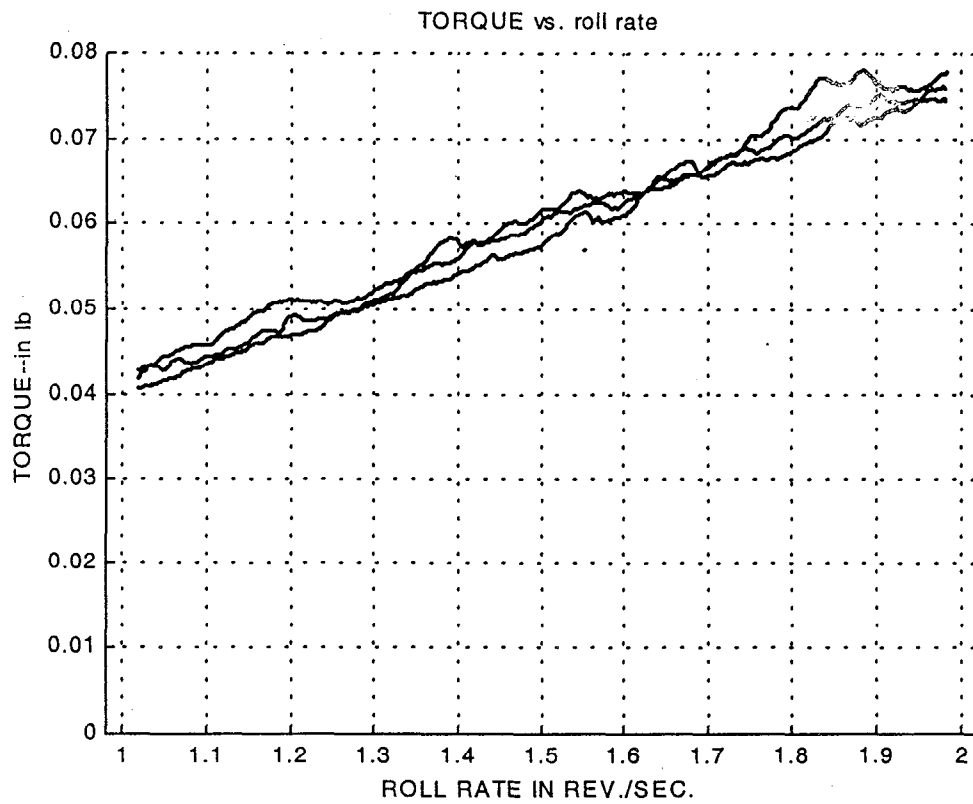


Figure 8. Configuration 2b. Spin-Down tests 1 - 3.

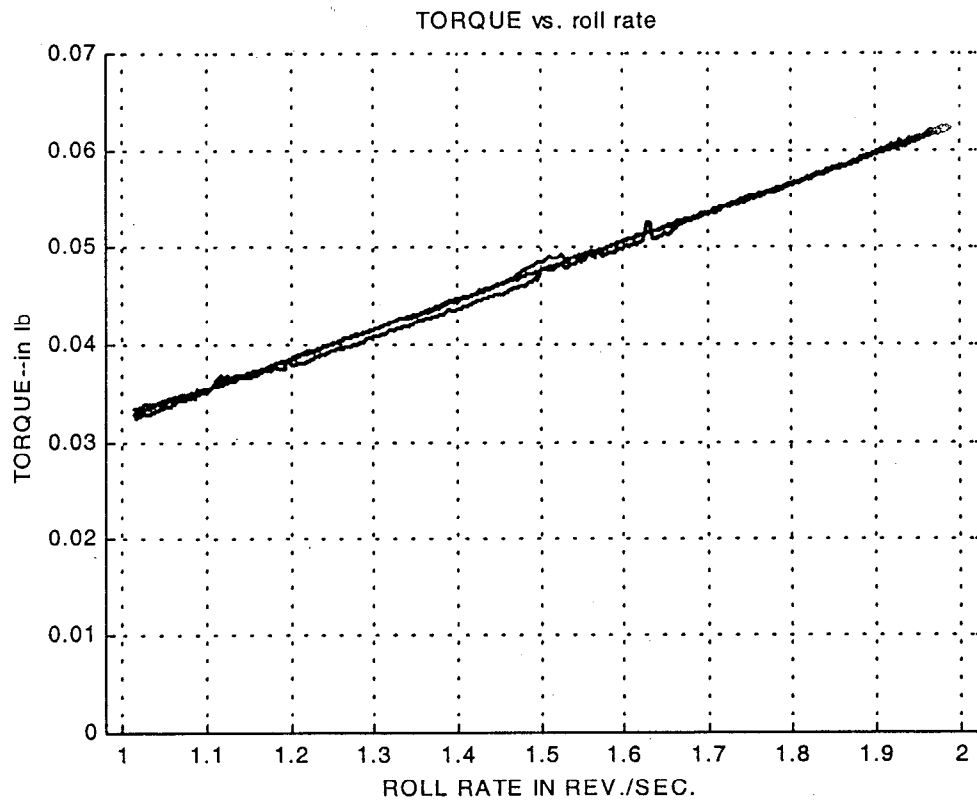


Figure 9. Configuration 3a. Spin-Down tests 1, 4, and 5.

