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# Enhancing the Use of Coals by Gas Reburning - Sorbent Injection - Volume 5 - Guideline Manual

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#### **ABSTRACT**

The purpose of the Guideline Manual is to provide recommendations for the application of combined gas reburning-sorbent injection (GR-SI) technologies to pre-NSPS boilers. The manual includes design recommendations, performance predictions, economic projections and comparisons with competing technologies. The report also includes an assessment of boiler impacts. Two full-scale demonstrations of gas reburning-sorbent injection form the basis of the Guideline Manual.

Under the U.S. Department of Energy's Clean Coal Technology Program (Round 1), a project was completed to demonstrate control of boiler emissions that comprise acid rain precursors, specifically oxides of nitrogen (NO<sub>x</sub>) and sulfur dioxide (SO<sub>2</sub>). Other project sponsors were the Gas Research Institute and the Illinois State Department of Commerce and Community Affairs.

The project involved demonstrating the combined use of Gas Reburning and Sorbent Injection (GR-SI) to assess the air emissions reduction potential of these technologies.. Three potential coal-fired utility boiler host sites were evaluated: Illinois Power's tangentially-fired 71 MWe (net) Hennepin Unit #1, City Water Light and Power's cyclone-fired 33 MWe (gross) Lakeside Unit #7, and Central Illinois Light Company's wall-fired 117 MWe (net) Edwards Unit #1. Commercial demonstrations were completed on the Hennepin and Lakeside Units. The Edwards Unit was removed from consideration for a site demonstration due to retrofit cost considerations.

Gas Reburning (GR) controls air emissions of  $NO_x$ . Natural gas is introduced into the furnace hot flue gas creating a reducing reburning zone to convert  $NO_x$  to diatomic nitrogen ( $N_2$ ). Overfire air is injected into the furnace above the reburning zone to complete the combustion of the reducing (fuel) gases created in the reburning zone. Sorbent Injection (SI) consists of the injection of dry, calcium-based sorbents into furnace hot flue gas to achieve  $SO_2$  capture.

At each site where the technologies were to be demonstrated, performance goals were set to achieve air emission reductions of 60 percent for  $NO_x$  and 50 percent for  $SO_2$ . These performance goals were exceeded during long term demonstration testing. For the tangentially fired unit,  $NO_x$  emissions were reduced by 67.2% and  $SO_2$  emissions by 52.6%. For the cyclone-fired unit,  $NO_x$  emissions were reduced by 62.9% and  $SO_2$  emissions by 57.9%.

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#### LIST OF ABBREVIATIONS

CAAA Clean Air Act Amendments
CCT Clean Coal Technology
CRT Cathode Ray Tube

CWLP City, Water, Light and Power City of Springfield, IL. EER Energy and Environmental Research Corporation

EPA Environmental Protection Agency

ESP Electrostatic Precipitator FGD Flue Gas Desulfurization FGR Flue Gas Recirculation

GR Gas Reburning

GR-SI Gas Reburning-Sorbent Injection

HAR Heat Absorption Ratio
HVT High Velocity Temperature

NFPA National Fire Protection Association NSPS New Source Performance Standards

OFA Overfire Air

OTR Ozone Transport Region

PFR Plug Flow Reactor PRB Powder River Basin

PSD Prevention of Significant Deterioration
RACT Reasonable Available Control Technology

SCR Selective Catalytic Reduction

SI Sorbent Injection

SNCR Selective Non-Catalytic Reduction

SR Air:fuel Stoichiometric Ratio

WSR Well Stirred Reactor

# LIST OF UNITS

acfm Actual Cubic Foot per Minute

Btu British Thermal Unit °F Degree Fahrenheit

ft³ Cubic Foot gal Gallon in Inch kW Kilowatt

kWe Kilowatt Electric kWh Kilowatt Hour

lb Pound

lbs/hr Pound per Hour

MW Megawatt

MWe Megawatt Electric

psig Pound per Square Inch (Gauge)

ton/yr Ton per Year
W.C. Water Column
10<sup>6</sup> Btu Million Btu

" Inch
% Percent

#### **GLOSSARY OF TERMS**

C<sub>2</sub>H<sub>2</sub> Acetylene C<sub>2</sub>H<sub>4</sub> Ethylene

Ca(OH)<sub>2</sub> Calcium Hydroxide CH Hydrocarbon Radical CH<sub>2</sub> Hydrocarbon Radical

CH₄ Methane

CO Carbon Monoxide CO<sub>2</sub> Carbon Dioxide

Fe Iron

H<sub>2</sub> Hydrogen (Diatomic) H<sub>2</sub>S Hydrogen Sulfide

H<sub>2</sub>O Water

N<sub>2</sub> Nitrogen (Diatomic)

NH<sub>3</sub> Ammonia

NO<sub>x</sub> Nitrogen Oxides O<sub>2</sub> Oxygen (Diatomic)

S Sulfur

SO<sub>2</sub> Sulfur Dioxide

#### EXECUTIVE SUMMARY

The purpose of the Guideline Manual is to provide recommendations for the application of combined gas reburning-sorbent injection (GR-SI) technologies to pre-NSPS boilers. The manual includes design recommendations, performance predictions, economic projections and comparisons with competing technologies. The report also includes an assessment of boiler impacts. Two full-scale demonstrations of gas reburning-sorbent injection form the basis of the Guideline Manual. A GR-SI demonstration was completed on Illinois Power Company's Hennepin Unit #1. It is a 71 MW<sub>e</sub> (net) tangentially fired utility boiler and the primary fuel is an Illinois Bituminous coal containing approximately 3 wt% sulfur. A second GR-SI demonstration was completed at City Water, Light and Power's Lakeside Station Unit #7, located in Springfield, Illinois. Unit #7 is a 33 MW<sub>e</sub> cyclone-fired unit that burns medium to high sulfur Illinois Bituminous coal.

Gas Reburning (GR) involves reducing the levels of coal and combustion air in the burner area and injecting natural gas above the burners followed by the injection of overfire air (OFA) above the reburning zone. This three-zone process creates a reducing area in the boiler furnace within which NO<sub>x</sub> created in the primary zone is reduced to elemental nitrogen and other less harmful nitrogen species. Each zone has a unique stoichiometric ratio (ratio of air to that theoretically required for complete combustion) as determined by the flow of coal, burner air, natural gas, and OFA.

Sorbent Injection (SI) technology controls  $SO_2$  emissions through injection of a calcium-based sorbent such as hydrated lime  $[Ca(OH_2)]$  into the boiler furnace where it reacts with gaseous  $SO_2$  to form solid calcium sulfate. This compound is then removed from the flue gas in the baghouse or electrostatic precipitator.

GR and SI are applied simultaneously to achieve combined NO<sub>x</sub> and SO<sub>2</sub> control. Although significantly reducing the NO<sub>x</sub> emissions, GR also achieves an incremental

reduction in SO<sub>2</sub> emissions, since natural gas contains no sulfur. This complements the SO<sub>2</sub> reduction of the SI process and reduces the amount of sorbent otherwise required.

# **Process Design**

The process design for application of GR-SI technology is developed according to a generalized methodology involving the application of various experimental and analytical tools. Functional design requirements are based on characteristics of the subject boiler, the GR-SI process requirements, and the system performance goals. considerations that are essential to the design and practical application of the reburning system are: the reburn zone stoichiometric ratio, the temperature (or location) at which the natural gas is injected, the OFA injection location, and any impacts on boiler thermal performance. Rapid and complete mixing of the injected reactants with the local furnace gases is critical to the successful application of the GR-SI process. Detailed information concerning the flow field of the subject boiler can be accomplished by isothermal flow modeling. Flow visualization is accomplished using smoke and neutrally buoyant bubble injection. Velocity measurements are made within the model using hot wire anemometer and Kurtz probe instrumentation. The hot wire anemometer is used with observations of the yarn tufts to produce velocity and mass distribution profiles at various measurement planes in the model. Dispersion measurements are made to determine the degree of mixing at locations downstream of a proposed natural gas, OFA, or SI configuration.

# **Engineering Design**

Installation of a GR-SI system involves retrofit of the equipment onto an existing boiler. Due consideration must be given to design of the following items and areas:

- Supply of natural gas, pressure, piping size requirements
- Mass flow rate requirement for flue gas recirculation (if needed)

- Injector configuration vs. boiler structural constraints
- Cooling medium for injectors
- Sorbent storage and transport system vs. material selection
- Boiler tubewall penetrations
- Existing regenerative air heater design
- Capacity of sootblowing system
- Equipment footprint
- Electrical power distribution
- Plant and instrument air
- Existing controls system
- Existing particulate control system
- Existing ash handling and disposal system

# **System Operation**

Control and monitoring of the GR-SI system was accomplished through a Westinghouse Electric Process Control WDPF system. The system consists of a variable mix of functional units (drops) communicating freely and rapidly via the WDPF Data Highway. The WDPF sends and receives signals from various components in the GR-SI system, in addition to interfacing with other microprocessors.

The GR system is composed of three integrated systems: (1) natural gas injection, (2) FGR, and (3) OFA injection. The natural gas flow rate is controlled to the desired value for optimum NO<sub>x</sub> destruction. The FGR flow is controlled to a value to give the natural gas momentum for optimum distribution in the furnace. The OFA is controlled to a value to complete combustion of all unburned fuel leaving the reburning zone. The three integrated systems were interlocked, operated and monitored by the WDPF control system. The SI system variables were modulated by the WDPF control system to obtain a target sulfur

emission reduction while maintaining maximum sorbent utilization. The GR-SI control system worked well as designed.

# **Technology Performance**

The GR-SI process involves the injection of reburning fuel (in this case natural gas) into the furnace above the coal burners for reduction of nitrogen oxides ( $NO_x$ ) and injection of dry sorbent into the upper furnace for capture of sulfur dioxide ( $SO_2$ ). During field evaluations, a wide range of boiler operating conditions and GR–SI parameters were tested to determine the optimum conditions for  $NO_x$  and  $SO_2$  emission reductions. The project goals of 60 and 50% reductions in  $NO_x$  and  $SO_2$ , respectively, were achieved for both demonstrations.

One of the GR-SI demonstrations was completed on Illinois Power Company's Hennepin Unit #1. The pre-GR-SI baseline  $NO_x$  was 0.75 lb  $NO_x$  /106Btu (323 mg/MJ) at full load. GR parametric tests showed  $NO_x$  emissions ranging from a low of 0.19 lb/106Btu (81 mg/MJ) to a high of 0.36 lb/106Btu (157 mg/MJ) over the operating limits of gas heat inputs ranging from 10 to 22%, primary zone air:fuel stoichiometric ratios ranging from 1.06 to 1.15, and reburn zone stoichiometric ratios ranging from 0.88 to 1.01. The GR system was designed to maximize  $NO_x$  emission reductions for an 18-20% reburn gas heat input. The goal was to achieve a minimum of 60%  $NO_x$  reduction and long term testing at full load yielded an average  $NO_x$  reduction of 67%.

Baseline  $SO_2$  emissions averaged 5.24 lb/10<sup>6</sup>Btu (2,250 mg/MJ) and sorbent  $SO_2$  capture ranged from a low of 25.3% to a high of 60.6%. GR testing only showed the expected  $SO_2$  reductions, with 8 and 12% reburn gas heat input compared to baseline, 8 to 13%  $SO_2$  reductions occurred. The GR-SI test, showed an  $SO_2$  reduction of 52.6%, when operating with 18% reburn gas heat input and a Ca/S of 1.57.

Fly ash samples were collected to evaluate carbon loss. At a primary zone air:fuel stoichiometric ratio (SR) above 1.06, carbon in fly ash varied from 2.2 to 5.2% compared to the baseline value of 2.6%. The unit could be run at a primary zone SR of 1.07 and above and maintain the low carbon-in-ash seen for baseline operation.

A second GR-SI demonstration was completed at City Water, Light and Power's Lakeside Station Unit #7. The baseline  $NO_x$  for Lakeside Unit #7 was 0.97 lb/10<sup>6</sup>Btu. Testing showed that operation at a primary SR of 1.08 and reburning zone SR of 0.83 resulted in the highest  $NO_x$  reduction at full load (67%), while operation at a primary zone SR of 1.15 and reburning zone SR of 0.90 achieved the target  $NO_x$  reduction of 60%.

Furnace SI was designed to achieve  $SO_2$  reductions in the 25 to 50% range. When combined with GR, higher  $SO_2$  reductions occur due to coal replacement with natural gas. Reductions in  $SO_2$  were calculated from the 5.9 lb/10 $^6$ Btu (2540 mg/MJ) baseline. The  $SO_2$  emission reductions under SI-only operation, for the range of load 19 to 34 MW $_e$  were 25% at a Ca/S of 1.1 and 42% at a Ca/S of 2.1. Under GR-SI operation with gas heat inputs of 22 to 25%,  $SO_2$  reductions were 51% at a Ca/S of 1.1 and 61% at a Ca/S of 2.1. In the majority of GR-SI tests, with 22 to 25% gas heat input, the goal of achieving a 50%  $SO_2$  reduction was achieved. The maximum  $SO_2$  reduction measured under GR-SI was 68% at a Ca/S of 2.09 and gas heat input of 23%.

# **Boiler Impacts**

In steam generating units, the heat released from combustion of fuels must be absorbed by heat exchangers with high efficiency. GR operation can affect the thermal performance of the unit in two ways. First, GR affects the furnace heat release profile and second, GR operation changes local stoichiometric ratios and particulate loading resulting in minor changes in lower and upper furnace deposition patterns. The demonstrations showed that the overall impact of GR operation on the heat absorption profile was very minor. There

was a reduction in thermal efficiency of approximately 1% due to the increased H/C ratio of the natural gas compared to coal which resulted in more moisture heat loss to the atmosphere. During SI operation, there were reductions in convective heat absorption due to sorbent fouling, especially in the generating bank. Near continual use of sootblowers were required to remove deposits and maintain steam temperatures. The increased particulate loading also caused a shift in heat absorption from the furnace and upper convective pass to the convective backpass. This shift had the following impacts:

- Reduction in thermal efficiency (approximately 1.2%)
- Increase in temperature and velocity of the flue gas
- Higher fly ash resistivity that reduced the collection efficiency of the ESP

These demonstrations, however, showed that GR-SI operation does not exacerbate slagging in the furnace. Some buildup of slag in the lower furnace, from the top burner elevation to the OFA ports did occur, but not the excessive levels seen with baseline operation. Little change in upper furnace slagging was observed. Injection of natural gas and FGR did promote slag accumulations around the nozzles forming "eyebrows" and on the waterwall areas above the natural gas/FGR injectors. Periodic manual cleaning (nominally weekly) was sufficient to maintain normal GR operation.

During GR, approximately 10-20% of the coal fuel is replaced by natural gas. The use of gas allows for a corresponding reduction in coal usage, reducing mechanical wear on the coal processing equipment. Normal rates of tube wastage in coal-fired boilers are in the range of 0.001" to 0.003" per year. During the two field demonstrations one-year tests were conducted to evaluate tube wastage. Following extensive mechanical testing and statistical evaluation, it was concluded that there was no significant tube wastage in the boilers nor was any metallurgical degradation was found.

Further, no adverse conditions in the air heater were noted as a direct result of GR-SI operation. However, SI required an increase in sootblowing which could have a degrading effect on the air heater.

Particulate decreases proportionally to the amount of natural gas used with GR. However, a marked increase occurs with SI. For this reason it was necessary to perform an evaluation of the ESP, prior to installation of SI, to verify that it had sufficient capacity to handle the increased particulate loading. Due to ESP limitations, humidification was required for Hennepin Unit #1 and kept particulate emissions to pre-GR-SI emission levels.

#### **Economics**

Capital and operating costs for the GR system for NO<sub>x</sub> emissions reduction and the SI system for SO<sub>2</sub> emissions reduction were completed for the retrofit of a 300 MWe coal cyclone-fired power plant. The reason for selecting a cyclone-fired unit is that there are not many low cost NOx control options for these type of units. The degree of complexity regarding retrofit costs were factored based on the retrofit costs for the GR-SI demonstrations completed under this DOE contract. These two systems were treated as separate technologies; the only major synergistic effect of the GR system on the SI system is the reduction of SO<sub>2</sub> based on the replacement of sulfur-containing coal with natural gas devoid of sulfur. Economic projections were made using current dollars which include an inflation rate of 4.0%, and constant dollars which ignore inflation.

# Gas Reburning

Table 1 shows the performance and cost for a 300 MWe GR System that is retro-fitted to a cyclone-fired boiler. NOx reduction costs are based on a 65% capacity factor with 20.1% of the heat input supplied by natural gas at a gas to coal price differential of \$1.00/million Btu. The incremental increase in the levelized cost of power, including capital charges

# TABLE 1. 300 MWe GR RETROFIT PERFORMANCE AND COST

# **Summary of Data**

#### **Power Plant Attributes**

	Units	Value
Plant capacity, net	MWe	300
Power produced, net	10 <sup>9</sup> kWhr/yr	1.71
Capacity factor	%	65
Plant life	yr	15
Coal feed	tons/yr	683,280
Sulfur in Coal	wt%	3.0
Emis	sions Control Data	
	Units	NOx
Removal efficiency	%	67.0
Emissions standard	lb/10 <sup>6</sup> Btu	0.86
Emissions without controls	lb/10 <sup>6</sup> Btu	1.30

#### **Levelized Cost of Power**

Emissions with controls

Amount reduced

lb/10<sup>6</sup> Btu

tons/yr

0.43

7,439

	Current Dollars		Constant Dollars	
	Factor	Mills/kWhr	Factor	Mills/kWhr
Capital Charge	0.160	0.47	0.124	0.37
Fixed O&M	1.314	0.22	1.000	0.17
Variable Operating Cost	1.314	2.99	1.000	2.28
Total Cost		3.69		2.81
SO <sub>2</sub> Credits	1.314	(0.58)	1.000	(0.44)
Total Cost w/SO <sub>2</sub> Credits		3.11		2.37

#### **Levelized Cost - NOx Removal Basis**

	\$/ton			\$/ton
	Factor	removed	Factor	removed
Capital Charge	0.160	109	0.124	84
Fixed O&M	1.314	51	1.000	39
Variable Operating Cost	1.314	687	1.000	523
Total Cost		847		646
SO <sub>2</sub> Credits	1.314	(133)	1.000	(101)
Total Cost w/SO 2 Credits		713		544

Basis: 67% NOx reduction assumed based on larger unit with longer Reburn zone residence time than CWLP 33 MWe cyclone unit (58% NOx reduction w/20% gas heat input as Reburn fuel).

is estimated at 2.81 mills/kWhr in constant dollars and 3.69 mills/kWhr in current dollars. If an  $SO_2$  credit is applied based on a sulfur reduction when firing natural gas, the net incremental increase in the levelized cost of power is estimated at 2.37 mills/kWhr in constant dollars and 3.11 mills/kWhr in current dollars. The levelized cost of  $NO_x$  removal is estimated at \$646/ton and \$847/ton for current and constant dollar projections, respectively. If an  $SO_2$  credit is applied based on fuel sulfur reduction, the net levelized cost of  $NO_x$  removal is estimated at \$544/ton and \$713/ton for current and constant dollar projections, respectively.

An independent study completed for the U.S. EPA, "Investigation of Performance and Cost of  $NO_x$  Controls as Applied to Group 2 Boilers", compared the costs of competing  $NO_x$  control technologies. The costs for various  $NO_x$  reduction systems applied to cyclone-fired units were developed as part of this study. The cost of Gas Reburning, Coal Reburning, Selective Non-Catalytic Reduction (SNCR) and Selective Catalytic Reduction (SCR), based on \$/kWe and \$/ton of  $NO_x$  removed are shown for 400 MWe cyclone-fired units. As shown in Table 2, the cyclone-fired boiler  $NO_x$  control technologies show a cost per ton of  $NO_x$  removed that ranges from approximately \$490 to \$690.

TABLE 2. 400 MWe CYCLONE-FIRED NO, CONTROL COMPARISON

Technology	NOx Reduced %	Capital Cost \$/kWe	NO <sub>x</sub> Removed <sup>4</sup> \$/ton
Gas Reburning <sup>1</sup>	60	15.2	590
Coal Reburning	50	52.7	490
SNCR <sup>2</sup>	35	7.3	690
SCR <sup>3</sup>	50	40.0	575

<sup>(1)</sup> Natural Gas @ \$2.47/106 Btu and Coal @ \$1.47/106 Btu

<sup>(2) 50%</sup> Urea solution @ \$0.50/gal

<sup>(3)</sup> Anhydrous Ammonia @ \$162/ton & SCR catalyst replacement (3 yr life) @ \$350/ft<sup>3</sup>

Based on this comparison Coal-Reburning is the least-expensive and SNCR the most expensive with GR and SCR being near equal when the price differential between natural gas and the primary coal is \$1.00 /10<sup>6</sup> Btu. The NO<sub>x</sub> reduction for SCR assumed for this study was low, only 50%. However, SCR systems have achieved 85%+ reductions with increased catalyst volume.

# Sorbent Injection

The SI system was designed to store, meter, and convey micronized hydrated lime (sorbent) to the injection nozzles in the upper furnace of the 300 MWe cyclone-fired Unit. The SI system is comprised of the following major components: sorbent storage silo, weigh hopper, rotary valve feeder, screw pump, air transport blower, conveying line, sorbent splitter, SI air fan, and furnace injection nozzles.

Economic projections were made using current dollars, which include an inflation rate of 4.0%, and constant dollars which ignore inflation. Table 3 shows the performance and cost for an SI System that is retro-fitted to a 300 MWe cyclone-fired boiler. The incremental increase in the levelized cost of power is estimated at 5.32 mills/kWhr in constant dollars and 6.98 mills/kWhr in current dollars. The levelized cost of SO<sub>2</sub> removal is estimated at \$490/ton and \$643/ton for constant and current dollar projections, respectively.

The primary methods of coal-fired utilities used in complying with the 1990 Clean Air Act Amendments (CAAA) include switching from high sulfur Eastern coal to low sulfur Western coal, purchase of SO<sub>2</sub> allowance credits and stack gas scrubbing. With relatively lower capital costs and higher operating costs as compared to the wet scrubbing systems, SI might be suited for application to aging plants with low capacity factors or relatively short operating lives. To reach compliance when using SI, utilities may consider allowance purchasing, or scrubbing on newer units in their system. The SO<sub>2</sub> removal cost of SI is compared to wet scrubbers, coal switching and SO<sub>2</sub> allowance credits in Table 4.

# TABLE 1. 300 MWe\_SI RETROFIT PERFORMANCE AND COST

# **Summary of Data**

#### **Power Plant Attributes**

	Units	Value
Plant capacity, net	MWe	300
Power produced, net	10 <sup>9</sup> kVVhr/yr	1.71
Capacity factor	%	65
Plant life	yr	15
Coal feed	tons/yr	683,280
Sulfur in coal	wt%	3.0

# **Emissions Control Data**

	Units	SO <sub>2</sub>
Removal efficiency	%	45
Emissions standard	lb/10 <sup>6</sup> Btu	1.20
Emissions without controls	lb/10 <sup>6</sup> Btu	4.80
Emissions with SI control	lb/10 <sup>6</sup> Btu	2.64
Amount reduced	tons/yr	18,654

#### Levelized Cost of Power

LCTCIILCG GOOL OI I C	1101		
Current Dollars		<b>Constant Dollars</b>	
Factor	Mills/kWhr	Factor	Mills/kWhr
0.160	0.36	0.124	0.28
1.314	0.19	1.000	0.14
1.314	6.44	1.000	4.90
	6.98		5.32
	Curren Factor 0.160 1.314	Factor         Mills/kWhr           0.160         0.36           1.314         0.19           1.314         6.44	Current Dollars         Consta           Factor         Mills/kWhr         Factor           0.160         0.36         0.124           1.314         0.19         1.000           1.314         6.44         1.000

# Levelized Cost - SO<sub>2</sub> Removal Basis

	\$/ton			\$/ton	
	Factor	removed	Factor	removed	
Capital Charge	0.160	33	0.124	26	
Fixed O&M	1.314	20	1.000	15	
Variable Operating Cost	1.314	589	1.000	448	
Total Cost		643		490	

Basis: 300 MWe unit size and 65% onstream capacity factor

26% calcium utilization @ Ca/S ratio of 1.75 and \$83/ton price for 95% Ca(OH)2 hydrated lime

TABLE 4. SI COMPARISON WITH OTHER SO<sub>2</sub> COMPLIANCE OPTIONS

Technology	SO₂ Removed %	SO₂ Removed \$/ton
SI	30-50	490 <sup>1</sup>
SO <sub>2</sub> Scrubbers	90-95	322 <sup>2</sup>
Switch to Low Sulfur Coal	65-85	113³
SO <sub>2</sub> Allowance Credits	0-100	110⁴

- (1) Nominal cost for 300 MWe unit based on 26% calcium utilization and hydrated lime at \$83/ton
- (2) Power Generation publication, 4/4/97, based on average cost for 80 scrubbers (1996 survey)
- (3) Power Generation publication, 4/4/97, average of 112 units that switched to low sulfur Western coal
- (4) Power Generation publication, 4/4/97, average selling price in 1996

With current economics, the SI technology will probably not be cost competitive with other available utility options for base and intermediate load unit applications unless the SI calcium utilization can be increased and the cost of the lime based reagent decreased.

# 1.0 OVERVIEW

# 1.1 Purpose of the Report

The purpose of the Guideline Manual is to provide recommendations for the application of combined gas reburning-sorbent injection (GR-SI) technologies to pre-NSPS boilers. The manual includes design recommendations, performance predictions, economic projections and comparisons with competing technologies. The report also includes an assessment of boiler impacts.

# 1.2 Basis of the Report

Two full-scale demonstrations conducted as a part of the U.S. Department of Energy's Clean Coal Technology Program (Round 1) form the basis of the Guideline Manual. The purpose of the demonstrations was to assess the air emissions reduction potential of GR-SI on coal-fired utility boilers. The targeted emissions are those that comprise acid rain precursors, specifically oxides of nitrogen (NO<sub>x</sub>) and sulfur dioxide (SO ½). The specific performance goals were to demonstrate NO<sub>x</sub> and SO<sub>2</sub> emission reductions of 60% and 50%, respectively with acceptable unit operability and minimal operating cost. The emission reduction goals for both demonstrations were achieved, showing that the design methodology developed and applied by EER was valid.

The first demonstration was performed at Illinois Power's Hennepin Unit #1, located in Hennepin, Illinois. This unit is a 71 MWe (net) tangentially-fired boiler that fires high-sulfur Illinois coal. The second demonstration was performed at City Water Light & Power's Lakeside Unit #7, located in Springfield, Illinois. This unit is a 33 MWe (gross) cyclone-fired boiler that also fires high-sulfur Illinois coal.

Following design and installation of the equipment, the optimum operational setpoints were established through a series of pre-planned parametric tests. Optimum conditions are defined as those providing the maximum benefit (reductions of NO<sub>x</sub> and SO<sub>2</sub> emissions) for the minimum cost (natural gas usage) while operating within established boiler constraints. The parametric tests were followed by normal operation for approximately one year. Both prior to and following the long-term tests, a series of inspections were performed to assess the durability of the boiler tubes and other boiler equipment.

# 1.3 Reference Material

For a detailed technical description of the two GR-SI projects described above, please refer to the following reports:

Enhancing the Use of Coals by Gas Reburning-Sorbent Injection

Volume 1 - Program Overview

Volume 4 - Gas Reburning-Sorbent Injection at Lakeside Unit 7, City Water Light and Power, Springfield, Illinois

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Illinois Department of Commerce and Community Affairs

# Prepared by:

Energy and Environmental Research Corporation

For additional information, please refer to the technical papers listed in the references and the technical reports listed in the bibliography of this report.

#### 2.0 PROCESS DESIGN

# 2.1 Process Description

# 2.1.1 Gas Reburning

Gas Reburning (GR) is a very flexible NOx reduction technology that can be run in several ways to provide varying degrees of NOx reduction. The GR system can be operated under the three following conditions:

# 2.1.1.1 Baseline mode with no reburn fuel

Under this condition, although no reburn fuel is being added, there are low rates of cooling air flowing around the gas injectors and through the overfire ports. Based on maintaining the same oxygen level in the flue gas exiting the furnace as for the pre-GR application, a slight air staging occurs that reduces NOx emissions slightly (~10% reduction) compared to pre-GR retro-fit emissions. Carbon burnout under this mode of operation will be very similar to the pre-GR retro-fit.

# 2.1.1.2 Overfire air (OFA) only

By adding overfire air without the use of reburn fuel, staged combustion can be put into place to reduce NOx emissions. In this mode of operation, as the overfire air rate is increased, the air rate to the primary burners automatically decreases to maintain the O<sub>2</sub> set point at the exit of the boiler economizer.

With a reduced air rate to the burners, the localized burner zone becomes hotter which has the tendency to increase NOx production under oxidizing conditions, but since there is less fuel being fired through the burners a greater percentage of heat is absorbed in the furnace walls that would cool the burner zone. Even if the localized temperatures increase, the temperature mechanism for increasing NOx emissions is more than offset by the reduced partial pressure of the oxygen in the burner zone. The lower the partial pressure of oxygen, the lower the NOx production, and in the burner zone the oxygen concentration is controlling.

With this type of staged combustion approach, overfire air is added at a point downstream of the burners where the flue gas is cool enough to minimize the production of thermal NOx. With deeper staging (lowering of excess air levels in the primary burner zone) NOx emissions will reduce. The degree of staging is partially limited by the potential for higher corrosion in the hot burner zone due to higher CO concentrations, the deeper the staging the greater the potential for corrosion.

The other limiting factor is the carbon in the fly ash which increases with deeper staging. High carbon in ash could affect the ability of the utility to sell its fly ash to the cement industry. Overall NOx reduction using a near optimum overfire air addition rate, taking into consideration the concerns delineated above, will yield approximately a 35% reduction compared to pre-GR retrofit operation.

#### 2.1.1.3 Reburn mode

Under full GR implementation, the combustion process is divided into three zones as illustrated in Figure 2-1. In the primary zone, the main fuel is fired through conventional burners but at a reduced rate to compensate for the reburning fuel which is injected downstream.

In the reburning zone, injection of the reburning fuel consumes the excess air (oxygen) from the primary zone, producing a slightly fuel rich region where NOx is reduced by reactions with hydrocarbon radicals, carbon monoxide and hydrogen. Flue gas

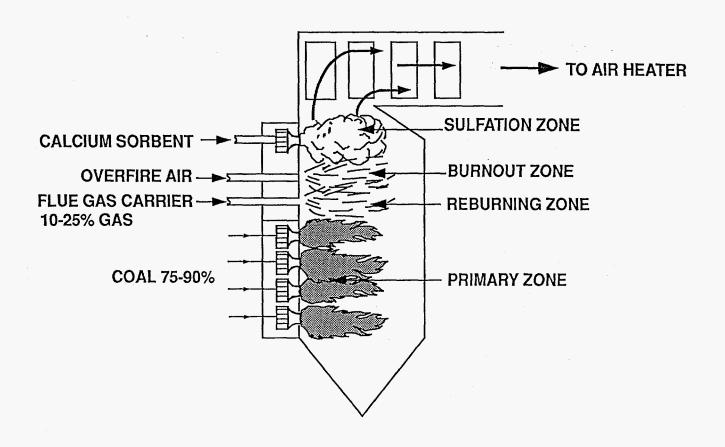


Figure 2-1. Schematic of GR-SI System

recirculation (FGR) may be used to provide momentum to the natural gas injection. FGR has a low O<sub>2</sub> content and therefore has a minor impact on reburning and burnout zone stoichiometric ratios. OFA is added in the burnout zone to complete the combustion of the fuel gases produced in the reburning zone and to adjust the overall excess air to yield good carbon burnout. Thus, except for relatively minor changes in boiler efficiency, the total heat input to the furnace is the same as baseline operation, but is divided into two fuel streams. Similarly, the total air supplied to the furnace remains essentially unchanged but is divided into two streams, supplying air to the conventional burners and also to the OFA ports.

The three zones are described in more detail as follows:

- Primary (burner) Zone: Coal is fired at a rate corresponding to 75 to 90 percent of the total heat input, under low excess air (SR = 1.05 to 1.15).  $NO_x$  emissions in this zone are reduced by the lower heat release and the reduced oxygen concentrations.
- Reburn Zone: Reburn fuel (natural gas in this case) injection creates a fuel rich region wherein hydrocarbon fragments (CH, CH<sub>2</sub>, etc.) and carbon monoxide and hydrogen are formed which react with NO<sub>x</sub>, reducing it to atmospheric nitrogen. The optimum reburning zone stoichiometric ratio is approximately 0.90, achieved by injecting natural gas at a rate corresponding to about 15 to 20 percent of the total heat input. FGR may be injected with the natural gas to provide for better penetration and mixing with the furnace flue gas.
- Burnout (exit) Zone: OFA is injected higher up in the furnace to complete the combustion. OFA is typically 20 percent of the total air flow; a minimum excess air of 15 percent in maintained. OFA injection is optimized to minimize CO emissions and unburned carbon-in-fly ash.

With the GR system, natural gas is routed to the reburning zone of the boiler and introduced into the boiler gas stream through a series of injection nozzles. The flow rate of gas to the reburn injectors is controlled automatically by the boiler operation control

system. FGR, if used, is extracted from the boiler backpass, enhanced by a booster fan and injected simultaneously with the natural gas. To complete the fuel combustion, air at 500 to 600°F is extracted from the secondary air duct or windbox and is injected into the boiler downstream of the reburning zone through a series of OFA injection nozzles (see Figure 2-2). With GR, depending on initial NOx concentrations and reburn zone residence time, NOx reductions of 60 to 75% may be achieved.

A minimal flow of hot secondary air is maintained though the OFA injection nozzles when the GR system is not in service to keep the OFA nozzles cool and ambient air is used to cool the gas injection nozzles.

#### 2.1.2 Sorbent Injection

Sorbent Injection (SI) technology controls  $SO_2$  emissions through injection of a calcium-based sorbent such as hydrated lime  $[Ca(OH_2)]$  into the boiler furnace under the proper temperature condition where it reacts with gaseous  $SO_2$  to form solid calcium sulfate (see Figure 2-2).

Due to the high furnace temperature the injected hydrated lime is quickly calcined into calcium oxide [ $Ca(OH)_2 + heat \rightarrow CaO + H_2O$ ]. A portion of the calcium oxide then reacts with sulfur dioxide and oxygen in the furnace flue gases [ $CaO + SO_2 + 1/2O_2 \rightarrow CaSO_4$ ] forming calcium sulfate (gypsum). The partially sulfated sorbent material then passes through the boiler convective pass with the flue gas to be eventually collected by the particulate control device. Calcium oxide reactivity and residence time under sulfation conditions are the two major parameters which control utilization of the calcium of the sorbent and thus the cost effectiveness of the process. Calcium oxide reactivity depends strongly on its porosity and particle size as well as its thermal history. At a point, reactivity tends to decrease with increases in sorbent particle temperature due to grain growth. The

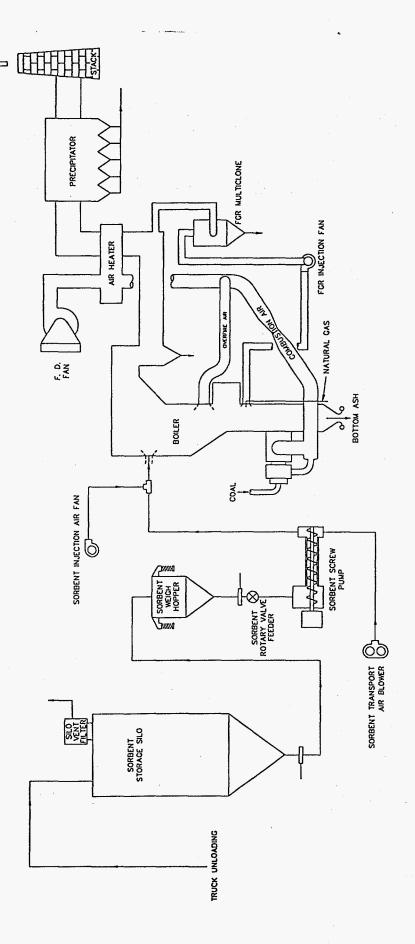


Figure 2-2. GR-SI Simplified Process Flow Diagram

residence time of the calcined sorbent particle in an optimum temperature window has a significant impact on sulfur (dioxide) capture potential.

Significant sulfation cannot occur above approximately 2250°F since the surface of the sorbent particle will be sintered (dead burned) to thwart the gas-solid reactions from occurring. Further, at these temperatures the calcium sulfate will start to decompose back to calcium oxide and sulfur dioxide. With a fluidized bed combustor the optimum temperature for lime based sorbent capture of sulfur dioxide is 1500 to 1600°F. However, with furnace injection sulfation becomes negligible below approximately 1600°F, due to the reaction rate requirements for the relatively short residence times in the furnace.

Sorbent is transported from a storage silo to the boiler and introduced into the flue gas through injection nozzles. A flow splitter in the sorbent line equally distributes the sorbent to the nozzles. To obtain the optimum sorbent mass flow and nozzle velocities required for adequate boiler dispersion, additional injection air is provided from a booster fan. Ambient air is used to cool the nozzles when the sorbent system is not in operation.

# 2.1.3 GR-SI Integration

GR and SI are applied simultaneously to achieve combined  $NO_x$  and  $SO_2$  control. Although significantly reducing the  $NO_x$  emissions, GR also achieves an incremental reduction in  $SO_2$  emissions, since natural gas contains no sulfur. This complements the  $SO_2$  reduction of the SI process and reduces the amount of sorbent otherwise required. The GR-SI system control may be integrated with any utility boiler control system. All start/modulation/stop operations may be performed in the boiler control room.

# 2.2 <u>Process Design Guidelines</u>

# 2.2.1 Gas Reburning

Since reburning requires no physical changes to the main combustion system, it can be applied to furnaces with virtually any firing configuration and fuel. The reburning process can be applied to all types of firing equipment including cyclone, tangential, wall, and stoker coal fired boilers. In addition, reburning can be applied to furnaces fired with any fossil fuel (coal, oil, gas, etc...). Reburning can also be applied to municipal waste incinerators, industrial boilers, and a range of industrial process furnaces.

The variables to be considered for an effective retro-fit of a GR system to an existing utility boiler are many, see Table 2-1. First of all, baseline NOx must be determined and the desired level of NOx reduction must be set. An evaluation must be made concerning the boiler configuration as input into determining the available residence times for the Reburn and OFA zones. A detailed boiler inspection is required to determine any physical constraints imposed regarding the locations of the gas reburn piping, OFA ducting, and reburn and OFA injectors.

# 2.2.1.1 Primary Zone

The burners in the primary zone do not require any modifications. However, by replacing excess air burners with low  $NO_x$  burners,  $NO_x$  reduction will be enhanced. The burners should be operated in a balanced mode and with the lowest excess air commensurate with acceptable lower furnace performance considering flame stability, carbon in ash, flame impingement and waterwall corrosion. Typically the optimum air for burner operation with GR is a rate that provides for about a 10% excess air condition in the primary zone.

TABLE-2-1. GR DESIGN GUIDELINES

Parameter	Units	Value	Comments
Primary Stoichiometry	SR	~1.10	As low as possible commensurate with good lower furnace performance and good carbon burnout
Reburn Injector Vertical Location	NA	Maximum temperature zone above burners	Primary burner fuel combustion must be essentially complete
Reburn gas flow	% of total heat input	~18	Design for a maximum of 25% for flexibility
Reburn zone Stoichiometry	SR	~0.90	Varies with gas injection rate to control NO <sub>x</sub> and primary burner zone SR.
Reburn injector array	NA	Rapid and complete mixing across furnace cross section	Site specific design
Reburn gas carrier fluid	NA	Flue gas recirculation (FGR) is preferred	Carrier gas with zero oxygen is the best, FGR, low in O <sub>2</sub> is the most cost effective
Reburn zone residence time	Sec.	0.25 minimum 0.50 and up is best	Above 0.50 sec., FGR may not be required
Overfire air (OFA) vertical position	NA	Located as high in the furnace as possible with complete combustion prior to convective pass entry	Site specific design
Overfire air (OFA) injector array	NA	Rapid and complete mixing across the furnace cross section	Site specific design
Overfire air (OFA) zone	SR	1.15 to 1.20	Sufficient to achieve baseline flue gas O <sub>2</sub> . May be adjusted to affect carbon burnout

### 2.2.1.2 Reburning Zone

<u>Vertical location</u> The reburn fuel injectors should be located above the uppermost row of burners. Optimum performance is achieved by positioning the injectors at the highest possible temperature (which means a location closest to the burners) where the burner fuel combustion is essentially complete. This point can be established by field testing using infurnace measurements to establish O<sub>2</sub>, CO and carbon in ash augmented by visual flame inspection through available ports. Optionally, or in addition to this empirical approach, the burner flame zones can be analytically modeled. It may also be necessary to make some adjustments to the vertical location of the injectors to avoid buckstays, platforms or other interferences external to the boiler.

Rate of Reburn Fuel Injection It is assumed that the objective of each utility, based on economics will be to achieve the maximum possible NO<sub>x</sub> reduction with the least amount of gas reburn fuel. The optimum condition for achieving this typically occurs when the rate of reburn fuel is set to yield about a 0.90 stoichiometric air/fuel ratio in the reburn zone. Based on the 10% excess air example for the primary zone, a 90% theoretical air in the reburn zone will require a reburn fuel rate that provides about 18% of the total boiler fuel input. To provide a margin of comfort regarding the optimum rate, the system should be designed to handle somewhat more gas flow, say 25%. It should be recognized that the gas flow rate is a variable and will be adjusted during operation as NO<sub>x</sub> control needs vary, higher gas rates yield higher NOx reductions and vice versa.