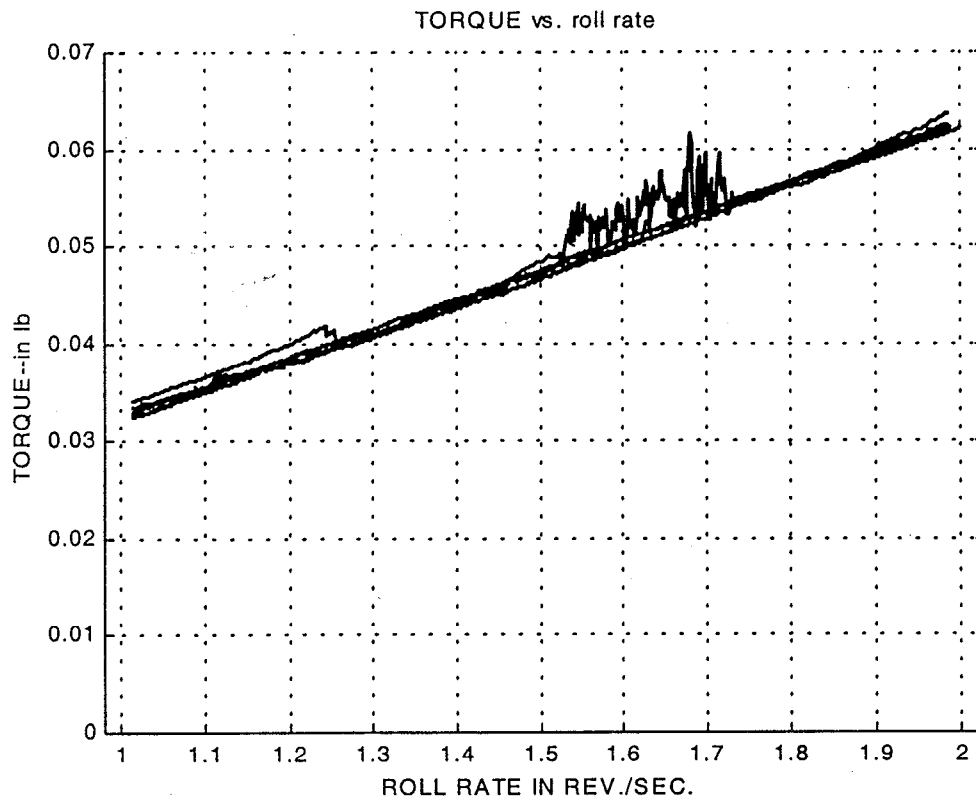


Figure 10. Configuration 3a. Spin-Down tests 1 - 5.

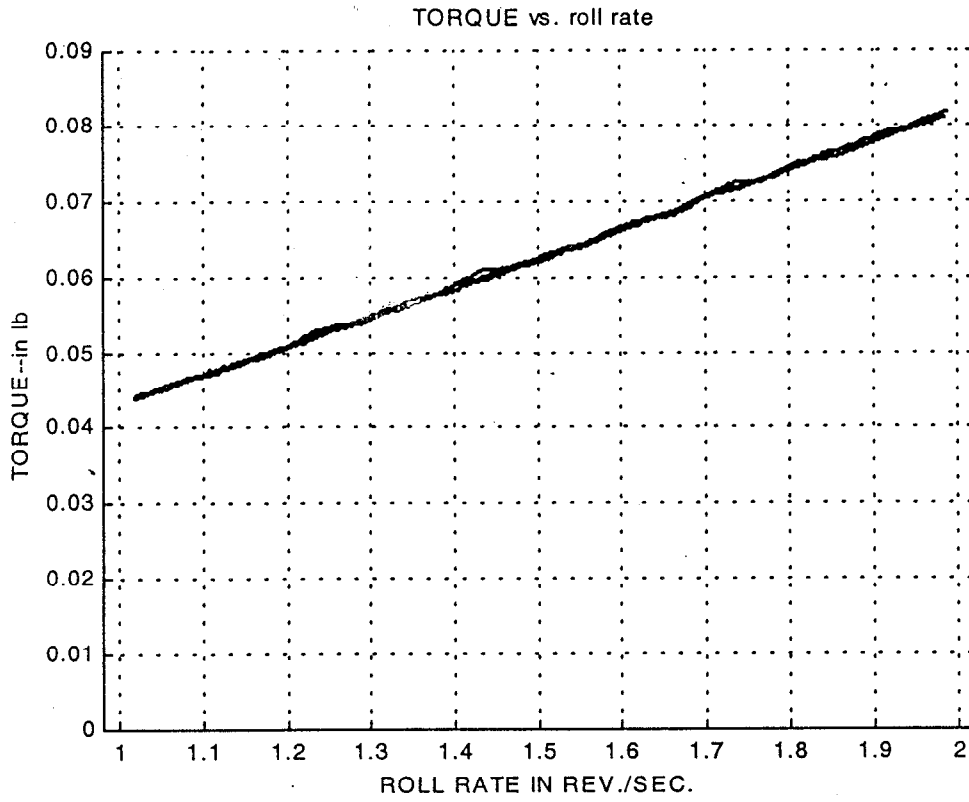


Figure 11. Configuration 3b. Spin-Down tests 1 - 3.

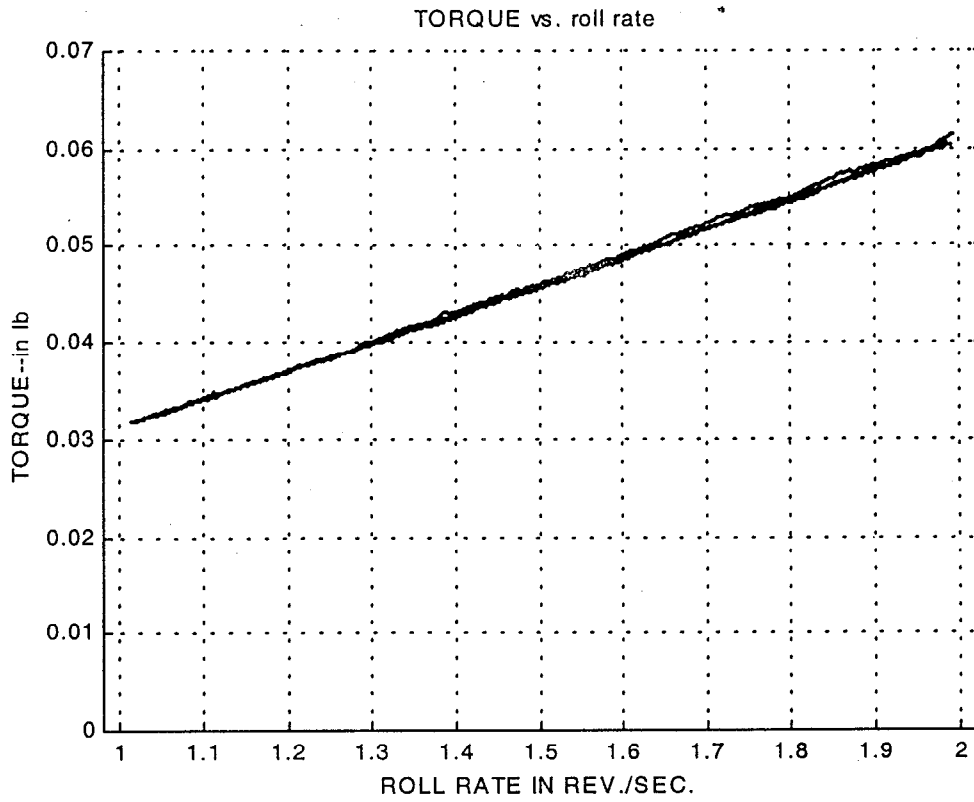


Figure 12. Configuration 3c. Spin-Down tests 1 - 3.