Gas Injector Array Once the vertical position for the gas injectors has been established, the injector array can be designed. The injectors must be designed to achieve uniform and complete mixing of the reburn gas across the full boiler cross section. The rate of mixing should be accomplished in minimum time so as to the maximize reburn zone residence time. The variables to adjust to achieve this include the number and position of reburn injectors and injection design parameters (mass flow rate of gas and any carrier gas,

injection velocity and injection angle). A number of analytical and empirical techniques, see Section 2.3, are used to design the injector array.

<u>Carrier Gas</u> A carrier gas, such as FGR, in certain applications can help to maximize the NOx reductions of a GR system. The carrier gas is used to increase the penetration and rate of mixing of the natural gas throughout the reburn zone. A carrier gas could be required to provide adequate penetration in large furnace boxes or for applications where the reburn zone residence times are short (<0.50 sec.).

The carrier gas has the following impacts on reburning:

- Provides rapid and effective mixing, the momentum of the injected gas can be enhanced by injecting the gas along with a carrier medium.
- Oxygen in the carrier medium is deleterious to reburn performance. The reason is that optimum NO<sub>x</sub> reduction is achieved under fuel rich conditions. As oxygen is added to the reburn zone via the carrier gas, additional reburn gas must be injected to consume this oxygen. This can result in a significant increase in the amount of natural gas required to achieve a specific NO<sub>x</sub> emissions level. Since the natural gas cost is the most significant component of the operating cost, this has the potential to adversely affect economics. Three carrier mediums can be considered: air, steam and flue Air has 21% O<sub>2</sub> and therefore is a poor choice based on gas consumption. Similarly a reburn injector, configured as a burner with air injected along with the fuel, requires more gas. Steam doesn't introduce O<sub>2</sub>; however, it has to be produced which requires both energy and water treatment. Flue gas is typically the best carrier medium. It has low O<sub>2</sub> (typically 3%) and requires no energy to produce. It does require a dust collector, fan and duct work. In this CCT-1 project, EER used FGR as a carrier gas. In the CCT-3 project, EER developed a second generation approach whereby the carrier requirement was eliminated (see CCT-3 report).
- Provides the advantage of being able to control injection parameters independent of the natural gas flow. Typically, the FGR carrier flow rate significantly exceeds the gas flow rate. Therefore as the gas flow rate varies, the injection velocity and flow rate are nearly constant. This allows for good mixing of the reburn fuel with the furnace gases as the natural gas flow is

turned down. Alternately, by varying the carrier medium flow, mixing conditions can be adjusted independent of the gas injection rate. This provides operational flexibility.

Reburn Zone Residence Time This refers to the time of passage of combustion products flowing through the reburn zone from the point of gas injection to the point of overfire air injection. The  $NO_X$  reduction reactions occur rapidly. Once the gas has been mixed with the flue gas, most of the  $NO_X$  reduction occurs within 100 milliseconds. Longer residence times result in additional  $NO_X$  reduction. Allowing for mixing times, a residence time on the order of 0.25 seconds is adequate to achieve good performance. A residence time of 0.50 seconds and greater provides excellent performance. EER uses a  $NO_X$  model to calculate the  $NO_X$  reduction for a specific application. It is applied considering the finite mixing rates. It should be noted that in some boilers there is significant flow separation. For example in the cyclone unit tested in this program at CWLP, a large recirculation region was present in the upper furnace. The residence time of concern for reburning is the residence time passing through the non-separated region.

#### 2.2.1.3 Overfire Air

<u>Vertical Position</u> The vertical position of the overfire air ports is established by balancing the need to maximize reburn zone residence time (which suggests ports higher in the furnace) and the need to ensure complete combustion prior to the convective pass (which suggests ports lower in the furnace). An oxidation model is applied to evaluate the conditions necessary to essentially complete combustion prior to the convective pass.

Rate of Overfire Air The overfire air injection rate should be sufficient to raise the stoichiometry of the combustion products to an excess air condition typical of baseline operation (~3-4% excess O<sub>2</sub>). It should be noted that in a conventional single stage combustion system, the overall excess air is the same as the burner excess air. In such a system, the operating excess air is established by the operators considering its impact

on burner performance, ash deposition in the lower and upper furnace, steam temperature and carbon burnout.

In a reburning system, the burner performance is de-coupled from the overall excess air. This provides the boiler operators with enhanced flexibility to adjust overfire air. By designing an overfire air system for rapid and complete mixing, it may be possible to operate the unit at excess air levels lower than baseline while still achieving good carbon burnout.

Overfire Air Injector Array Once the vertical position has been established, the overfire air injector array can be designed. The overfire air injector performance impacts carbon burnout and more specifically the minimum excess O<sub>2</sub> necessary to achieve burnout. The overfire air injectors must be designed to achieve uniform and complete mixing of the overfire air across the full boiler cross section in minimum time. The variables to adjust to achieve this include the number and position of the overfire air injectors, and injection parameters (injection velocity and injection angle). A number of analytical and empirical techniques can be used to design the injector array as indicated in Section 2.3.

### 2.2.2 Sorbent Injection

Like GR for NOx control, SI for SO<sub>2</sub> control may also be applied to any type of coal-fired utility boiler; including cyclone, tangential, wall, and stoker fired units. However, certain variables must be considered when evaluating the retro-fit of an SI system to a specific utility boiler, see Table 2-2.

The SO<sub>2</sub> reduction potential of the SI technology is limited and best applies to those units that require modest levels of SO<sub>2</sub> reductions. The higher the SO<sub>2</sub> removal desired, the higher the Ca/S ratio must be to achieve such reductions and the lower the sorbent calcium utilization. Lower calcium utilization in combination with the need for added sootblowing

TABLE-2-2. SI DESIGN GUIDELINES

Parameter	Units	Value	Comments
Sorbent Type	NA	Calcium Hydroxide	Highest reactivity
Injection Temperature	°F	~2200	"Dead Burning" may occur at 2250°F and greater
Residence time in Temp. Window	Sec.	~0.5	Temperature window from 2200 to 1600°F
Sorbent injector array	NA	Rapid and complete mixing across furnace cross section	Site specific design
Carrier Fluid	NA	Air	Can be integrated with overfire air
Injection rate (Ca/S)	Molar Ratio	~2.0	Vary to control SO <sub>2</sub> as required.  Maximum dictated by fouling
Maximum SO <sub>2</sub> Control	%	40-80	Site specific

with high Ca/S ratios, have technical/economic effects which limit the practical maximum degree of SO<sub>2</sub> reduction (~50%) with the SI technology.

# 2.2.2.1 <u>Sorbent Type</u>

A number of calcium based sorbents can be used; including limestone, dolomite, calcium hydroxide. The sorbents may be produced with various processes to enhance reactivity. However, it is difficult to evaluate reactivity based on sorbent properties, and in particular the surface area of the sorbent and the particle size cannot be used. When the sorbent is injected into the furnace, its characteristics are changed by the high temperature environment. For limestone and dolomite, the CO<sub>2</sub> in the carbonate is driven off, leaving

the oxide CaO; this is termed calcining. For hydrated lime, the  $H_2O$  is driven off leaving CaO. The CaO reacts with  $SO_2$ . Therefore, it is the characteristics of the "activated" CaO particles that affect  $SO_2$  capture. The activated CaO has external surface and internal pore surface areas.

The external surface area is always available for reaction with SO<sub>2</sub>. However, to utilize the pore area, SO<sub>2</sub> must diffuse into the pores and sulfate deposits at the pore openings can restrict mass transfer to limit the usefulness of the internal surface area. Each sorbent material produces a different CaO reactivity (such reactivity translates to SO<sub>2</sub> penetration into the sorbent pores). Impurities or additives in the sorbent, preparation methods, and sorbent injection conditions (primarily temperature) can all affect the reactivity of the activated CaO. To diminish the effect of pore access on calcium utilization, everything else being equal, the smaller the sorbent particles, the higher the calcium utilization.

To evaluate sorbent reactivity, a test should be conducted using the specific sorbent. The test should duplicate the  $SO_2$  level in the furnace and the temperature history of the injected sorbent in the full scale application. The  $SO_2$  reduction should be measured well downstream of the key temperature window. The figure of merit for sorbents most commonly used is the  $SO_2$  reduction achieved at a Ca/S molar ratio of 2.0. A sorbent with good reactivity injected under optimum process conditions can typically achieve 50%  $SO_2$  reduction from medium to high sulfur coals at Ca/S=2.0.

# 2.2.2.2 <u>Sorbent Injection Temperature (vertical location)</u>

The empirical evaluation of sorbent reactivity discussed above can also provide information on the effect of sorbent temperature history on performance. The furnace flue gas quench rate is dictated by the design of the boiler which establishes the heat absorption profile. Changes in quench rate require altering the heat exchanger surface or load/excess air changes which have costs and/or performance impacts associated with them that normally

make quench rate change options-unacceptable. Therefore, the sorbent injection system must be optimized to match the given characteristics of the specific boiler application.

For typical convective pass quench rates, the optimum temperature for sorbent injection is about 2200°F. Injection at higher temperatures will melt the surface of the sorbent ("dead burning") making the particle pores impervious to SO<sub>2</sub> entry and effectively eliminating the internal surface reaction area. Injection at lower temperatures reduces the residence time available for SO<sub>2</sub> reactions in the temperature window. With high quench rates (low volumetric heat release units), the residence time in the window is decreased. For these applications the sorbent injection location should be moved upstream to increase the sorbent residence time in the window and vice versa for those units with low quench rates (high volumetric heat release units).

It is important to note that the temperature of the furnace gases varies in three dimensions and with boiler operating conditions; load, excess air, burner settings - including tilt with tangential units, operation of the gas reburning system, and amount and condition of ash deposits in the furnace and convective pass. For optimum SO<sub>2</sub> control, the sorbent injection must accommodate these spacial and temporal variations. Depending on the system, it may be necessary to use multiple injection elevations.

# 2.2.2.3 <u>Sorbent Injector Array</u>

Once the vertical sorbent injection location has been set, the sorbent injector array can be designed. As with the overfire air and reburn gas injector designs, the objective is to inject the sorbent into the furnace to mix it with the furnace gases rapidly and completely, thus maximizing the residence time in the temperature window.

The variables to adjust to change mixing condition include the number and position of the sorbent injectors, and injection parameters (injection velocity and injection angle). A

number of analytical and empirical techniques can be used to design the sorbent injector array as indicated in Section 2.3.

#### 2.2.2.4 Carrier Fluid

Any convenient carrier fluid can be used to inject the sorbent. The amount of air injected is typically a few percent of boiler flow. If the sorbent injection system is integrated with a gas reburning system, and the overfire air is injected at a temperature commensurate with good sorbent performance (~2200°F), then the sorbent injectors can be integrated with the overfire air ports. This provides some reduction in system complexity and cost.

### 2.2.2.5 Maximum SO<sub>2</sub> Control

The maximum  $SO_2$  control achievable by sorbent injection is site specific and is related to the impacts of the injected sorbent on ash deposition and dust collection. For sorbent injection alone, the total dust loading passing through the furnace is the sum of the coal ash and the spent sorbent. With gas reburning-sorbent injection, the coal flow rate is reduced to accommodate the gas with attendant reduction in the coal ash passing through the furnace. Thus, for small sorbent injection rates, the total dust loading with GR-SI could be less than baseline. However, as the sorbent injection flow rate is increased, the total dust loading increases and at a design point of Ca/S=2.0, the total dust loading is increased from baseline substantially. This can be easily calculated from the flow rates of coal ash and sorbent entering the furnace. Baseline data can be used to estimate the fraction of the coal ash which is carried over as flyash. To be conservative it should be assumed that all of the sorbent is blown out of the furnace with the flyash. The primary limiting factors for the SI technology are (1) ash deposition and (2) dust collection. They are discussed below.

#### 2.2.2.6 - Ash deposition

The sorbent is typically injected high in the furnace and therefore has little impact on ash deposition and heat transfer in the lower furnace. In the upper furnace and convective pass the sorbent adds to the flue gas dust loading. The sorbent deposits generally have lower radiant energy absorption (more reflective deposits) than coal ash deposits and this reduces radiation heat transfer. Also, the increased dust loading results in increased deposition rates, particularly in the high temperature sections of the convective pass. As the sorbent injection rate is ramped up from zero, initially these impacts are minor and can be accommodated by slight shifts in burner tilts (with tangential units) and attemperation flow rates and increased sootblowing frequency. However, as the sorbent injection rate is increased further, the fouling and heat transfer shifts become unacceptable and effectively limit the maximum sorbent injection rate.

#### 2.2.2.7 <u>Dust Collection</u>

The particulate exiting the boiler is captured in a downstream dust collector, typically an electrostatic precipitator (ESP) or a fabric filter. Both boilers retrofitted with the GR-SI systems in this project had cold-side electrostatic precipitators.

Sorbent injection has two adverse effects on dust collection: increased dust loading and increased dust resistivity. If the collection efficiency of the ESP remains constant, the dust emissions from the system will be proportional to the dust loading entering the ESP. Since any increase in dust emissions is usually unacceptable, the collection efficiency of the ESP must be enhanced. The collection efficiency of an ESP depends on a number of factors and analytical models are available for predicting performance. One important parameter is the resistivity of the flyash that establishes the voltage gradient through the ash deposits on the precipitator plates. This in turn controls the maximum current that can be applied without "spark-over". High resistivity flyash decreases the maximum current and thus

reduces dust collection efficiency. Calcium sorbents, because they react with the SO<sub>3</sub><sup>=</sup> (a gas that reduces the fly ash resistivity) exiting the furnace produces a high resistivity flyash which degrades precipitator collection efficiency, thus compounding the dust emission problem. One way to counter this effect is to cool the flue gas upstream of the ESP. In this project humidification (liquid water injection) was used effectively.

These combined effects typically limit the maximum sorbent injection rate for medium to high sulfur coals to the range of Ca/S=2.0. Thus, the maximum SO<sub>2</sub> control depends on sorbent reactivity which is established for the specific sorbent and application in bench scale combustion tests.

### 2.3 Process Design Tools

The application of the GR-SI technology is developed according to a design methodology that involves the application of various experimental and analytical tools. Bench and pilot scale studies, as well as full scale demonstrations, have established that the major process parameters controlling the effectiveness of the reburning process are the operating stoichiometric ratio (air/fuel), temperature, and residence time and distribution of the reactants in the reburning zone. Pilot-scale studies have also shown the importance of effective mixing in both the reburning and burn out zones. In utility boilers with relatively short upper furnace residence times, mixing rates of the reburning gas and OFA largely control the effectiveness of the process. The following tools are used in the design of the GR-SI system.

# 2.3.1 Isothermal Flow Modeling

Rapid and complete mixing of the injected reactants with the local furnace gases is of critical importance in the successful application of the GR-SI process. Detailed information concerning the flow field of the subject boiler can be accomplished by isothermal flow

modeling. EER has successfully-used  $^{1}I_{12}$  scale boiler models to evaluate the flow field using the experimental techniques of flow visualization, velocity measurements, and dispersion measurements. Models are scaled geometrically with convective pass sections designed to match full-scale pressure loss coefficients. The model burners are designed to match full-scale momentum flux ratios, and flow rates are adjusted to match the large-scale flow patterns. The model is calibrated against actual furnace gas directional velocity measurements.

#### 2.3.2 Flow Visualization

Using the \$^{1}\$\text{\$^{1}\$}\_{12}\$ scale model, visualization of flow fields within the boiler furnace is accomplished using smoke and neutrally buoyant bubble injection. The model is constructed of transparent plastic, allowing the visualization of the flow fields to be preserved by photographs of the smoke and bubble trajectories. Due to the high mixing rates and large flow volumes, smoke injection does not lend itself well to mapping the flow fields of individual burners. However, smoke injection is used to conveniently visualize the flow pattern and trajectory of a particular natural gas, OFA, or SI system. Neutrally buoyant bubble injection is useful in tracking the flow fields of individual burners or burner rows. The bubbles are generated by a machine that suspends helium filled soap bubbles in a flowing air stream. Bubbles are then directed through a particular burner via a tube inserted through the side of the model's burner windbox.

# 2.3.3 Velocity Measurements

Velocity measurements are made within the model using hot wire anemometer and Kurtz probe instrumentation. The Kurtz probe is an air velocity meter providing real time analog readings of air velocities and is used for rough measurements of burner velocities and checking uniformity of velocities between injectors. Tufts of yarn are used within the model to provide relative directions of flow fields.

The hot wire anemometer is used with observations of the yarn tufts to produce velocity and mass distribution profiles at various measurement planes in the model. The velocity information provided by the hot wire anemometer is collected and processed by computer. The hot wire anemometry data and the tuft observations are used to calculate velocity components and mass fraction profiles for sample data points throughout the furnace (model). The velocity profiles are used for characterization of the model flow fields and comparison (calibration) against actual velocity measurements made on the full-scale boiler. The mass fraction profiles are used in conjunction with corresponding dispersion measurements (described below) to give sulfur capture predictions for SI configurations to be tested by the model.

### 2.3.4 <u>Dispersion Measurements</u>

Dispersion measurements are made to determine the degree of mixing at locations downstream of a proposed natural gas, OFA, or SI configuration. To perform the measurements, the air supplied to the proposed injection configuration is seeded with a gas (typically methane) whose concentration is readily measurable. Injection of the gas is located in a position well upstream of the injection system windbox so that complete mixing occurs prior to distribution to the various injection nozzles. A primary probe is inserted into the model at the sample point of interest, and a secondary or reference probe is inserted in the exhaust duct of the model where the gas is assumed to be well mixed with the air. Concentration profiles are then normalized for comparison of the penetration and mixing performance of the proposed injection system configurations.

#### 2.3.5 Overall NOx Reduction Model

The current approach used for the modeling of NO<sub>x</sub> formation is a compromise between semi-empirical and fully predictive models. In this approach, a boiler heat transfer model and a two-phase modular model are coupled for reburning NO<sub>x</sub> predictions. The heat

transfer model is used to calculate mean time-temperature profiles for the boiler, based on actual operating conditions. Information on flue gas temperature and residence time is then used in the modular model for reaction kinetics calculations. In setting up the modular model, distinctive zones in the boiler are identified and represented by combinations of Well Stirred Reactors (WSR) and Plug Flow Reactors (PFR). The PFR allows for the injection of side streams so that the reburn fuel and the OFA injection can be simulated. The modular model also incorporates a detailed submodel describing coal devolatilization, fuel nitrogen conversion, and char combustion.

For many situations, a detailed chemical kinetics model is not absolutely essential for boiler  $NO_x$  modeling. For GR, however, detailed chemistry models become essential for two main reasons: (1) NO reduction in the reburn zone is accomplished by fuel fragments such as CH; simpler models that do not consider such chemistry cannot be applied; and (2) a coal chemistry submodel is required to provide information on the composition of the species pool exiting the burner zone and to define initial conditions for the reburn zone  $NO_x$  calculations. Without this submodel, the reburning model becomes less effective because  $NO_x$  and other intermediate species generated in the burner zone remain unknown.

A two-phase modular model is set up around a specified number of WSR and PFR reactors connected in series. The PFR reactors allow for the injection of side streams with user specified stream composition, temperature, and injection mass flow rates. The model has a kinetics package that consists of elementary reaction mechanisms up to  $C_2$  chemistry. The package in its current form considers 60 species and 209 reactions. Certain of these species have been used specifically for describing the kinetics of coal devolatilization and combustion. In addition, gas phase species important for  $NO_x$  formation and destruction, such as  $NO_3$ , HCN,  $NO_x$ , and  $N_2O$  are all included.

The modular model also incorporates a detailed devolatilization submodel. Coal is assumed to devolatilize into light volatiles, volatile nitrogen, and tar according to first-order

rate equations. The remaining-solid phase material contains ash and char, which also contains residual fuel nitrogen (or char-nitrogen). The devolatilization model assumes, as suggested by experimental data, that volatile nitrogen is HCN, and the small amount of NH<sub>3</sub> is neglected. The aliphatic species contained in the light volatiles are made up of CH<sub>4</sub>, C<sub>2</sub>H<sub>2</sub>, C<sub>2</sub>H<sub>4</sub>, H<sub>2</sub>, and soot. Tar is assumed to have the same composition as the parent coal, based on the resemblance of IR spectra between the parent coal and the tar (Solomon and Colket, 1979). Devolatilization rate constants and partitioning of light volatiles, tar, and fuel nitrogen are based on experimental data (Kobayashi et. al., 1977; and Pohl and Sarofim, 1977; Suuberg, et. al., 1979).