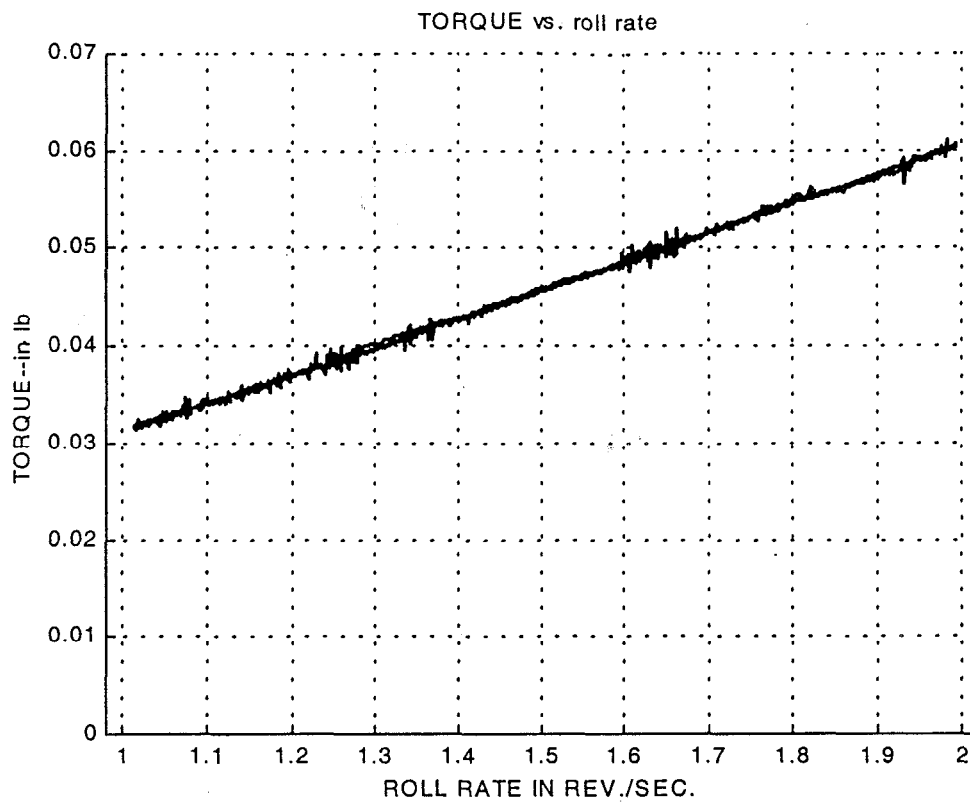
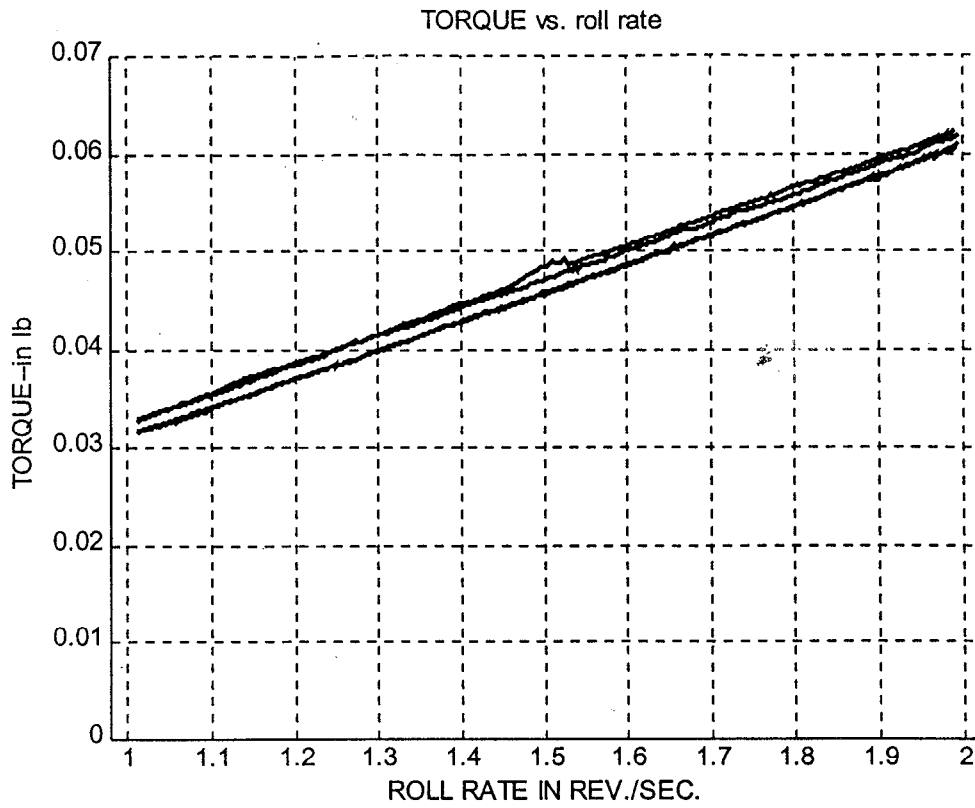
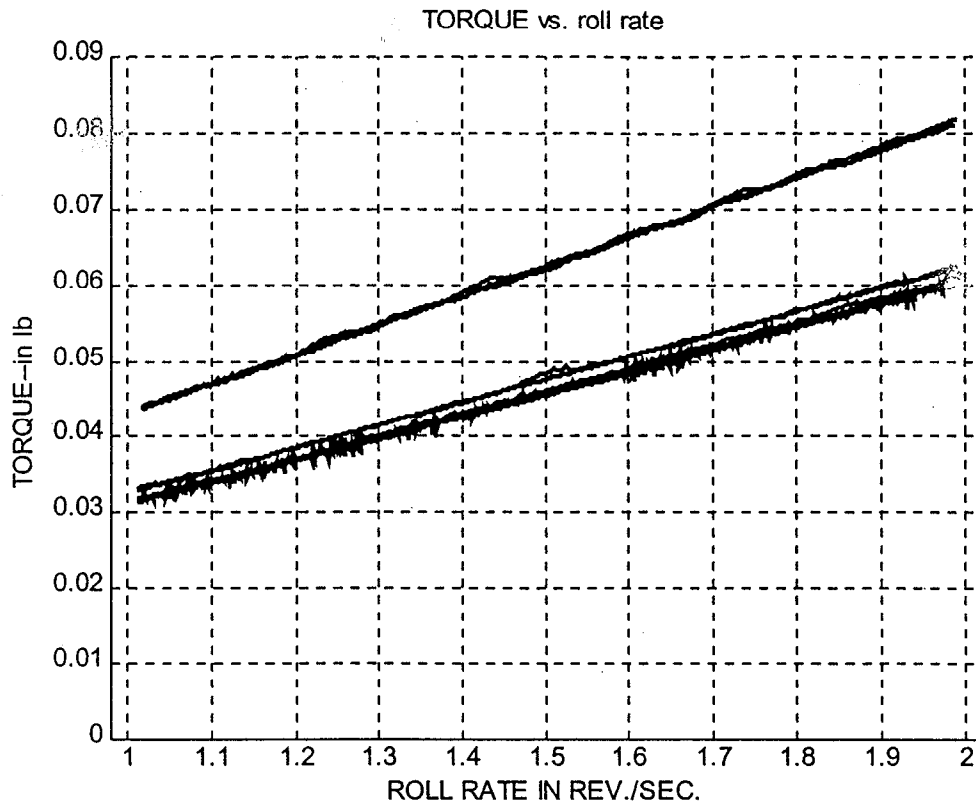