The modular model includes reaction mechanisms for char oxidation and gasification. Char reacts with  $O_2$ ,  $CO_2$ , and  $H_2O$  to form  $CO_2$ , CO and  $H_2$ . Conversion of the charnitrogen into  $N_2$  by heterogeneous reactions on the char surface is included.

The model can be run for gas only or coal-gas cofiring. If a coal particle is present, the model assumes hydrodynamic equilibrium between the particle and the gas phase flow. Therefore, the particle follows the flow of the gas. The particle, however, can be at different temperatures than the gas phase; i.e., equilibrium is not reached in a thermodynamic sense. In this instance, a user-prescribed Nusselt number is provided for gas-particle convective heat transfer calculations. The model also uses a simple thin-flame radiative model for the calculation of heat exchange between gas-particle-furnace walls. Radiative species considered are CO, CO<sub>2</sub>, H<sub>2</sub>O, and solid particles. In the boiler environment, the thin flame radiation model is considered to be inaccurate, and therefore is not used in this analysis. Instead, the required temperature information for NO<sub>x</sub> modeling is taken from the results of the boiler heat transfer model.

Based on the tools developed, EER has a proven design methodology that permits an optimized reburning system to be designed, and which permits the prediction of site specific impacts of reburning on NOx emissions and boiler performance.

#### 3.0 ENGINEERING DESIGN

### 3.1 Gas Reburning System

### 3.1.1 Natural Gas System

For the two full scale electric utility reburning demonstrations, the reburn fuel used was natural gas. Approximately 15 to 25 percent of the total heat input to the furnace is supplied via natural gas for the reburning process. From this information, one can roughly size the volumetric rate (scfm) requirements for the natural gas to be supplied to the furnace. Standard piping design practices in conjunction with the rate requirements are used to size supply and distribution piping from existing headers within the facility and also, in some applications, pipelines off site.

Line pressures are designed to accommodate volumetric requirements while maintaining reasonable pipe sizes. Normal pressures are 100 psig in headers, 20 psig at the control valve trains, and 1-4 psig at the injection nozzles. On GR installations, with FGR as the inert media for assisting in the penetration and mixing of the natural gas with the furnace gases, the natural gas pressure is supplied to the injection nozzles is slightly above the pressure of the flue gas at the nozzle.

Control of natural gas flow into the furnace is critical not only for optimizing the GR process, but for maintaining boiler firing control and safety. GR itself is a chemical process that is different from combustion, but natural gas flow into the boiler is treated as another fuel input. Specific equipment and design recommendations with regard to gas firing are available in the National Fire Protection Association (NFPA) Standards 85B, "Prevention of Furnace Explosions in Natural Gas-Fired Multiple Burner Boiler-Furnaces", and 85C, "Prevention of Furnace Explosions/Implosions in Multiple Burner Boiler-Furnaces.

### 3.1.2 Flue Gas Recirculation System

The mass flow rate of natural gas injected into the reburning zone for the reburning process does not always possess sufficient momentum to penetrate into the furnace flue gases for adequate mixing. As such, an inert gas may be required to be mixed with the smaller amount of natural gas before injection into the furnace via several high velocity jets, the combination of the higher velocity with the higher mass flow providing the necessary momentum for good in-furnace mixing.

The logical source of the inert gas is the combustion flue gas at the boiler exit. At this location in the process, oxygen levels in the flue gas are at their lowest since air heater leakage downstream may significantly increases the oxygen concentration of the flue gas. The temperature of the flue gas extracted at this location eliminates the need for preheating the gas. Note that injection of low temperature gas streams into the furnace may quench the reburning process and contribute to ash and slag formations known as "eyebrows" at the openings.

Selection of the GR nozzle configuration (size, jet velocity, number, and location) is based on furnace gas flow modeling with consideration given to penetration into and mixing with the furnace combustion gases. The velocity head of the flue gas necessary to generate the required nozzle velocity is provided by a FGR fan. Due to the high velocity head requirements with relatively low flow, the FGR fan is a non-standard design having a large rotor diameter (high velocity head) and narrow rotor width (low flow). Although up to eighty percent of abrasive particulate matter is removed upstream of the fan by a multi-cyclone dust collector, the fan should be designed with wear liners to accommodate the fly ash that remains in the gas stream. As with any of the process streams, FGR flow must be metered to control the reburn process. A venturi is the preferred metering device since it can accommodate the ash loading and high temperatures (600°F). A clean air purge assembly is attached that prevents fly ash obstructions in the pressure sensing lines.

## 3.1.3 Natural Gas Injection

Natural gas is mixed with high velocity FGR at the injection nozzles. In order to survive the high temperatures of the furnace environment, both water-cooled metal and high temperature ceramic nozzle designs have been used. Nozzle provisions should also be made to resist erosion from fly ash. Cooling fans are required to provide cooling air to the injection nozzles during non-operation of the GR system and to provide seal air on positive pressure units.

## 3.2 Sorbent Injection System

#### 3.2.1 Sorbent Storage, Feed, Transport, and Distribution

Sorbent usage rates are calculated from the percentage of sulfur in the coal, the desired reduction in sulfur dioxide, the load (coal flow) anticipated, and the expected utilization of the sorbent (pulverized limestone or hydrated lime). Storage capacity for the sorbent silo is then a function of the desired product inventory, distance of the supplier from the facility, transportation methods and quantities (i.e., truck, rail, barge), and transportation costs.

Storage silo designs are either bolted or welded steel, or combinations of concrete and steel. Bolted steel designs offer the lowest capital cost and ease of construction, but have capacity limitations. The silos must be capable of receiving sorbent via pneumatic unloading equipment without excessive dust emissions. The most common delivery method, over the road bulk trucks, uses positive displacement blowers capable of delivering 600 acfm of air at 15 psig to unload sorbent into the storage silo. The receiving silo must be equipped with a fabric or pleated paper cartridge vent filter to prevent pressurization of the silo while retaining entrained dust. The silos should also be equipped with level detectors to prevent overfilling and combination pressure and vacuum relief valves to protect the silo structure as sorbent is delivered to or fed from the silo.

Sorbent is discharged from the silo by gravity through a central exit line on the conical bottom of the tank. Bridging and rat-holing at the cone outlet is prevented by fabric air slides using low pressure air to partially fluidize the sorbent and promote flow. Based on experience with limestone and hydrated lime sorbents, silo cone outlets and feed hoppers should have an included angle of approximately sixty degrees.

A wide variety of feed equipment is available to meter the sorbent flow rate from the silo into a pneumatic conveying line. Selection of feed equipment depends on the degree of process control required by the utility. In test or demonstration programs, weigh hoppers fitted with accurate rate-of-weight loss sensors have been used in conjunction with variable speed rotary valves to meter the sorbent flow rate. This configuration allows a feed rate (lbs/hr) display in the main control room. In applications where the utility is only interested in limiting SO<sub>2</sub> stack emissions by varying the sorbent flow delivered to the boiler, the weigh hopper and rate of feed indication may not be required. In this case, a variable speed feed device would be used to vary sorbent flow. Sorbent flow rates would then be calculated based on silo inventory.

Feed devices that are commercially available include rotary valves, screw, and belt feeders. Sorbents (especially hydrated lime) will sometimes flow like a liquid through feed devices that do not use positive feed control. Admission of air into the feed system via flow aids (i.e., air slides) may exacerbate this flow problem. Sorbents also have high flowabilty after being pneumatically loaded into the silo. Several hours of settling time may be required after delivery before the sorbent can be fed through the system without handling problems. Despite vendor claims, screw and weigh belt feeders do not offer positive feed control and system upsets are likely if this high flowability condition occurs. Only rotary valves have been successfully used to positively control sorbent feed.

Conveying of the sorbent from the silo feed equipment to the boiler is accomplished via a dilute phase transport system. Dilute phase transport systems operate with a solids

(sorbent)-to-air ratio of 5 to 1 or less (pound-to-pound basis). Dilute phase transport lends itself well to the distribution of the single sorbent stream into the several streams feeding the injection nozzles since the sorbent is evenly distributed throughout the transport piping. Dense phase or two phase flow systems offer smaller transport piping sizes, but require more complicated flow splitting techniques. Dense phase systems also require higher transport air pressures and therefore more expensive blowers or compressors with higher attendant power consumption.

Dilute phase conveying systems typically operate at pressures of 15 psig or less, allowing the use of positive displacement type (Roots) blowers for the conveying air supply. Line charging equipment is required to inject the sorbent from the feed and metering equipment into the conveying line. This equipment must deliver the sorbent to the conveying line across a pressure differential of up to 15 psig, without allowing air leakage back into the feed system which could lead to upsets and increased dust emissions. A solids screw Fuller-Kinyon pump developed and offered by Fuller Company is well suited for this application. The pump uses a screw with progressive flights to compact the sorbent and extrude it though a check valve into the conveying line.

Sorbent is delivered from the silo feed equipment into a single transport line. It is then distributed, or split, equally into several streams to feed the SI nozzles. Several splitter designs have been developed. EER developed its own flow-splitter. The splitter has a conical shape that transforms the cross section flow area at the inlet into an annular, or ring, shaped area at the outlet. The inlet diameter is sized to be identical to the attached sorbent transport hose, and the diameter of the annular outlet is sized just large enough to accommodate the number of outlet pipes required to supply the injectors. The design maintains adequate conveying velocities through the splitter so that saltation of the sorbent does not occur. The splitter relies on a uniform distribution of sorbent in the conveying airstream entering the splitter inlet. In horizontal runs, gravity segregates the sorbent to the bottom of the transport hose, so splitting with this design is done in a vertical

orientation. A good design rule of thumb is to provide at least twenty diameters of vertical, straight transport hose prior to the splitter inlet.

The sorbent storage silo should be located as close to the furnace as possible to minimize conveying distances. Aside from the rate of sorbent to be conveyed, the distance to be transported strongly determines the conveying air pressure requirements of the system. Conveying systems designed to operate at 12 psig or less can use less expensive positive displacement blower equipment as compared to sliding vane or multistage compressors capable of 15 psig or more.

### 3.2.2 Sorbent Injection

The configuration (number, size, location, and jet velocities) of the SI system is determined during process design. Air supplied by the sorbent conveying process proper is generally not sufficient to provide the required injection nozzle mass flow rates. As such, a booster fan may be required to provide additional air at sufficient velocity head for injection. The booster fan is similar in design to that of the FGR system, having a large diameter to develop the high velocity head, but with minimal width due to the low flow requirements. Since the fan will boost ambient air only, there are no requirements for rotor and housing wear liners.

SI nozzles are typically a coaxial jet design, with the inner pipe used for the sorbent and transport air mixture, and the outer pipe providing the balance of air required for injection, from the separate air booster fan. At the end of the injector nozzle, prior to furnace entry, the sorbent/transport air is mixed with the added injection air, providing the required velocity for sorbent injection into the furnace.

SI nozzles are of typically stainless steel construction to resist corrosion and the high temperature effects of the furnace. With the SI system in operation, the nozzles are cooled with the transport and injection air flows. When the SI system is not in service, cooling air flow must be established through the nozzles. Cooling air is supplied by relatively small sized centrifugal fans coupled to the injection air duct work. Since a cooling fan failure could create nozzle slagging of warping problems, redundant fans are normally required.

On positive pressure units, cooling air fans also supply seal air to the SI nozzles to prevent back flow of hot furnace gases into the SI air system. Also on positive fired units, a positive shut-off is provided in the sorbent transport system to prevent back flow of gases into the rubber hose conveying lines and sorbent feed system.

#### 3.3 Furnace / Boiler

## 3.3.1 Bent Tube Openings

Depending on results of the process design, application of the GR-SI technology may require as many as thirty tube wall penetrations to be made in the furnace water walls to accommodate the injection nozzles and the furnace gas temperature monitoring equipment. Each water wall opening may require from four to eight bent tubes to be installed, possibly affecting over one hundred of the water wall tube circuits. In considering the application of GR-SI, the impact of the bent tube openings on circulation and steam generation in the lower furnace water walls should be investigated.

#### 3.3.2 Tube Metals

SI results in very light colored to white deposits on the tube surfaces of the upper furnace and radiant tube banks of the superheater and reheater. The change in reflectivity of these surfaces can reduce the radiative heat transfer from the furnace gases to the tubes surfaces and shift (increase) the gas temperature profile through the unit. In practice, it has been found that nearly constant sootblowing is required to clean these light deposits

from the tube surfaces to maintain steam and gas side temperatures. Due to the delayed burn-out associated with the gas reburn process, gas temperatures may also be increased in the same areas. Based on predicted gas temperature increases, the temperature limits of metals used in the existing upper furnace water walls and radiant and convective tube banks should be investigated. If existing metals are shown to be marginal, the utility may opt for monitoring of tube temperatures via pad welded thermocouples during GR-SI operation rather than tube replacement which may not be warranted.

### 3.3.3 Air Heater Design

With the GR-SI operation, the downstream air heater will be affected by increased flue gas temperatures and particulate loading and fouling. Air heater designs on larger utility units may be either of the tubular or regenerative type design. When considering the application of GR-SI a review should be made of the sizing and condition of the existing air heater with the anticipated impacts of increased flue gas temperature and particulate fouling. Tubular air heaters, in poor condition, may require re-tubing to prevent tube failure and limit carry-over of the spent sorbent into the combustion air ductwork and control components.

Aging regenerative designs may be temperature limited due to the condition of air heater posts and diaphragms. Regenerative designs may also require replacement or adjustment of radial and circumferential seals to limit carry-over of spent sorbent into the combustion air system. Cleaning of air heater surfaces may be increased from once per shift to nearly continuous with SI. The condition of sootblowing equipment on either design should be evaluated in light of the increased service requirements. Tubular designs may require retrofit of circulating shot cleaning systems to limit fouling to acceptable levels since sootblower effectiveness may be limited on these units.

### 3.3.4 Sootblowing Equipment

Operation of the GR process results in slight changes to furnace water wall heat flux profiles, but experience has shown no detrimental impact on furnace wall slagging or fouling tendencies. SI however, results in a deposition of a thin and highly reflective (white) deposits on tube surfaces. This deposit layer affects radiation heat transfer in the upper furnace and in the radiant tube banks of the superheater and reheater (if equipped). This deposit layer is dry, not fused to the tube surface, and is easily removed by sootblowing. Experience has shown that nearly continuous operation of long retractable sootblowers is required in the radiant tube bank sections. Since SI takes place in the extreme upper portion of the furnace, the effect on water wall sootblowing requirements is negligible.

Sootblowing equipment should be reviewed with respect to coverage of the radiant sections of the furnace tube banks, age, and capacity of the air or steam supply in light of the expected continual operation. The existing complement of long retractable sootblowers in the radiant tube bank sections should be reviewed in terms of coverage and sootblowing pressure requirements. Additional sootblowers may be required in this area, with the possibility of additional water wall penetrations. Since nearly constant sootblowing is required, poor reliability of the sootblowing equipment may affect unit availability or require SI to be ceased, resulting in increased SO<sub>2</sub> emissions. The reliability assessment should include not only sootblowers but steam and/or air control valves, motor control centers, and system air compressors.

Sootblowing steam or air supply systems may be designed to service more than one unit, and may not have the capacity for normal sootblowing on the non-GR-SI unit(s) when providing the near constant sootblowing demand of the GR-SI equipped unit. Existing systems may also not accommodate air heater sootblowing concurrent with furnace sootblower operation. On air systems, additional sootblowing air compressors may be required. Steam sootblowing systems may be limited by make-up water capabilities or

feedwater heater capacities. Older units may be operating with sootblowing control equipment that lacks full automation and requires significant boiler operator attention. Consideration should be given to the state-of-the-art sootblowing control equipment with automated programming of a variety of sootblowing sequences, minimizing the operator interface requirements.

## 3.3.5 Combustion Air (Overfire Air Source)

Air required for the OFA system is usually taken from the combustion air system. It has been preheated and may have a sufficient velocity head for injection through the OFA nozzles into the furnace gases. Process design information will provide the necessary OFA flow and velocity head requirements. The existing combustion air supply system is reviewed in terms of fan capacity and available velocity head. Available velocity head can be increased by closing dampers that supply air to the primary combustion zone. However, the capacity of the forced draft fan(s) may be limiting. If capacity is available but the velocity head is not sufficient, a booster fan will be required for the OFA supply.

#### 3.3.6 FGR Source

FGR is used as an inert propellant for natural gas in the reburning process. Flue gas is drawn after the last heat transfer tube bank (economizer or boiler bank) as to not affect steam temperatures, and prior to the air heater since leakage there increases the oxygen level of the FGR. The configuration of the duct work leading from the boiler outlet to the air heater inlet should be reviewed with respect to a location for the flue gas tap. The tap should be located such that access to the center of the flue gas duct is possible to minimize tramp air entry. Tramp air (from casing leaks on balance draft units) and seal or cooling air from burners or other furnace water wall penetrations enters the flue gas and follows the furnace and duct walls. The flue gas tap should also be located to allow placement of a multi clone dust collector as close to the gas source as possible. Since the

gas is cleaned of particulate immediately after being extracted from the boiler exit, problems with ash accumulation and erosion in the FGR duct work are eliminated.

### 3.3.7 Equipment Footprint

Installation of GR-SI systems will require the placement of equipment, duct work and piping in a boiler house that may already be space limited. Following is a list of major GR-SI equipment, duct work and piping for which space requirements should be considered in a GR-SI retrofit:

#### GR System

- FGR fan, cooling fans, multi clone ash collector, flow measuring venturi, and interconnecting duct work.
- Natural gas metering, control, and shut-off valve station, and supply, distribution and vent piping.
- OFA duct work and booster fan (if required).

#### SI System

- SI air fan, cooling fans, and interconnecting duct work.
- Sorbent transport piping, flow splitter, and distribution hose layout.

# Ancillary Equipment

- New electrical power transformers and motor control centers
- Duct work modifications, air compressors, and water pumps for ESP enhancement humidification systems or chemical flue gas conditioning systems.
- Sootblowing air compressor additions.
- Dry ash handling equipment versus wet sluice systems.

## 3.3.8 Unit Operation

The location of furnace openings for SI is highly dependent on the furnace temperature profile which changes significantly with load. Penetrations are therefore located based on the normal load of the unit. The temperature window for optimum sorbent utilization is approximately 2000 to 2250°F. Operating at higher loads will result in sintering of the surface of the sorbent particle, reducing gas-solid reactions and under-utilizing the sorbent. When operating at lower loads (and temperatures), the gas-solid reactions will continue but at a slower rate, and again, sorbent is under-utilized.

Based on the above, load-following units may not be best suited for upper furnace SI. Units which cycle between full or max load during peak hours, and some reduced or minimum load during off peak, may be fitted with two levels of SI nozzles (upper and lower) to accommodate the two distinct loads and resulting furnace temperature profiles. With any unit that is not base-loaded and to be retrofit with GR-SI technology, turndown capability must also be incorporated into the system design to accommodate the load variations. Turndown capability impacts the equipment and metering of process streams, sorbent flow and transport air requirements, SI air flow, natural gas flow, and FGR flow.

#### 3.4 Balance of Plant

## 3.4.1 Electrical Power Distribution

GR-SI process equipment may be supplied power from the plant's auxiliary power system. The existing capacity of the electrical distribution and control system must be reviewed in light of the process needs. The primary electrical power consumers in the GR-SI systems are injection and cooling fan motors, pump and compressor motors, and silo area heat and lighting. Critical equipment such as cooling fans, sootblower controls, boiler controls, and

turning gear should be supplied from motor control centers having redundant feeds to assure an un-interruptible supply.

#### 3.4.2 Plant and Instrument Air

GR-SI system equipment, controls, and instrumentation will require dry instrument quality air for control valve operation, and also seal and cooling air. Plant air will be required for cleaning the baghouses and bin vent filters used on the sorbent unloading and feed equipment. If the furnace is a positive pressure design, plant air will also be required for the aspirated boiler waterwall penetrations.

Existing plant and instrument air systems, including dryers, should be reviewed in terms of capacity and air quality (oil and water content), to determine if the needs of the proposed GR-SI equipment can be met. Air usage by the sorbent silo equipment (baghouses, cleaning, seal, and control air) represents a significant portion of the total GR-SI system air requirements. Because of the remote location of this equipment from the boiler house, consideration should be given to a dedicated compressor and dryer at this location.

#### 3.4.3 Controls

Process control equipment and instrumentation installed as part of the GR-SI system will undoubtedly be state-of-the-art digital equipment. A wide variety of boiler control equipment exists in use at utilities representing various generations of pneumatic, analog, and digital control equipment. Equipment installed on any one unit may be a mixture of these technologies, i.e., pneumatic, analog and digital field devices tied to microprocessor-based digital bench board equipment in the control room.

Consideration must be given to the control scheme for the new process equipment, especially in regard to interface capabilities relating to safety interlocks, firing control, and

safety trips. If the interface capability is present, the utility may opt to add the new process equipment to the existing control equipment provided necessary input/output space is available or can be added, or add additional control equipment for GR-SI operation which interfaces with the existing boiler controls. Particular attention must be paid to proper buffering and isolation of the two controls systems so that the integrity and reliability of the existing boiler control system is maintained, but the transfer of data is also maintained between systems to ensure proper control strategy. Lacking the proper interface capability, an upgrade of the entire control room equipment may be warranted.

Since the GR process relies on setting precise stoichiometric ratios in the main burner, reburn, and burnout zones, above average combustion air control methods are required. Control systems that operate from an air-to-coal curve (lbs. of air per lb. of coal) do not lend themselves well to retrofit of the GR technology. This control method does not make adjustments for changes in coal heating value, moisture content, and air density, and actual stoichiometric ratios may differ from those desired. Since with GR, natural gas replaces a portion of the coal input to the unit, complicated control schemes are required on units operating under an air-to-fuel curve. The addition of boiler O<sub>2</sub> trim into the air control scheme can overcome these awkward limitations and optimize the GR process. O<sub>2</sub> trim is provided by in-situ flue gas oxygen analyzers located at the boiler outlet. State of the art control systems allows O<sub>2</sub> trim to be biased for the oxygen not participating in the combustion and burn-out processes which enters the flue gas via SI air flow or other tramp air sources (wall box seal air or casing leakage).

# 3.4.4 Silo Footprint

A sorbent storage silo with capacity for several days of operation cannot be erected within a typical boiler house, and is therefore located at ground level at the nearest available site. SI transport systems use dilute phase conveying to simplify distribution of the sorbent to the several injection nozzles. Aside from the sorbent flow rate, the length of the conveying

hose from the silo to the boiler-contributes to the conveying system pressure drop and dictates the size (horsepower) of the conveying blower or compressor.

Silo product load, as well as wind load and seismic design requirements, dictate a heavy silo foundation, usually supported by structural steel or pressure grouted pilings. Site selection must consider constraints on placement of the foundation, including possible relocation of underground facilities.

#### 3.4.5 Particulate Control

Control of particulate emissions from utility boilers is accomplished by baghouses and electrostatic precipitators (ESPs), the latter being the most used on larger units. With regard to baghouses, the sole consideration is the increase in particulate mass to be collected (up to forty percent). If not already equipped, a multi-cyclone collector (multi clone) may be added upstream of the bag house to collect the larger particle size constituents of the ash, reducing the loading on bag house. Based on the sizing and design of the existing bag house, it may be necessary to modify the bag house by adding additional compartments.

Operation of an existing ESP will be significantly impacted by the SI process. Impacts include increased particulate loading by a factor of two, a dust resistivity increase of a factor of one hundred due to decreased sulfur trioxide concentrations in the flue gas, and further increases in resistivity with increases in flue gas temperature as the boiler and air heater become fouled with sorbent. Unless the existing ESP is generously sized, some type of precipitator enhancement or expansion will be required. Precipitator enhancement technologies include chemical addition to the flue gas such as sulfur trioxide or ammonia gas, or cooling of the flue gas via duct humidification for reduction of ash resistivity. EER selected humidification over chemical treatment for enhancement of marginally-sized ESPs, since it was speculated that only humidification could maintain baseline emissions

and had a higher probability of-success. The capital and operating costs for addition of electrical fields to a precipitator must be compared to the alternative of chemical treatment or humidification. It must be noted that humidification requires at least two seconds of residence time in a straight run of duct to ensure complete evaporation of the introduced water spray. Duct work modifications, as well as possible relocation of draft fans, must be considered with the cost of humidification equipment as compared to chemical treatment or precipitator modifications.

## 3.4.6 Ash Handling

Depending on the boiler design (tangential, wall, or cyclone fired), SI can potentially increase the particulate loading to the ash handling equipment by as much as forty percent. It should be noted that since the SI process takes place in the upper furnace, it does not significantly impact bottom ash handling requirements. Due to the increased flue gas particulate mass to be handled, the utility considering SI should review the capacity of its existing economizer, air heater, and precipitator hoppers. Special consideration should be given to hoppers which experience flow problems, possibly being modified in design with steeper slopes, flow liners, or flow aids such as air cannons or vibrators.

Ash handling systems should be reviewed in terms of capacity (especially when servicing several units), general reliability, and required operator attention. Consideration should be given to automated ash handling control retrofits in lieu of additional operating labor requirements.

Handling systems which use a sluice water eductor (hydroveyor) have experienced problems with cementitious deposits in the throat of the eductor. Here, dust passing through the collection device mixes with water to form a cement and accumulates to the point that the necessary vacuum to operate the system becomes no longer possible. Ash handling system vendors have encountered this problem with the high calcium ashes of

Western lignites, and have designed rams or scrapers to manually clean these systems without disassembly.

#### 3.4.7 Ash Disposal

The GR process has no detrimental effect on the quality of the fly ash (carbon-in-ash) with respect to the ability to sell the ash, nor does it affect the chemical composition. SI however, adds a significant portion of partially sulfated sorbent to the total make-up of the ash product. With regard to wet (sluice) disposal practices, the pH of the sluice water is increased dramatically (10 to 13) with the addition of the partially used sorbent. Most state and federal environmental regulatory agencies limit the pH of discharge from ponds or settling basins to a range of pH 6 to pH 9. Carbon dioxide injection systems have been successfully used to neutralize sluice water in the conveying piping. Further treatment may be necessary at the pond out-fall due to continued chemical reactions taking place in the settling pond(s).

Design of ash retention ponds must be reviewed with respect to the increased particulate mass to be accommodated as well as changes in the chemical properties in the ash. Such a review should include pond sizing, settling time requirements, liners, leachate collection, and monitoring wells.

Facilities that have existing dry ash handling equipment, or are considering retrofit in light of the ash changes and disposal requirements, may consider sales of the high-calcium fly ash to cement companies that have used the ash as an admixture for cement and concrete. Design and permitting of existing dry ash disposal landfills may also prohibit disposal of the fly ash produced with SI, and disposal alternatives may be required.

#### 4.0 SYSTEM OPERATION ---

# 4.1 <u>Control System</u>

Control and monitoring of the GR-SI system is not complicated and may be accomplished with any modern control system that can be integrated into an existing boiler control system. The design of the GR-SI control system is based on the following criteria:

- All normal operations that are required to start, stop, or modulate the various pieces of equipment shall be performed in the control room.
- Sufficient information shall be displayed in the control room to enable the
  operator to determine the status of all equipment. The operator interface
  shall be designed so that the above information is displayed in a manner to
  enable rapid understanding of system status.
- Certain operations shall be interlocked to prevent inadvertent operation of equipment when such operation may present an operating hazard or other undesirable condition.
- Certain shut-down procedures shall be initiated automatically by the control system when such operations are deemed necessary for safety or good operating practice.
- Microprocessor based technology shall be used for the controls and interlocks.
- Operator interface shall be of the Keyboard-CRT type with custom graphics.
- The system will readily interface with existing plant instrumentation and be of a design that will enable operator familiarity and understanding with a minimum of training.

Interlocks are included which are designed to start the equipment in an orderly fashion and prevent the operator from allowing the unit's safety to become compromised either through erroneous operation or due to equipment failure. All major commands issued by the

control system are verified by a feedback signal. Trip signals are continuously monitored and will prevent startup or shutdown of equipment already in operation.

## 4.2 <u>Operation</u>

## 4.2.1 Gas Reburning System

The GR system is composed of three integrated systems: (1) natural gas injection, (2) FGR, and (3) OFA injection. The natural gas flow rate is controlled to the desired value for optimum NO<sub>x</sub> destruction. The FGR flow is controlled at a rate that provides adequate natural gas momentum for optimum mixing in the furnace. The OFA is controlled to a rate to complete combustion of all unburned fuel leaving the reburning zone. The three integrated systems are interlocked, operated and monitored by the control system.

The control logic for natural gas injection consists of a flow controller which receives a calculated set point from the boiler master and the natural gas flow transmitter. A comparison is made in the fuel controller between the set point and feedback signals and the controller output modulates the natural gas control valve to reduce any error to zero. The boiler master controls gas flow with coal flow to obtain the Btu input needed over the load range. A percentage of the boiler master signal is calculated and becomes the set point for the desired natural gas flow.

The desired FGR flow control set point is a calculated value determined from the boiler master signal. This set point signal is compared with the actual value of FGR flow rate in a PID controller which acts on any detected error signal. The control system will automatically adjust the FGR fan to reduce the error to zero.

Control of the OFA system consists of sending a set point signal calculated from the boiler master signal to a controller where it is compared with the total of the OFA air flows. The OFA nozzles are modulated to-reduce any detected difference in the set point and total OFA flow to zero. The control system compares the signals from the OFA flow transmitters to balance the flow of air.

Another control feature of the GR system is the cross limit between the OFA flow and natural gas flow. The set point for natural gas is compared with the OFA flow. If the natural gas flow set point is greater than the amount of OFA flow required for complete combustion of natural gas, the control system will decrease the natural gas set point to a value that permits complete combustion of the natural gas by the OFA. If the natural gas flow is greater than the OFA flow, the set point signal for OFA is increased to a value that will permit complete combustion of the natural gas. The above sequence is called cross limiting between the fuel (natural gas) and OFA and is very similar to the cross limiting features in the main combustion control between the coal feed and secondary air flow.

There is another cross limit between the FGR flow and the natural gas flow. If the FGR flow falls below a value that insures optimum penetration of the natural gas into the boiler (i.e., good mixing with the products of the coal combustion process), the set point for natural gas flow will be reduced to a safe value.

## 4.2.2 <u>Sorbent Injection System</u>

The SI system has three (3) variables that are modulated by the control system to obtain a targeted sulfur emissions reduction while maintaining maximum sorbent utilization. These include a rotary feeder to control sorbent feed rate, a control valve to control sorbent transport air flow, and a control damper to control injection air flow. The operator set point for SO<sub>2</sub> is the value that is compared with the input from the flue gas SO<sub>2</sub> analyzer in a PID controller. The output of the PID controller is used as a multiplier for the "total" coal flow rate to give a sorbent flow demand rate. This calculated value for sorbent flow demand rate becomes a set point value for two control loops. The first control loop, sorbent flow

demand rate, sets the initial value-for sorbent feed rate controlled by the rotary feeder. In this loop a set point is compared with the actual measured value of sorbent feed in a PID controller. The output of the PID controller modulates the speed of the rotary feeder.

As demand for sorbent increases or decreases depending on sorbent flow demand, there is a also a requirement to increase or decrease the sorbent transport air flow. Therefore, in the second control loop the calculated sorbent flow demand rate set point is compared with the sorbent transport air flow in a PID controller which modulates the air flow control valve to increase or decrease the amount of air flow as required.

The last control loop regulates the amount of injection air flow required to inject the sorbent into the boiler. Any error between the set point and actual flow rate is acted on by the PID controller whose output is summed with the initial value of the control signal to the injection flow control damper, thereby reducing any error to zero.

## 4.3 Optimization

# 4.3.1 Gas Reburning System

Optimization of the GR system is performed using a series of parametric tests to characterize the independent reburning variables and associated responses of the system at various boiler loads. By using this data, the appropriate set points can be established for a range of NO<sub>x</sub> emissions reductions. Prior to optimization, baseline tests are performed in order to establish both the pre- and post-installation boiler conditions without GR in operation. Five independent variables are involved in the parametric tests including:

 <u>Boiler load</u> -- A sufficient number of load conditions must be tested to develop the curve generators for the control system that enable automatic load-following.

- Percentage of total heat input proportioned to natural gas The coal flow is reduced in direct proportion to the natural gas injected into the reburning zone.
- Percentage of total flue gas used in FGR The FGR system is used to provide momentum to the injected natural gas for optimum mixing with the boiler flue gas. The level of FGR directly impacts the NO<sub>x</sub> conversion capabilities of the system.
- <u>Percentage of total combustion air used at OFA</u> OFA impacts the ability to burnout the remaining combustibles in the flue gas.
- Primary zone stoichiometric ratio (SR<sub>1</sub>) Low SR<sub>1</sub>'s are optimal for NO<sub>x</sub> reduction. However, the utility must establish the lower limit of SR<sub>1</sub> that minimizes the potential for corrosion in the bottom of the boiler. Flame appearance must also be acceptable. For cyclone boilers, there will be little change in SR<sub>1</sub> due to the operational fuel-to-air ratio constraints characteristic of cyclones. The optimum SR<sub>1</sub> will be in the range of 1.05 to 1.15, depending on boiler type.

# Dependent variables include:

- Reburning zone stoichiometric ratio (SR<sub>2</sub>) SR<sub>2</sub> is directly proportional to the natural gas heat input for an established SR<sub>1</sub> condition. At a zero gas condition, SR<sub>2</sub> is equal to SR<sub>1</sub>. As gas is introduced, SR<sub>2</sub> decreases. The optimum level of SR<sub>2</sub> is around 0.90.
- <u>Burnout zone stoichiometric ratio (SR<sub>3</sub>)</u> All combustion air not used in the primary zone becomes OFA. Depending on the excess air level selected by the utility, SR<sub>3</sub> will be approximately 1.15.

Stoichiometric ratios are calculated using boiler data collected during the tests. Additional test data are used to measure boiler emissions and assess operating characteristics. The following measurements are used:

- <u>Stack emissions</u> These include NO<sub>x</sub>, O<sub>2</sub>, CO, and CO<sub>2</sub>.
- <u>Control room data</u> Data are used to calculate the stoichiometric ratios, thermal efficiency and heat absorption.

- <u>Coal samples</u> Samples are evaluated to determine coal fineness, composition and volatility.
- Ash samples Samples are evaluated to determine carbon-in-ash and loss of ignition.
- <u>In-furnace measurement</u> HVT tests are used to characterize the temperature and flow stratifications in the boiler for comparison with process design models. The HVT is also used to assess CO distribution.
- <u>Visual observation</u> The potential for slagging and fouling of boiler tubes and other areas of the boiler are assessed.

Parametric testing is performed using a pre-planned test matrix. The matrix involves various combinations of the five dependent variables listed above to determine the effects on NO<sub>x</sub> emissions and other boiler responses. Evaluation of these results plus consideration of any unique boiler operating constraints are required to approximate the optimal set points for reburning operation. Additional tests are performed using minor adjustments to the dependent variables to fine tune the system. Once the set points are established for various load conditions, the data are entered into the control system providing for an automatic load following capability.

# 4.3.2 <u>Sorbent Injection System</u>

Optimization of the SI system is performed with a series of parametric tests in order to characterize the independent variables and associated responses of the system at different boiler loads. By using this data and performing additional optimization tests, the appropriate set points can be established for a range of SO<sub>2</sub> emissions reductions. Prior to optimization, baseline tests are performed to establish both the pre- and post-installation boiler conditions without SI in operation. Five independent variables are involved in the parametric tests including:

- <u>Boiler load</u> A sufficient number load conditions must be tested in order to develop the curve generators for the control system that enables automatic load-following operation.
- <u>Calcium-to-sulfur ratio</u> This ratio is proportional to the reduction in SO<sub>2</sub> emissions.
- Percentage of total combustion air used as injection air Injection air provides momentum to the sorbent when injecting it into the furnace to enhance mixing.
- Sorbent type Significant characteristics include the method of preparation, surface area, particle size, and density. These factors can have an impact on the reduction in SO<sub>2</sub> emissions. Each sorbent has unique handling characteristics, some of which may not be suitable for specific transport equipment.
- <u>Sootblowing cycle</u> The degree of sootblowing carried out impacts the boiler steam temperature and efficiency.

The following measurements are used to perform an SI evaluation:

- Stack emissions These include SO<sub>2</sub>, O<sub>2</sub>, and opacity.
- <u>Control room data</u> Data are used to calculate the thermal efficiency and heat absorption.
- <u>Electrostatic precipitator performance</u> The removal efficiency of the ESP is evaluated to determine if the equipment has a sufficient number of fields to remove the additional particulate resulting from SI.
- <u>Visual observation</u> The potential for slagging and fouling of boiler tubes, air heater and other areas of the boiler are assessed.

Parametric testing is performed using a pre-planned test matrix. The matrix involves various combinations of the five dependent variables listed above to determine the effects on SO<sub>2</sub> emissions and other boiler responses. Evaluation of these results plus consideration of any unique boiler operating constraints are required to approximate the optimum set points for SI operation. Additional tests are performed using minor adjustments to the dependent variables to fine tune the system. Once the set points are

established for various load conditions, the data are entered into the control system permitting automatic load following capabilities.

# 4.3.3 <u>Gas Reburning-Sorbent Injection System</u>

The optimum conditions for GR with SI may be different from those without SI. Therefore, another of series of tests are required to simultaneously optimize the combined GR-SI system. Cross-over effects of GR and SI include the following:

- GR results in a measured reduction in SO<sub>2</sub> through the replacement of SO<sub>2</sub>-laden coal with natural gas. The benefit is directly proportional to the gas rate used.
- GR results in a measured reduction in particulate, proportional to the amount
  of natural gas used. This results in an increased margin for particulate
  control with the ESP.
- Natural gas in the GR process adds moisture to the flue gas, that decreases
  the resistivity of the particulate. Offsetting this may be the reduction in sulfur
  trioxide formation due to less coal sulfur entering the furnace. Lower
  resistivities, within a range, are beneficial for the removal of spent sorbent in
  the ESP.

## 5.0 TECHNOLOGY PERFORMANCE

This section presents the results of field GR-SI parametric testing conducted at Illinois Power Company's Hennepin Unit #1, located in Hennepin Illinois, and City Water Light and Power's Lakeside Unit #7, located in Springfield, Illinois. The GR-SI process involves the injection of reburning fuel (in this case natural gas) into the furnace above the coal burners for reduction of nitrogen oxides (NO<sub>x</sub>) and injection of dry sorbent into the upper furnace for capture of sulfur dioxide (SO<sub>2</sub>). The parametric test results were selected for presentation here to show the effect of various process parameters on GR-SI performance.

During field evaluations, a wide range of boiler operating conditions and GR–SI parameters were tested to determine the optimum conditions for  $NO_x$  and  $SO_2$  emission reductions. The project goals of 60 and 50% reductions in  $NO_x$  and  $SO_2$ , respectively, were achieved for both demonstrations. The two GR-SI demonstrations are discussed sequentially and for discussion purposes the GR and SI system process performances are presented separately followed by brief comments and data on the integrated GR-SI systems.

# 5.1 Hennepin Unit #1 GR-SI Demonstration

A GR-SI demonstration was completed on Illinois Power Company's Hennepin Unit #1. It is a 71 MW<sub>e</sub> (net) tangentially fired utility boiler and the primary fuel is an Illinois Bituminous coal containing approximately 3 wt% sulfur. The goals of the project were to reduce emissions of  $NO_x$  and  $SO_2$  by 60 and 50%, respectively.

#### 5.1.1 GR Results

To fully evaluate the impact of GR on boiler emissions, Unit #1 baseline  $NO_x$  and CO emissions were measured and compared with average GR results, see Figure 5-1. During baseline tests,  $NO_x$  emissions of up to 0.77 lb/10<sup>6</sup>Btu (331 mg/MJ), were measured at full

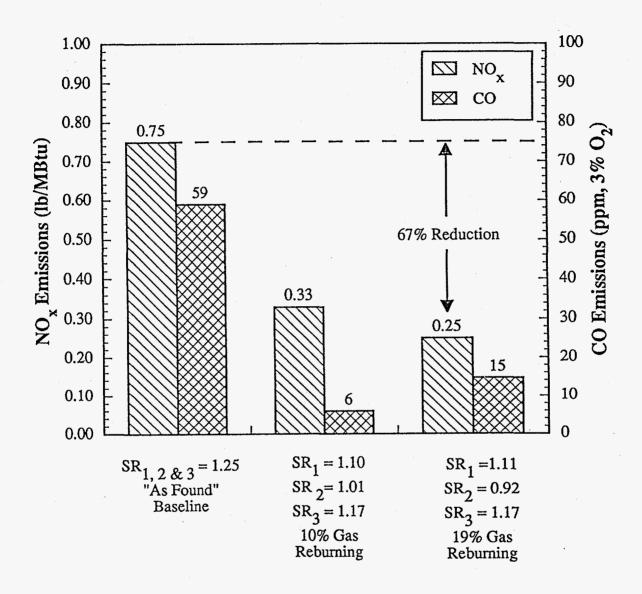


Figure 5-1. Baseline and Gas Reburning NOx and CO emissions.

load with excess air levels of-up to 25%. Lower  $NO_x$  emissions were observed at decreased boiler loads. At 60 MW<sub>e</sub>,  $NO_x$  emissions averaged approximately 0.48 lb/10<sup>6</sup>Btu (206 mg/MJ). For comparison purposes, 0.75 lb  $NO_x$  /10<sup>6</sup>Btu (323 mg/MJ) was used as the baseline for all full load tests. The GR optimization tests showed  $NO_x$  emission averages that ranged from a low of 0.19 lb/10<sup>6</sup>Btu (81 mg/MJ) to a high of 0.36 lb/10<sup>6</sup>Btu (157 mg/MJ), operating with gas heat inputs ranging from 10 to 22%, primary zone stoichiometric ratios ranging from 1.06 to 1.15, and reburn zone stoichiometric ratios ranging from 0.88 to 1.01.