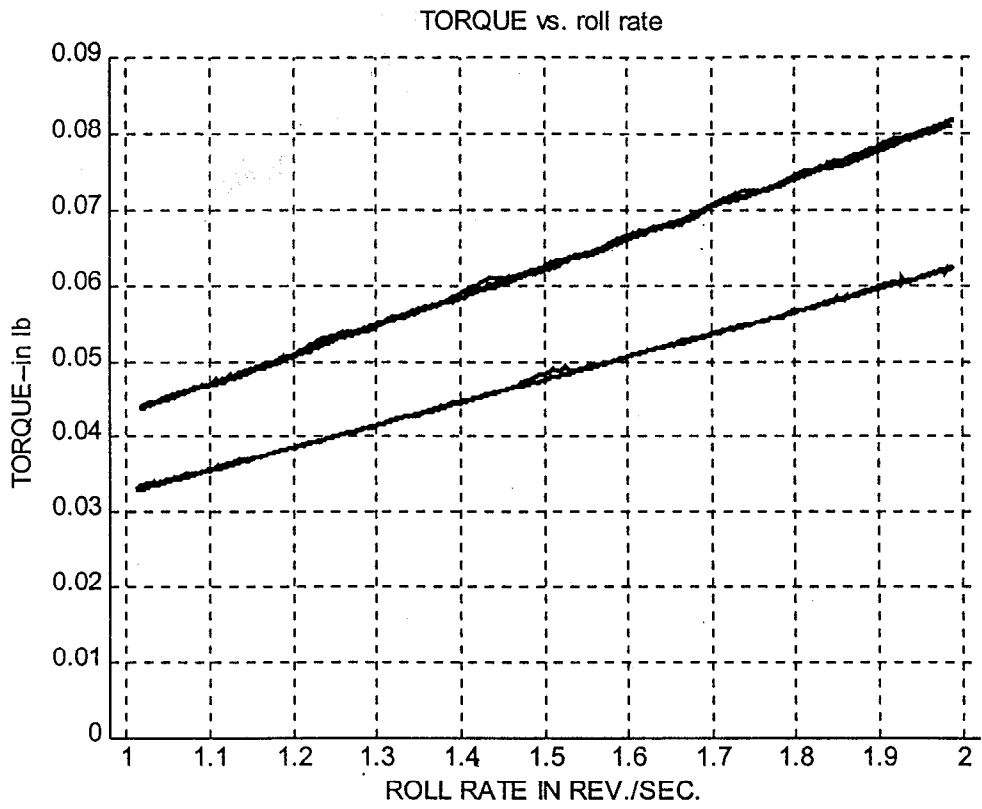
Figure 13. Configuration 3d. Spin-Down tests 1 - 3.



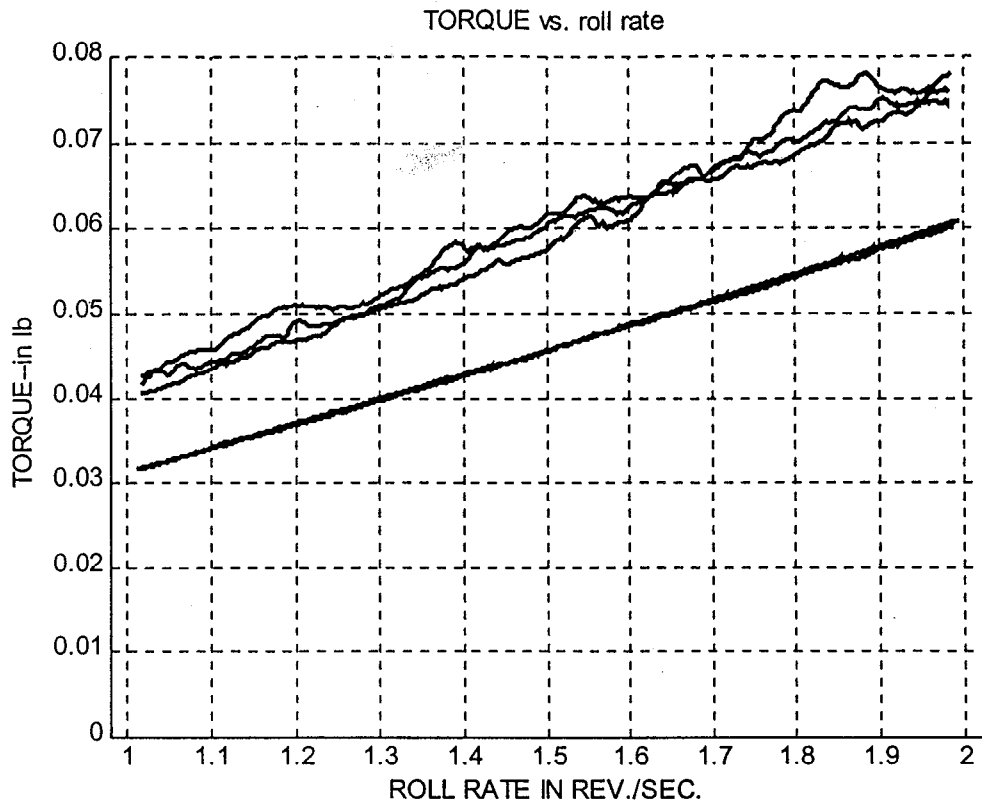
Configuration 1a, 2a, and 3a (1a is in upper and lower group. 2a is in lower group and 3a is in upper group)