GR was operated with a furnace exit excess air level of 15-18% compared to the baseline level of 20 to 25%. At Hennepin, the GR system was designed to maximize  $NO_x$  emission reductions for an 18–20% reburn gas heat input. The goal was to achieve a minimum of  $60\% NO_x$  reduction. Figure 5–1 shows that the  $NO_x$  reduction goal was achieved.

The GR system was optimized by varying the key process parameters that affect  $NO_x$  emissions. One of those parameters is the primary zone air/fuel stoichiometric ratio (SR). The primary zone SR established for the Hennepin Unit was in the range of 1.08 to 1.10. Further testing established optimum conditions for FGR, burner tilt, and reburning zone SR to reduce  $NO_x$  emissions to 0.25 lb/10<sup>6</sup>Btu (108 mg/MJ) or less. Following is an analysis of the major parameters that were varied to optimize the GR  $NO_x$  reduction performance:

- Zone stoichiometric ratios
- Percentage of FGR
- Boiler operational parameters

# 5.1.1.1 Zone Stoichiometric Ratios

The three combustion zones of the reburning process (primary, reburn, and burnout) were analyzed. The effect of primary zone SR on NO<sub>x</sub> emissions under full load GR operation

with 18% gas heat input, is illustrated in Figure 5-2. During testing of this variable, the primary zone SR was operated over the range of 1.07 to 1.11.  $NO_x$  emissions increased from an average of 0.21 lb/10<sup>6</sup>Btu (90 mg/MJ) at an SR of 1.07 to 0.26 lb/10<sup>6</sup>Btu (112 mg/MJ) at an SR of 1.11. The increase in  $NO_x$  emissions was due to two factors: higher  $NO_x$  formation in the combustion zone and a slight increase in the reburning zone SR which reduced the  $NO_x$  control level. During this test, CO emissions were maintained at a maximum of 3 ppm, indicating complete fuel burnout was achieved.

Fly ash samples were collected from various full load tests to determine if low operating primary zone stoichiometric ratios would increase carbon loss. At a primary zone SR above 1.06, carbon in fly ash varied from 2.2 to 5.2% as compared to the baseline value of 2.6%. Operating the primary combustion zone at an SR of 1.07 provided sufficient combustion air to provide for good carbon burnout.

The reburning zone SR is the most important parameter regarding NO<sub>x</sub> reduction. The primary zone SR sets how much reburning fuel is needed to reduce the reburn zone to its optimum SR of 0.90. In most cases, reburning fuel accounted for 18 to 20% of the total heat input into the furnace. Figure 5–3 shows the impact of reburning zone SR on NO<sub>x</sub> emissions. For operation with a primary zone SR between 1.07 and 1.13, NO<sub>x</sub> emissions were 0.22 lb/10<sup>6</sup>Btu (95 mg/MJ) at a reburning zone SR of 0.90. In addition, CO emissions were less than 25 ppm.

The burnout air (OFA) completes the combustion process and brings the combustion system to its normal operating exit flue gas condition SR. In practice, GR allowed the system to operate at lower overall excess air levels. Figure 5-4 shows the NO<sub>x</sub> emissions levels achievable under baseline operation, staged combustion only without GR, and GR operation. Baseline tests with overall excess air levels ranging between 13 and 21% resulted in NO<sub>x</sub> emissions ranging from 0.52 lb/10<sup>6</sup>Btu (224 mg/MJ) to 0.77 lb/10<sup>6</sup>Btu (331

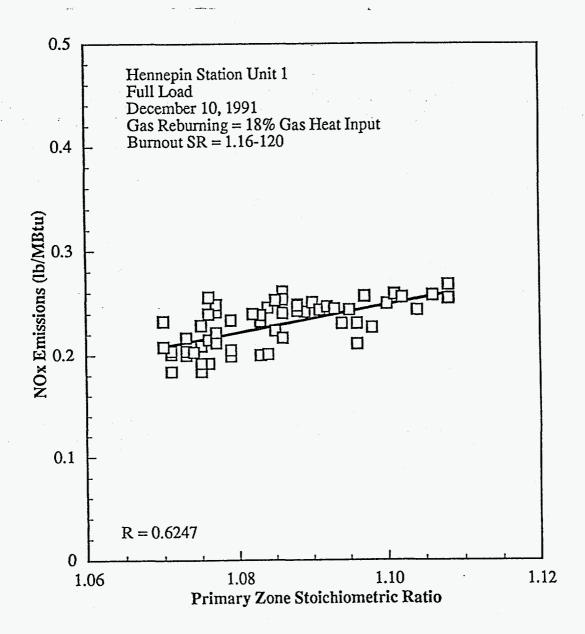


Figure 5-2. Effect of primary air on  $NO_x$  emissions

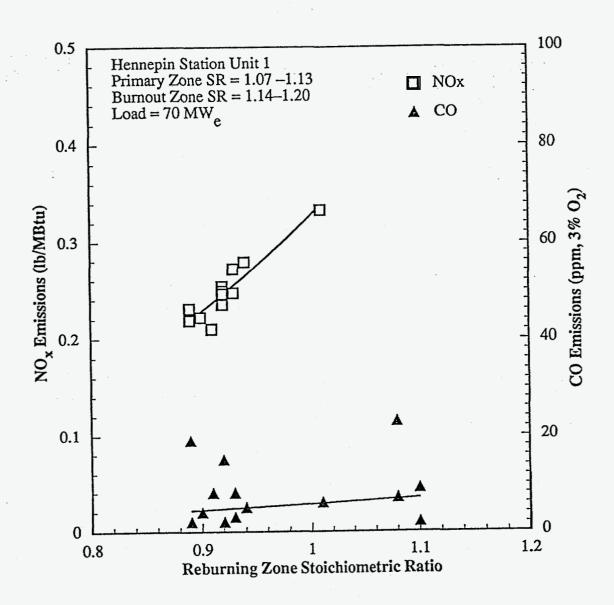


Figure 5-3. Effect of reburning zone stoichiometry on NOx emissions

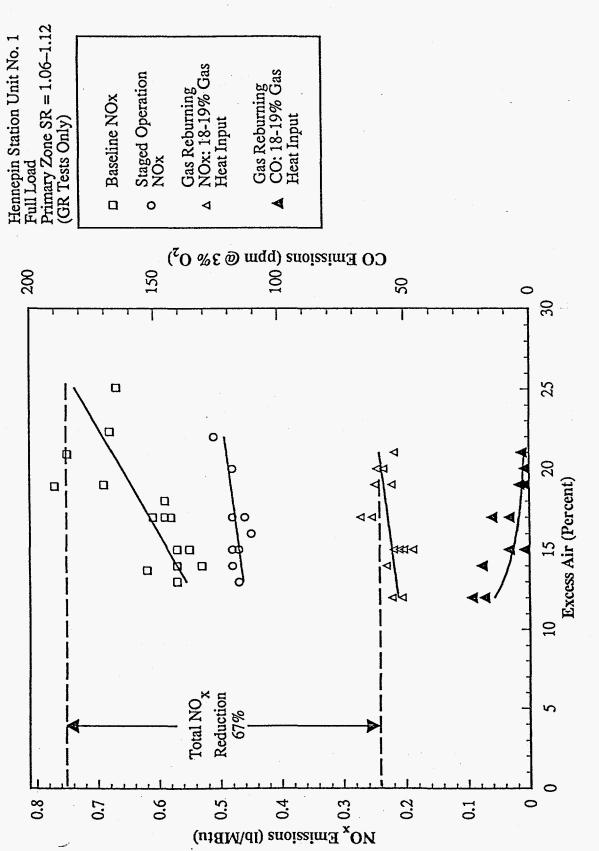


Figure 5-4. NOx reduction and CO emissions with Gas Reburning

mg/MJ). Staged combustion without GR showed  $NO_x$  emissions ranging from 0.46 lb/10<sup>6</sup>Btu (198 mg/MJ) to 0.51 lb/10<sup>6</sup>Btu (219 mg/MJ) or  $NO_x$  reductions ranging from 32 to 38% less than the baseline value at normal excess air conditions. GR was run with overall excess air levels ranging from 12 to 21%. The total  $NO_x$  reduction from the uncontrolled level, when utilizing 18% excess air at the burnout zone, was 0.25 lb/10<sup>6</sup>Btu (108 mg/MJ) or approximately a 67% reduction in  $NO_x$ .

# 5.1.1.2 Flue Gas Recirculation

In the GR optimization test series, the focus was to optimize mixing by varying the FGR flow rate. In this report, FGR flow rates are expressed as a percentage of the total flue gas flow exiting the furnace.

To determine the effect of mixing on  $NO_x$  emissions, tests were run with three levels of FGR at full load and with primary SR values ranging from 1.08 and 1.12. Figure 5-5 illustrates the effect of FGR on  $NO_x$  emissions performance as a function of the reburn zone stoichiometric ratio. Satisfactory mixing performance was evident for FGR levels between 2.6 and 3.6%. As the reburn fuel rate was increased, the difference in NOx reductions achieved for 2.6 and 3.6% FGR flow was insignificant. However, lower FGR flows had a slightly negative impact on  $NO_x$  emissions performance. Hennepin Unit #1 had a good reburn zone residence time (0.55 sec) and this minimized the need for FGR. For units with 0.50 second and greater, enough residence time occurs so that the reburn  $NO_x$  reducing reactions can occur. With lower reburn zone residence times, FGR helps establish the reburn zone more quickly and thus a greater  $NO_x$  reduction is seen.

# 5.1.1.3 <u>Boiler Operational Parameters</u>

One of the main objectives of the project was to design a process that would be effective throughout the entire operating range of the unit for the purpose of load following on

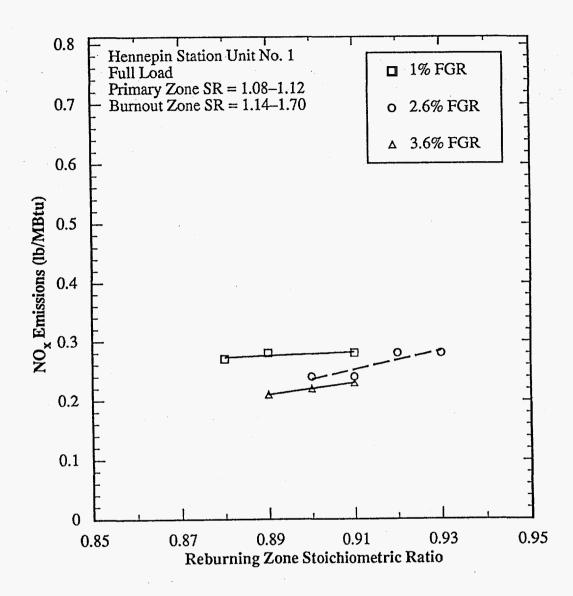


Figure 5-5. Effect of FGR on NO<sub>x</sub> emissions

dispatch control. Optimization tests were conducted at loads from 40 to 75 MW<sub>e</sub>. The boiler operational parameters that were evaluated included:

- Reburning Fuel Injector/Coal Burner Tilt Angle
- Boiler load
- Coal mills in service
- Furnace temperature profile

Reburning Fuel Injector/Coal Burner Tilt Angle The reburning fuel injectors were equipped with tilting mechanisms that allowed them to inject reburning fuel at the same angle as the coal burners. During the optimization test series, the reburning fuel injector tilts were tested through their full angle of rotation with no appreciable change in reduction of NO<sub>x</sub> emissions. The purpose of tilting reburning fuel injectors was to follow the primary burner tilts and thus the fire ball. Conceptually, this would allow a consistent separation between the two zones to allow combustion of the primary fuel to proceed nearly to completion, and provide the necessary reburn zone residence time. Figure 5-6 illustrates that reburning injector/coal burner tilting resulted in a negligible change in NO<sub>x</sub> emissions as the injector tilt angle varied from horizontal to -27 degrees at full load. The injector tilting capability was eliminated after optimization testing.

Boiler Load The impact of boiler load on GR performance is due to the primary NO<sub>x</sub> emissions level and gas temperature profile. As was observed in the baseline tests, baseline NO<sub>x</sub> levels varied from 0.75 lb/10<sup>6</sup>Btu (323 mg/MJ) at 75 MW<sub>e</sub>, to 0.48 lb/10<sup>6</sup>Btu (206 mg/MJ) at 60 MW<sub>e</sub>, to 0.50 lb/10<sup>6</sup>Btu (215 mg/MJ) at 45 MW<sub>e</sub>. Figure 5-7 illustrates the effect of load on NO<sub>x</sub> emissions. The baseline NO<sub>x</sub> emissions were not measured at constant excess O<sub>2</sub>; therefore some variations must be attributed to varying excess air levels. Data at 70 MW<sub>e</sub> have already been thoroughly discussed. At 60 MW<sub>e</sub>, GR performance is affected by lower primary NO<sub>x</sub> levels entering the reburning zone and lower

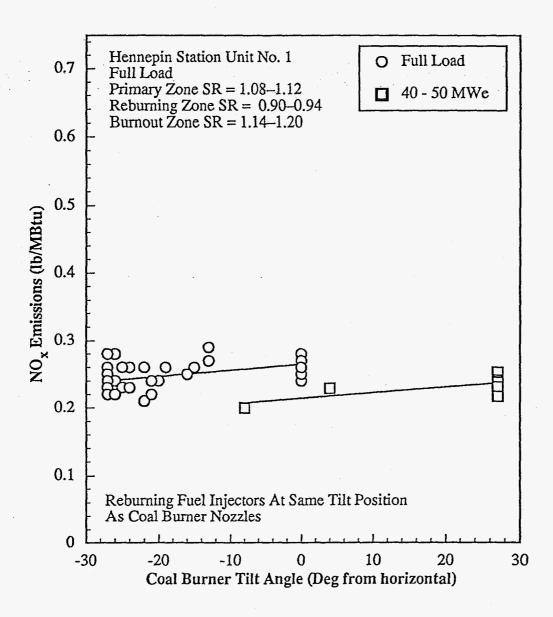


Figure 5-6. Effect of burner tilt on  $NO_x$  emissions

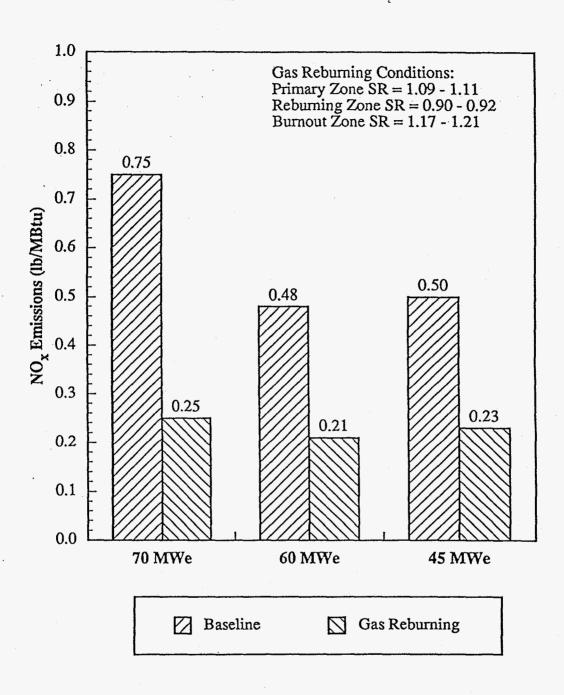


Figure 5-7. Effect of boiler load on  $NO_x$  emissions

furnace temperature. At low load, 45 MW<sub>e</sub>, the boiler was operated with 2 mills in operation and with the burner tilts raised to  $+27^{\circ}$  to maintain reheat steam temperature, similar to operation at 60 MW<sub>e</sub>. GR NOx reduction at 45 MW<sub>e</sub> was also lower due to a decreased reburning zone gas temperature and lower initial NO<sub>x</sub>.

<u>Mills in Service</u> At Hennepin, the normal mode of operation at loads below 55 MW<sub>e</sub> with GR is with two mills in service. The top row of burners is fed by Mill C, the middle row of burners is fed by Mill B, and the bottom row of burners is fed by Mill A. The effects of mills out of service at low load was evaluated without GR. Taking Mill C out of service, with no GR, resulted in a reduction in  $NO_x$  emissions due to combustion staging, see Figure 5-8.

Reburn Zone Gas Temperature Gas temperature measurements were taken to evaluate the impact of gas temperature on GR NOx reduction. GR is more effective at higher reburning zone temperatures due to increased kinetics and more rapid formation of hydrocarbon fragments and free radicals. The reburning zone temperatures may reflect the primary zone temperature in indicating the completeness of primary zone coal combustion. Figure 5-9 shows NO<sub>x</sub> emissions as a function of reburning zone SR, at two reburn gas temperatures. As expected, the NO<sub>x</sub> emissions were slightly lower for the higher temperature case for SRs less than one (reducing conditions). The reverse is true for SRs greater than one (oxidizing condition).

#### 5.1.2 SI Results

Baseline SO<sub>2</sub> emissions averaged 5.24 lb/10<sup>6</sup>Btu (2,250 mg/MJ) over the test period and sorbent SO<sub>2</sub> capture ranged from a low of 25.3% to a high of 60.6%. Figure 5-10 illustrates typical SO<sub>2</sub> emissions trends during GR and GR-SI operation. Initial baseline SO<sub>2</sub> emissions were taken prior to commencing GR for a period of 15 to 30 minutes. When reburning fuel was injected into the furnace, a second baseline level was established, prior to commencing SI. Once SI was initiated, SO<sub>2</sub> emissions began to decrease but did

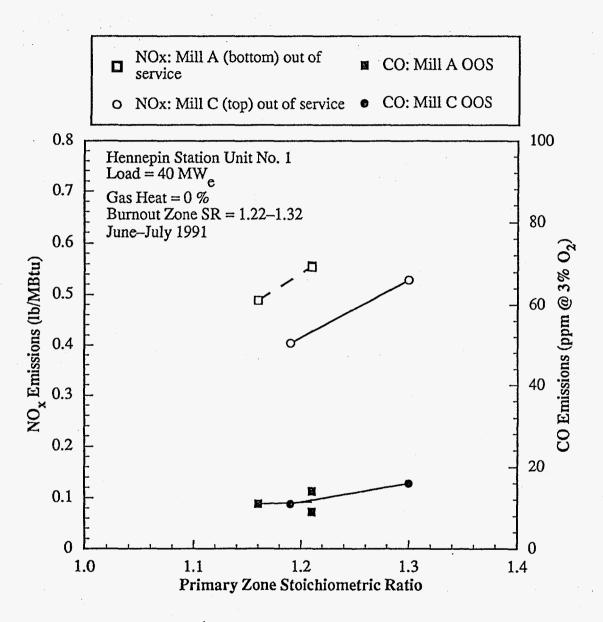


Figure 5-8. Effect of mills out of service at low loads

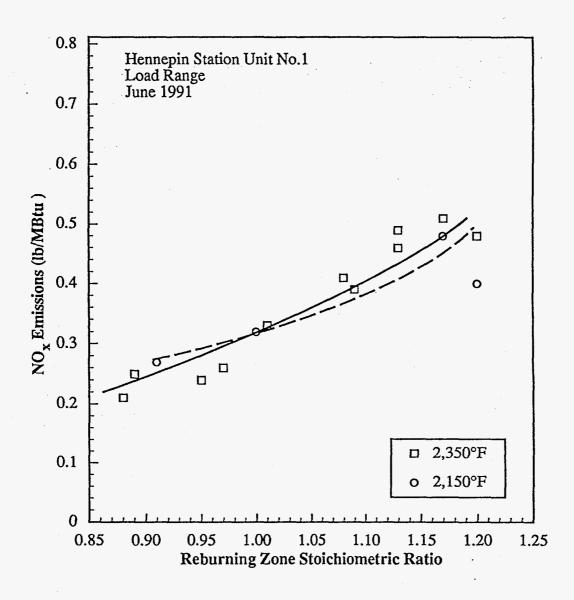


Figure 5-9. Effect of reburn zone SR and temperature on  $NO_x$  emissions

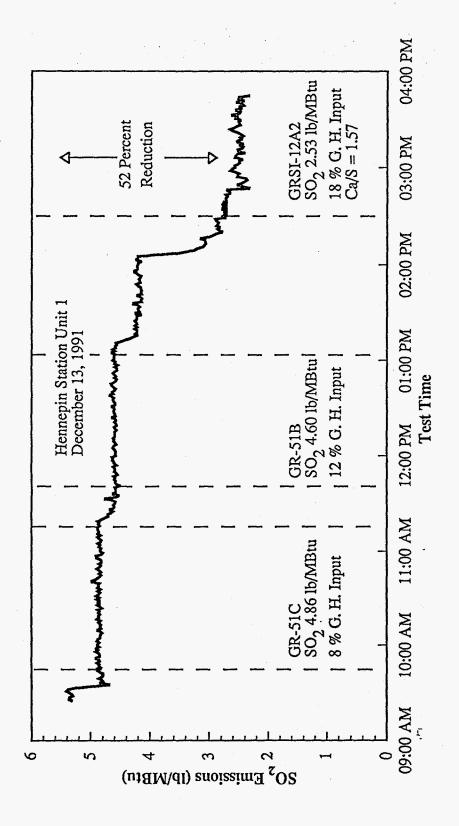


Figure 5-10. SO<sub>2</sub> emissions during GR and GR-SI testing

not reach steady state for about-30 minutes. Collection of test data did not begin until the steady state condition existed. GR tests showed the expected SO<sub>2</sub> reductions, at 8 and 12% reburn gas heat input compared to baseline, 8 to 13% SO<sub>2</sub> reductions occurred. The GR-SI test, showed an SO<sub>2</sub> reduction of 52.6%, when operating with 18% reburn gas heat input and a Ca/S of 1.57.