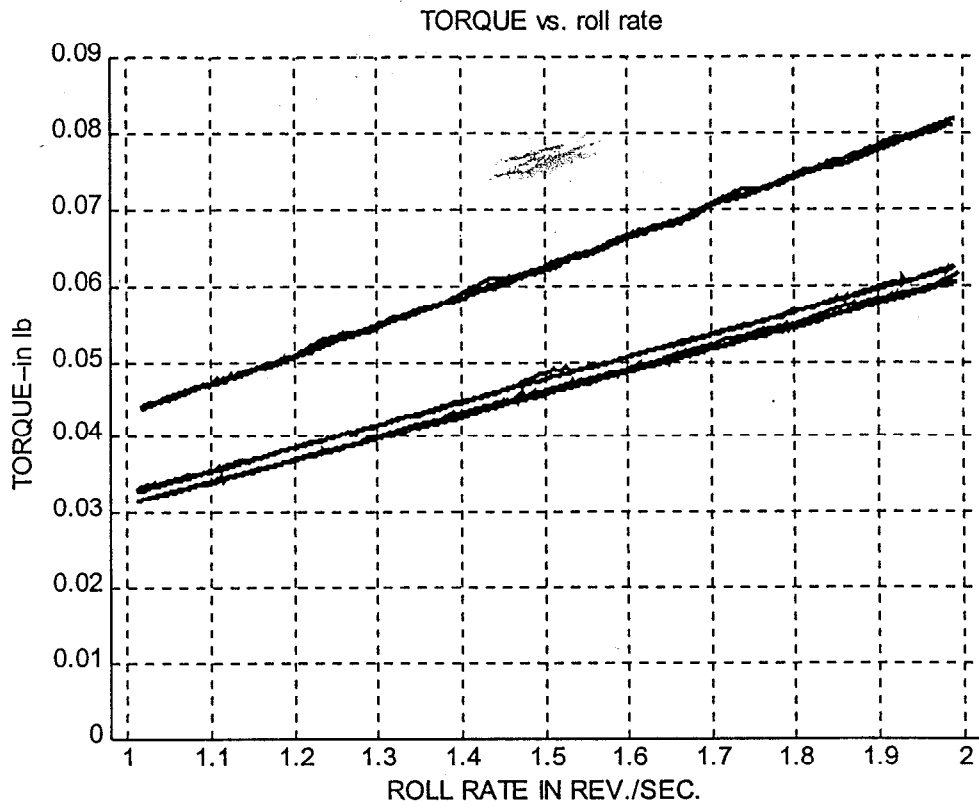
Configuration 3a (center group), 3b (upper group), 3c and 3d (lower group)



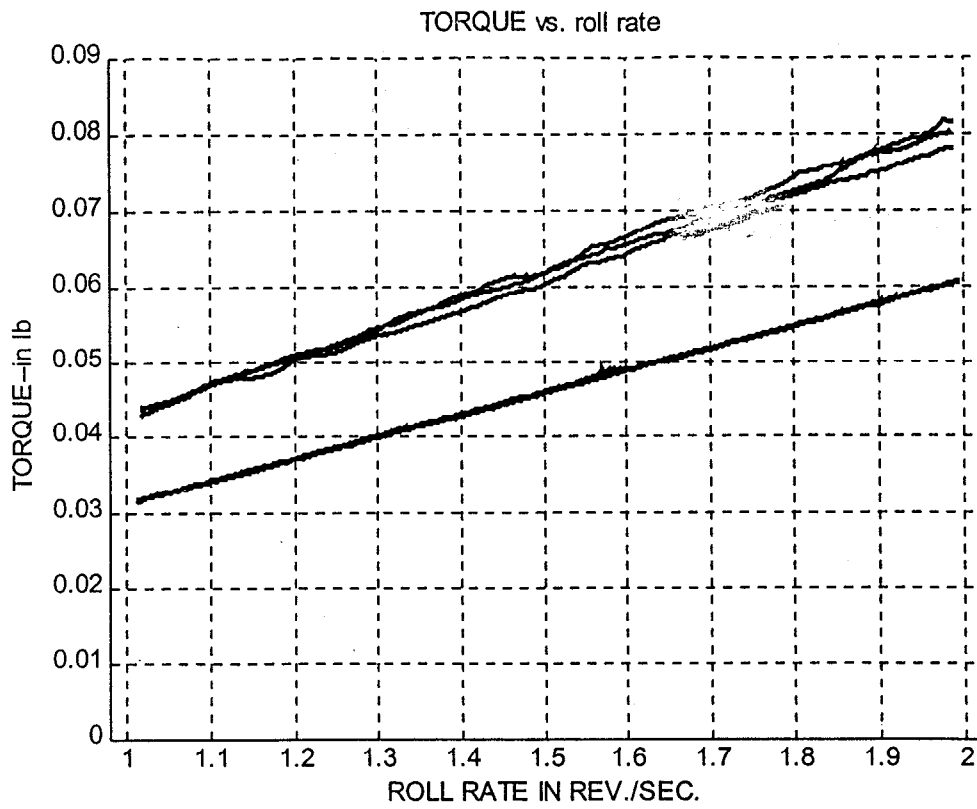
Configuration 3a (lower group). Configuration 3b (upper group)



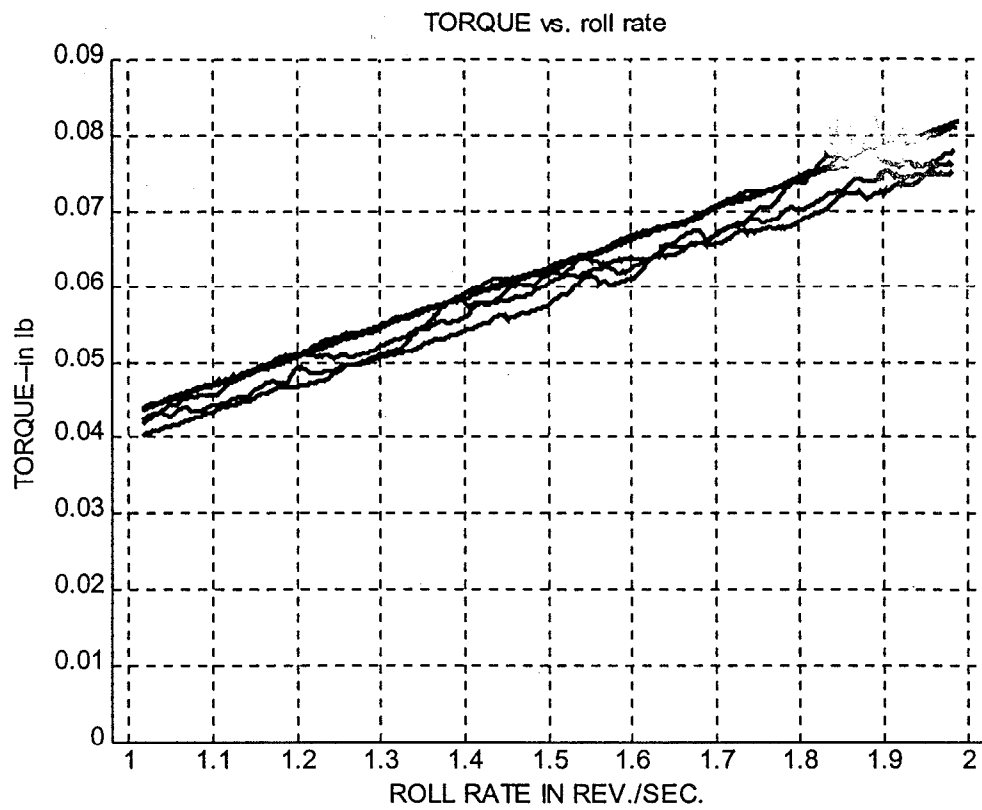
Configuration 2a, tests 5, 6, and 7 (lower group). Configuration 2b (upper group)



Configuration 3a (center group), 3b (upper group), 3c (lower group)