Studies of the SI process showed that six major parameters control the utilization of sorbent, and hence the cost effectiveness of the SI process. The parameters are:

- Sorbent reactivity
- Temperature
- Sorbent dispersion
- Ca/S molar ratio
- Injection configuration
- Boiler operational impacts

## 5.1.2.1 Sorbent Reactivity

Reactivity refers to a sorbent's ability to react with  $SO_2$ . The sorbents tested at Hennepin were all calcium hydroxide  $[Ca(OH)_2]$  based. Upon introduction into the furnace, the sorbent undergoes calcination to produce  $CaO[Ca(OH)_2 + heat - CaO + H_2O]$ . The CaO reacts with  $SO_2$  to form calcium sulfite  $[CaO + SO_2 - CaSO_3]$  and calcium sulfate  $[CaO + SO_2 + 1/2O_2 - CaSO_4]$ .

The reactivity of the CaO can be enhanced by altering the hydrating process used to form the calcium hydroxide. Also, the smaller the particle size, the greater the surface area and the better the reactivity. The three advanced sorbents tested in this project, were produced with promoting agents to increase the CaO reactivity.

# 5.1.2.2 <u>Temperature</u>

The furnace temperature at the point of sorbent introduction has a pronounced impact on CaO reactivity. Injection at high temperatures tends to sinter the surface of the CaO producing a low reactivity "dead burned" material. The optimum injection temperature for most sorbents is around 2,200°F (1,204°C). The temperature history of the sorbent following injection also affects its ability to capture SO<sub>2</sub>. The CaO reacts readily with SO<sub>2</sub> when injected; however, the reaction rate degrades as the temperature drops and is negligible at about 1,600°F (871°C). SO<sub>2</sub> control with SI is enhanced by long residence time in the 1,600°F to 2,200°F temperature window.

# 5.1.2.3 Sorbent Dispersion

Dispersion of sorbent throughout the complete flue gas stream is also important to ensure that the SO<sub>2</sub> has an opportunity to contact the sorbent particles for the longest possible time within the temperature window. EER's SI design methodology evaluated all of these factors. A combination of field measurements, heat transfer modeling and isothermal physical flow modeling was used to establish the furnace gas temperature and flow patterns. Optimum injection temperatures (and hence locations) were selected based on the sorbent characteristics measured in bench scale reactivity tests. Alternate arrangements of the sorbent injectors were evaluated. Physical flow modeling was used to finalize the SI design.

#### 5.1.2.4 Ca/S Molar Ratio

Extensive bench scale testing and computational modeling has shown that sorbent SO<sub>2</sub> capture efficiency is related to the SO<sub>2</sub> concentration in the flue gas and the Ca/S molar ratio. The effect of Ca/S on SO<sub>2</sub> control is shown in Figure 5-11 for three loads. Full load predictions are also shown for comparison and are in good agreement with the data. Since

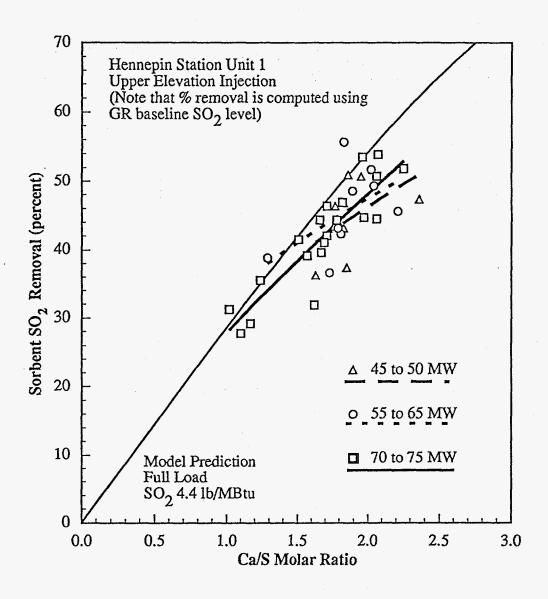


Figure 5-11. SO<sub>2</sub> reduction with GR-SI for three load ranges

the cost of SO<sub>2</sub> removal is dominated by the cost of the sorbent itself, the overall cost effectiveness increases with calcium utilization, expressed as the portion of the calcium which is reacted with sulfur. Calcium utilization is calculated as the percentage of SO<sub>2</sub> removal divided by the Ca/S molar ratio. Figure 5-12 shows the same data as Figure 5-11 re-plotted in terms of calcium utilization. The general trend found was that lower utilizations occur at high Ca/S molar ratio. This is consistent with bench scale tests and is caused by blinding of the active CaO as calcium sulfites/sulfates are formed.

# 5.1.2.5 <u>Injection Configuration</u>

In computational modeling studies, sorbent dispersion and  $SO_2$  concentrations were shown to have a significant effect on  $SO_2$  capture efficiencies. Sorbent dispersion is dependent on the penetration and coverage characteristics of the sorbent stream. It can be altered by varying the SI velocity and the injection configuration.

The SI injection system was designed with 6 jets to mix the sorbent uniformly across the furnace (4 on the front wall and 2 on the side walls). Figure 5-13 compares the performance in this configuration with that measured when the 2 side wall jets were out of service. Calcium utilization degraded significantly with these jets out of service.

# 5.1.2.6 <u>Effects of Boiler Operation</u>

Boiler operational parameters were also found to have impacts on the performance of the SI system. Those parameters are:

- Boiler load
- Burner tilt
- Excess air

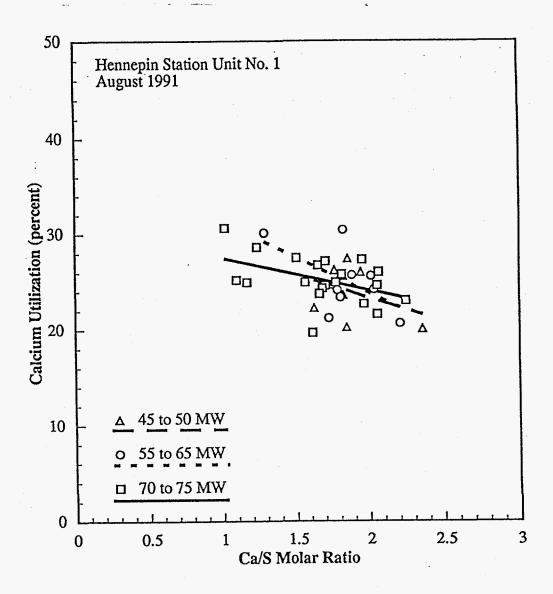


Figure 5-12. Calcium utilization with GR-SI for three load ranges

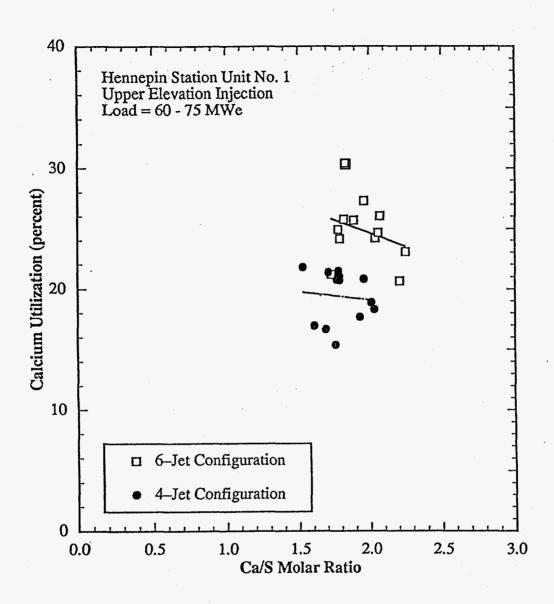


Figure 5-13. Effect of SI jet configuration on calcium utilization

Boiler Load As shown above, the SO<sub>2</sub> removal goal of the project was consistently achieved at full load using the upper level sorbent injectors. The SI system included a lower set of injectors at the OFA ports for low loads (< 45 MW<sub>e</sub>) when furnace temperatures decreased. Calcium utilization was similar using the upper and lower injectors over the load range.

Burner Tilt Tests at various burner tilt angles were conducted to evaluate the sensitivity of the temperature profile and its effect on sorbent SO<sub>2</sub> capture, particularly under lower load operation when burner tilt is adjusted to maintain reheat steam temperature. The midand full-load tests showed calcium utilization to be insensitive to burner tilt angles between -5 to +19 degrees. At low loads (45 MW<sub>e</sub> to 58 MW<sub>e</sub>) shown in Figure 5-14, calcium utilization was significantly affected in tests with burner tilts between -6 and +13 degrees, with the lower level sorbent injectors in service.

<u>Excess Air</u> The level of excess air had no effect on calcium utilization at full or reduced loads.

### 5.1.2.8 HCl and HF Emissions

Emissions of HCl and HF were also quantified in relatively short tests during baseline and GR-SI operation. Significant reductions in HCl and HF emissions due to GR-SI were measured. This is due to two processes: replacement of coal with natural gas (natural gas has no chlorine or fluorine) and the reaction of HCl and HF with the sorbent [ CaO + 2HCl  $\rightarrow$  CaCl<sub>2</sub> + H<sub>2</sub>O and CaO + 2HF  $\rightarrow$  CaF<sub>2</sub> + H<sub>2</sub>O). Reductions in HCl emissions in the range of 60.4 to 85.5% were measured during GR-SI operation at a gas heat input of 18.5 to 18.7% and a Ca/S molar ratio of 1.64 to 1.66. Reductions of HF emissions were in the 94.3 to 99.7% range. The emissions based on fuel fired were 9.29x10<sup>-3</sup> to 24.6x10<sup>-3</sup> lb HCl/10<sup>6</sup>Btu (3.99 to 10.6 mg/MJ) and 1.14x10<sup>-5</sup> to 19.1x10<sup>-5</sup> lb HF/10<sup>6</sup>Btu (4.90x10<sup>-3</sup> to 82.1x10<sup>-3</sup> mg/MJ). These may be compared to baseline levels of 6.71x10<sup>-2</sup> lb HCl/10<sup>6</sup>Btu

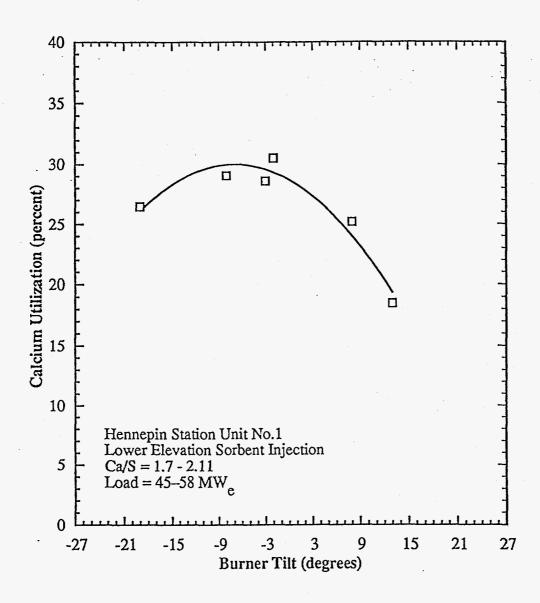


Figure 5-14. Effect of burner tilt on calcium utilization at low load

(28.9 mg/MJ) and 3.60x10<sup>-3</sup> lb HF/10<sup>6</sup>Btu (1.55 mg/MJ). CaF<sub>2</sub> and CaCl<sub>2</sub> are unstable at temperatures greater than 1150 to 1500°F (621 to 816°C). Therefore, calcium halide formation must take place after the sorbent has reacted with SO<sub>2</sub>. Emissions of HCl and HF on a yearly basis, with an assumed capacity factor of 32% (1992 capacity factor = 32.01%), would be reduced significantly by the GR-SI operation. HCl emissions would be reduced to a range of 6 to 22 tons per year (5.4 to 20.0 tonne/a) while HF emissions would be reduced to a range of 0.01 to 0.17 TPY (0.009 to 0.15 tonne/a). These are compared to baseline emissions of 70 tons HCl per year (64 tonne/a) and 3.8 tons HF per year (3.4 tonne/a).

## 5.1.3 GR-SI Results

The parametric tests showed that the GR-SI system installed on Hennepin Unit #1 not only achieved but exceeded the project goals of reducing  $NO_x$  and  $SO_2$  by 60 and 50%, respectively. Based on the results of the optimization tests, the following GR parameters were set for long term testing and at least a 60% reduction in  $NO_x$  emissions was achieved:

Percent Reburning Fuel	18
Primary Zone SR	1.08
Reburn Zone SR	0.90
Burnout Zone SR	1.18
Flue Gas Recirculation Flow	2,800 scfm (1.32 Nm³/s)

The following SI operating parameters were used to achieve a 50% reduction in SO<sub>2</sub> emissions:

Ca/S Molar Ratio	1.75
Injection Configuration	6 nozzles
Injection Elevation	Upper

# 5.2 <u>Lakeside Unit #7 GR-SI Demonstration</u>

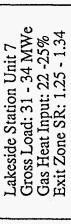
A second GR-SI demonstration was completed at City Water, Light and Power's Lakeside Station Unit #7, located in Springfield, Illinois. Unit #7 is a 33 MW<sub>e</sub> cyclone-fired unit that burns medium to high sulfur Illinois Bituminous coal. The project goals for this demonstration were the same as that for Hennepin Unit #1, to reduce  $NO_x$  and  $SO_2$  by 60 and 50%, respectively.

## 5.2.1 GR Results

The performance of the GR system in controlling  $NO_x$  and its impacts on other gaseous emissions including CO and  $SO_2$  are presented in this section. The program goal was to reduce  $NO_x$  by 60% at full load. To evaluate the GR system, parametric tests typically lasting one to two hours were conducted. Each process parameter was varied individually to evaluate its impact independent of the others. The major process parameters relevant to  $NO_x$  control with GR in this demonstration, that could be varied, included the SR of each zone (primary, reburn, and burnout), reburn fuel injector size, and FGR flow. The baseline  $NO_x$  for Lakeside Unit #7 was 0.97 lb/106Btu. This value was used as the base in determining the GR  $NO_x$  reductions.

### 5.2.1.2 Zone Stoichiometric Ratios

The stoichiometric ratios of the three zones significantly impact the  $NO_x$  control process. Limiting the primary (coal) zone SR limits the formation of  $NO_x$  in this high temperature zone. The impacts of the primary and reburning zone stoichiometric ratios at full load are shown in Figure 5-15. The data show that operation at a primary SR of 1.08 and reburning zone SR of 0.83 resulted in the highest  $NO_x$  reduction at full load (67%), while operation at a primary zone SR of 1.15 and reburning zone SR of 0.90 achieved the target  $NO_x$  reduction of 60%.



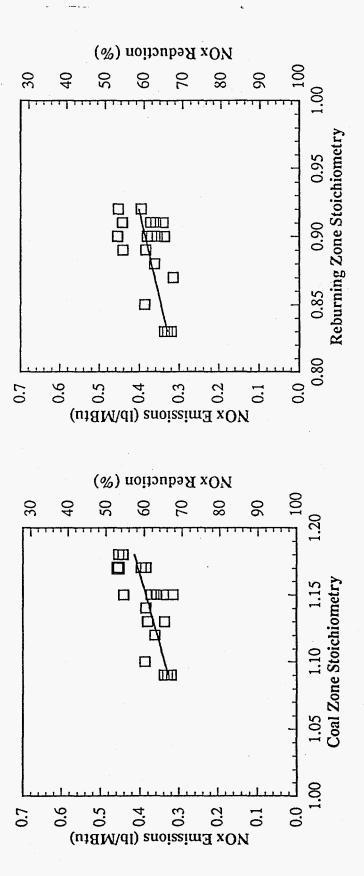


Figure 5-15. Effect of primary and reburn zone stoichiometric ratios on NO<sub>x</sub> emissions at full load

The impact of the burnout (exit)-zone SR on  $NO_x$  and CO emissions at full load is shown in Figure 5-16 and for mid and low loads is shown in Figure 5-17. Burnout air is used to complete combustion, therefore, lower CO levels are expected at higher exit zone stoichiometric ratios. At full load, an exit zone SR of 1.20 was expected to achieve fuel burnout. However, in practice an exit zone SR of 1.30 was needed to maintain CO emissions below 200 ppm. At mid and low loads, an exit zone SR of 1.35 was required. The exit zone SR has a relatively minor impact on the final  $NO_x$  level since the gas temperature at the point of OFA addition is relatively low. At mid and low load, there was no change in  $NO_x$  with excess air.

The fly ash carbon obtained during the baseline testing, yielded a range of 2% to 8% carbon. Under GR, the fly ash carbon content increased to 12 to 14% over the range of gas heat inputs of 15 to 25%. At full load, GR had a smaller effect, with fly ash carbon content increasing to 7 to 10%. While in pulverized coal-fired units an increase of 5% in carbon-in-fly ash results in a 1% increase in heat loss, for cyclone-fired units where 70 to 80% of the ash is removed in the cyclone barrels, an increase of 20% in carbon-in-fly ash results in a 1% increase in heat loss.

## 5.2.1.3 Reburning Fuel Injector Size

Two sizes of reburning fuel injectors were tested. The early tests were conducted with large injectors that had relatively low injection velocities. At the completion of several parametric tests, an evaluation of test data was undertaken that included furnace flow modeling. The evaluation showed that improved NO<sub>x</sub> reductions should be obtained with smaller diameter injectors. The smaller injectors were installed and used throughout the long-term GR-SI demonstration.