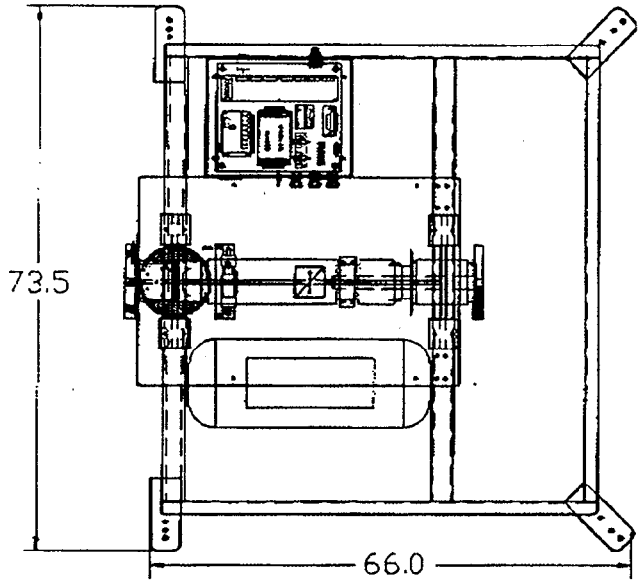
Configuration 1a, spin-down tests 1, 5, and 6 (lower grouping). Configuration 1b (upper grouping)



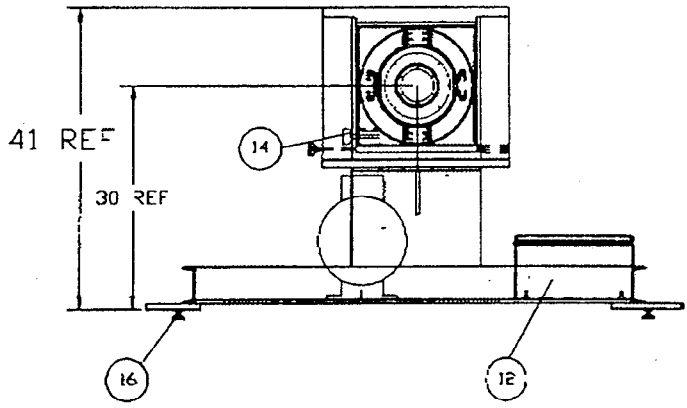
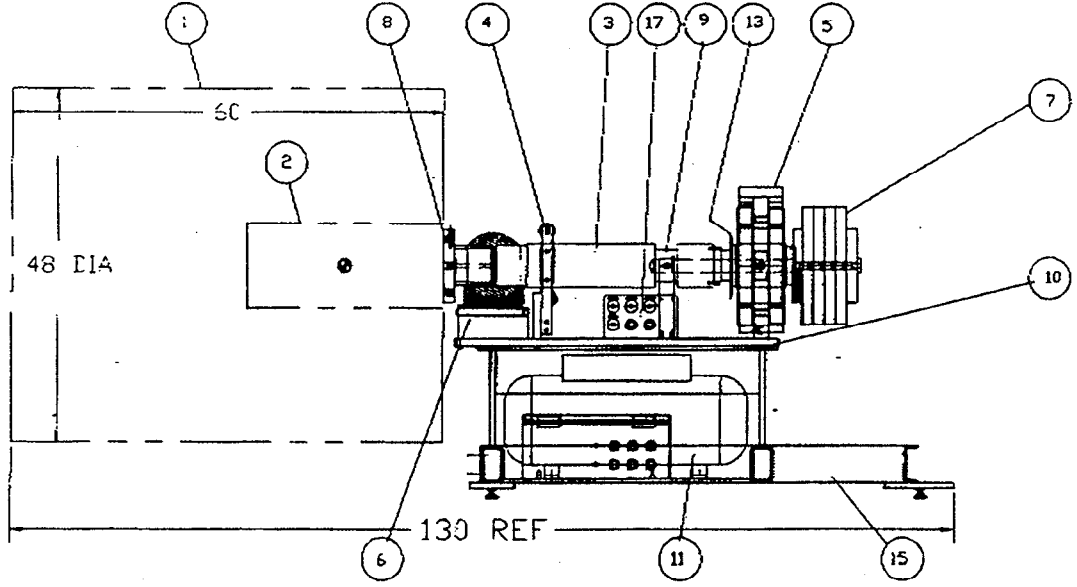
Configurations 3b and 2b.

2601010

REV	DATE	DR	CHK



DET	DESCRIPTION	PART DR DWG #
1	MAX TEST PART ENVELOPE	N/A
2	TEST PART	N/A
3	ROTOR ASSEMBLY	2278025
4	BRAKE ASSEMBLY	2278059
5	MOMENT BEARING ASSEMBLY	2278058
6	SPHERICAL BEARING CUP ASSEMBLY	2278060
7	COUNTERWEIGHT ASSEMBLY	2278025
8	INTERFACE PLATE	2278019
9	MOI ASSEMBLY	2278062
10	BASE PLATE	2278043
11	RESERVE AIR (GAS) TANK	P00620
12	ELECTRONICS ENCLOSURE	F00510
13	OPTRON ASSEMBLY	2278055
14	POI LOAD CELL ASSY	2278058
15	MAIN FRAME	2278037
16	LEVELING FOOT	LD0590
17	PNEUMATIC CONTROL PANEL	2278073



SPACE ELECTRONICS, INC. BERLIN, CT. GAS BEARING TEST INSTRUMENT ASSEMBLY	DR. BY:	DISK:
	DATE:	
	SCALE:	DWG. NO. 2601010