The impacts of injector optimization on  $NO_x$  emissions at full and mid load are shown in Figure 5-18. In both cases, the smaller injectors improved  $NO_x$  reduction. This is due to

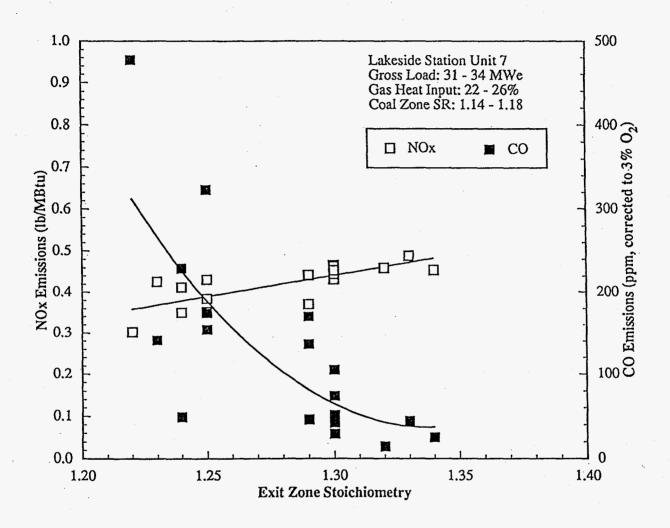


Figure 5-16. Effect of exit zone stoichiometry on  $NO_x$  and CO at full load

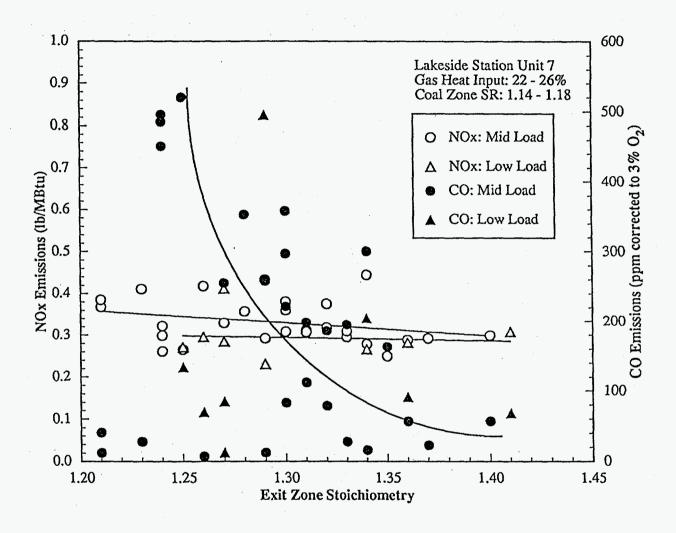
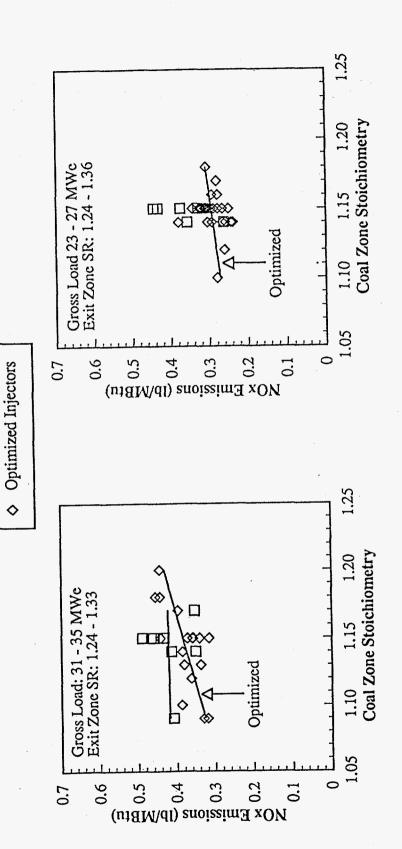


Figure 5-17. Effect of exit zone stoichiometry on  $\mathrm{NO}_{\mathrm{x}}$  and  $\mathrm{CO}$  at mid and low load



Lakeside Station Unit 7 Gas Heat Input: 22 - 26%

☐ Original Injectors

Figure 5-18. Effect of reburn fuel injector design on NO<sub>x</sub> emissions at mid and full load

improved reburning fuel jet mixing with the combustion products from the primary zone. On average,  $NO_x$  reductions improved by 3 to 5% with the smaller reburning fuel injectors.

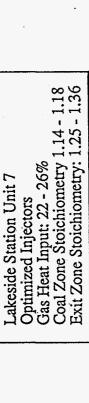
# 5.2.1.4 Recirculated Flue Gas

The FGR flow was varied widely, from 3000 to 6000 scfm (1.42 to 2.83 Nm³/s). FGR was used as a carrier gas to improve the mass flux of the reburning fuel jets and thereby reduce the mixing time. As expected, higher flows of FGR helped achieve the lowest  $NO_x$  level. This was most clearly the case at low load (19 to 20 MW<sub>e</sub>). The impacts of FGR, expressed as a percentage of total flue gas, at full, mid, and low loads are shown in Figures 5-19 and 5-20. At full load, FGR of 6 to 7% achieved optimum results, while at mid load 8 to 9% achieved highest  $NO_x$  reduction, and at minimum load 9 to 10% was optimum. The reburn zone residence time in this application was only 0.25 seconds, whereas in the Hennepin demonstration it was 0.55 seconds. From comparison of the two demonstrations it is seen that shorter reburn zone residence time units require increased FGR to achieve near optimum NOx reduction.

#### 5.2.2 SI Results

The performance of the SI system was initially evaluated with parametric SI-only tests. This was followed by a co-application of both GR and SI technologies over the long-term testing period. The parameters which impact SO<sub>2</sub> capture in SI are the Ca/S molar ratio, the SI air flow, and the injection temperature (and indirectly the load). Sorbent characteristics, such as type (hydrate or carbonate) and fineness also impact SO<sub>2</sub> capture.

Furnace SI was developed to achieve SO<sub>2</sub> reductions in the 25 to 50% range. When combined with GR, higher SO<sub>2</sub> reductions occur due to coal replacement with natural gas. Limited SI-only testing was characterized to optimize the process. The process was evaluated over the full load range with Ca/S molar ratios from 1.0 to 3.0. SI air flows were



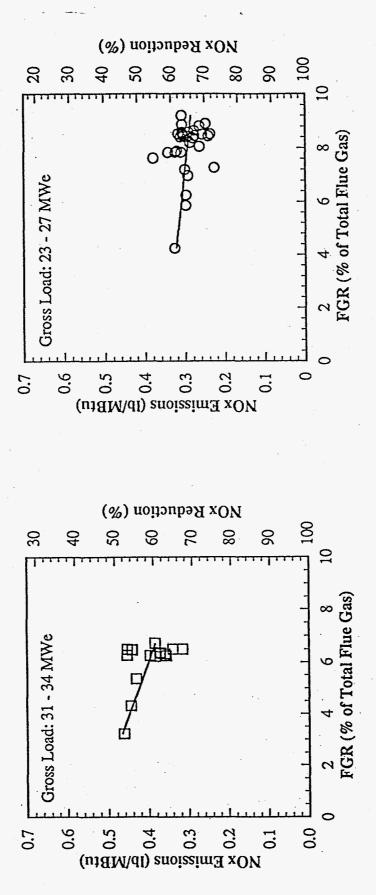


Figure 5-19. Effect of FGR on NO<sub>x</sub> emissions at full and mid loads

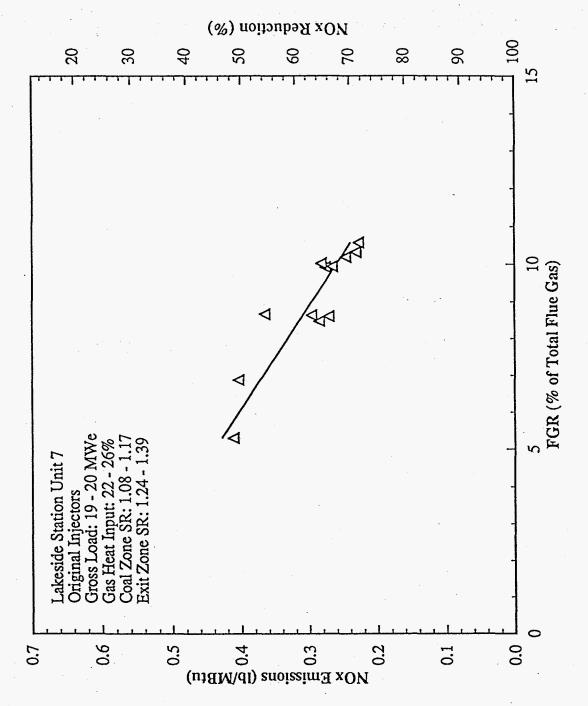


Figure 5-20. Effect of FGR on NO<sub>x</sub> emissions at low loads

evaluated in the range 1800 to 4600 scfm (0.85 to 2.17  $\rm Nm^3/s$ ). Reductions in  $\rm SO_2$  were calculated from the 5.9 lb/10<sup>6</sup>Btu (2540  $\rm mg/MJ$ ) baseline.

### 5.2.2.1 Ca/S Molar Ratio

The SO<sub>2</sub> emissions/reductions under SI operation, for the range of load 19 to 34 MW<sub>e</sub>, are shown in Figure 5-21. While succeeding figures differentiate data at different loads, on average the SO<sub>2</sub> reductions were 25% at a Ca/S of 1.1 and 42% at a Ca/S of 2.1. Figure 5-22 shows the SO<sub>2</sub> levels under GR-SI operation with gas heat inputs of 22 to 25%. On average, the SO<sub>2</sub> reductions were 51% at a Ca/S of 1.1 and 61% at a Ca/S of 2.1. In the majority of GR-SI cases with 22 to 25% gas heat input the design level of 50% SO<sub>2</sub> reduction, corresponding to 2.95 lb/10<sup>6</sup>Btu (1270 mg/MJ), was achieved. The maximum SO<sub>2</sub> reduction measured under GR-SI was 68% at a Ca/S of 2.09 and gas heat input of 23%.

The injection temperature and indirectly the operating load had strong impacts on SO<sub>2</sub> reduction and calcium utilization. While SI was evaluated with parametric tests over a wide range of Ca/S molar ratios, full load GR-SI operation was generally conducted with Ca/S in the 1.5 to 2.0 range. The data show that sorbent SO<sub>2</sub> capture was 4 to 6% higher when GR was applied with SI. GR resulted in an upward shift in the gas temperature, at the sorbent injection planes, to provide for a more suitable temperature for SO<sub>2</sub> capture. The corresponding Ca utilizations are shown in Figure 5-23. SI, with hydrated lime, generally results in a Ca utilization in the 20 to 30% range. That was the case in this application. On average, a 1.5 to 2.5% increase in Ca utilization resulted from GR-SI operation over levels for SI-only operation under full load. However, at reduced loads the impact of GR on the SO<sub>2</sub> capture process was more significant.

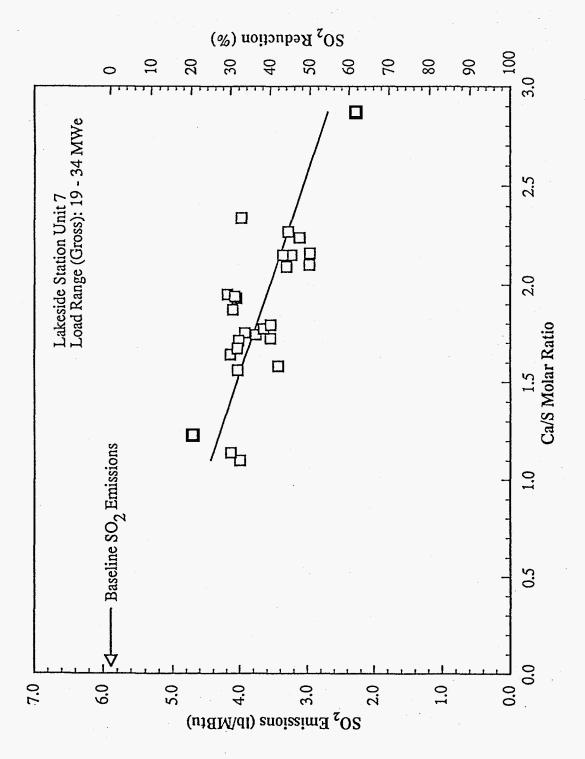


Figure 5-21. Effect of Ca/S ratio on SO<sub>2</sub> emissions with SI only

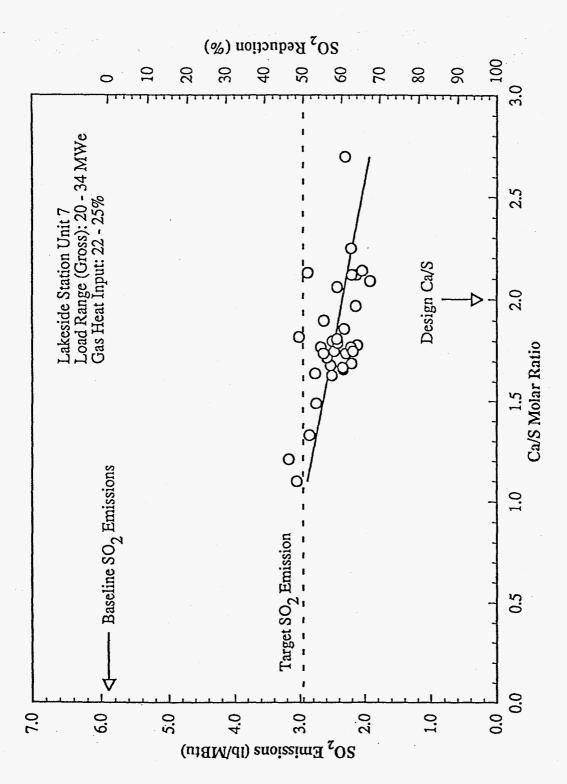


Figure 5-22. Effect of Ca/S ratio on SO<sub>2</sub> emissions with GR-SI

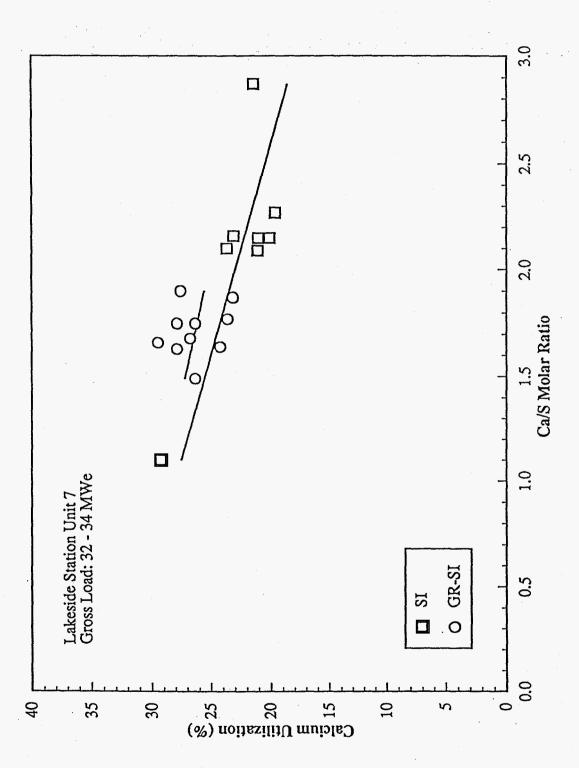


Figure 5-23. Effect of Ca/S ratio on calcium utilization

### 5.2.2.2 Sorbent Injection Air Flow

SI air was used to increase the mass flux of sorbent jets and thereby enhance mixing. Since the process is temperature dependent, rapid mixing with the flue gases at the exit of the furnace must take place, otherwise a loss in process efficiency results. Figure 5-24 shows SO<sub>2</sub> emissions under GR-SI and SI-only operation at full load, for a range of SI air flows. Modest reductions in SO<sub>2</sub> emissions were measured at high SI air flows. Under GR-SI, the SI air flow of 3700 scfm (1.75 Nm³/s) was commonly used and optimum results were achieved with 4600 scfm (2.17 Nm³/s).

### 5.2.3 GR-SI Results

The  $NO_x$  and  $SO_2$  reductions measured with GR-SI during long term testing are shown in Figure 5-25a and 5-25b. The target reductions of 60% for  $NO_x$  and 50% for  $SO_2$  are also shown. Generally, gas heat inputs of 22 to 24% were used, which approximate the design level of 24%, and the Ca/S ratio was in the range of 1.5 to 1.9 at full load and 1.9 to 2.1 at reduced load. On average, the Ca/S ratio during long-term testing was below the design level of 2.0. Over the long-term testing period,  $NO_x$  reduction averaged 63% and  $SO_2$  reduction averaged 58%.

# 5.3 GR-SI Technology Conclusions

Based on the nitrogen oxide and sulfur dioxide reductions achieved on the tangentially-fired unit and cyclone-fired unit demonstrations, it is seen that the GR-SI technology is equally applicable to these two types of utility boilers. In both demonstrations, the NOx and SO<sub>2</sub> reduction goals were achieved within the predicted operating parameters to achieve such reductions.

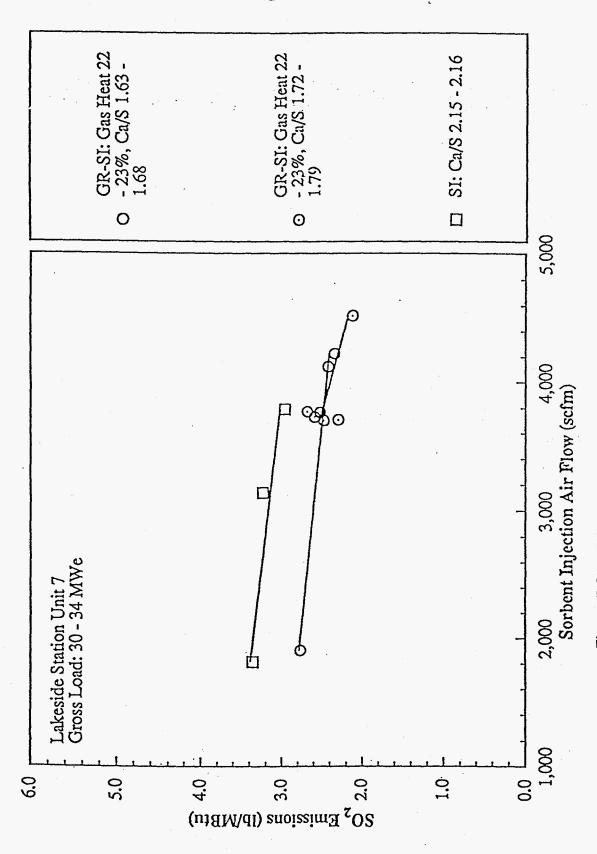


Figure 5-24. Effect of SI injection air rate on SO<sub>2</sub> emissions

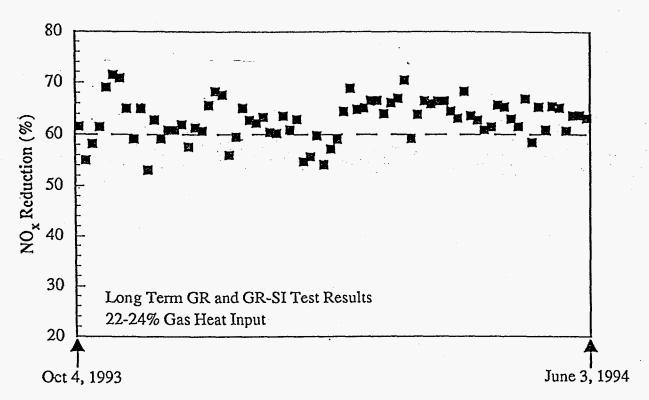


Figure 5-25a. Long term testing  $NO_x$  emissions reduction

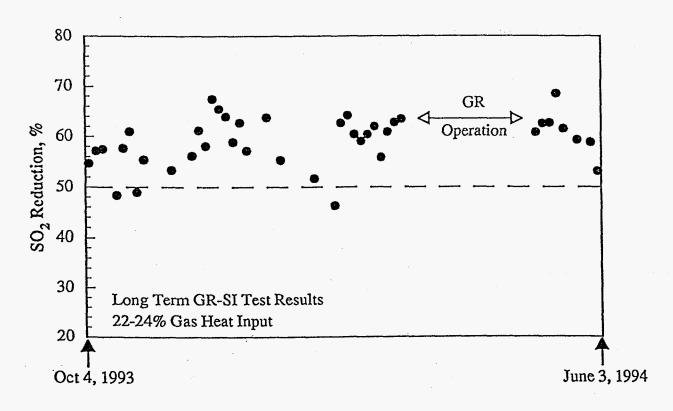


Figure 5-25b. Long term testing  $SO_2$  emissions reduction

### 6.0 BOILER IMPACTS

# 6.1 Operational Impacts

### 6.1.1 Thermal Performance

In this section the impacts of GR and SI operation on boiler thermal performance are discussed. In steam generating units, the heat released from combustion of fuels must be absorbed by heat exchangers with high efficiency. These include the furnace waterwall, the secondary superheater, primary superheater, generating bank, and air heater. Elevated steam temperature and pressure are preferred since they result in the lowest heat rate. The unit must operate with minimum deposition of ash on the furnace waterwall and convective heat exchangers.

GR operation can affect the thermal performance of the unit in two ways. First, GR affects the furnace heat release profile and second, GR operation changes local stoichiometric ratios and particulate loading resulting in minor changes in lower and upper furnace deposition patterns. Demonstrations of the technology showed that the overall impact of GR operation on the heat absorption profile was very minor. At full load a small decrease in furnace water wall heat absorption and small increases in heat absorption in the convective heat exchangers were calculated. At lower loads, data showed an increase in furnace heat absorption and small reductions by convective heat absorption. There will be a reduction in thermal efficiency of approximately 1% due to dry gas heat loss, moisture in fuel heat loss, and heat loss due to moisture from combustion. The decrease in heat absorption and the resulting rise in the flue gas temperature increase the dry gas heat loss. Fuel switching, i.e. replacement of coal energy with energy from natural gas, results in a reduction in the fuel moisture heat loss. Since natural gas has a higher hydrogen-to-carbon ratio than coal, its combustion results in formation of more moisture. Consequently there is a higher moisture loss.

During SI operation, there is an increase in heat absorbed by the furnace and reductions in convective heat absorption due to sorbent fouling, especially in the generating bank. Sootblowers must be used almost continuously in order to remove the deposits and maintain steam temperatures near the design point and limit upward excursions of flue gas temperature. The increased particulate loading caused a shift in heat absorption from the furnace and upper convective pass to the convective backpass. This shift has the following impacts:

- Reduction in thermal efficiency (approximately 1.2%)
- Increase in temperature and velocity of the flue gas
- Higher fly ash resistivity, which has been found to vary strongly with temperature, thus potentially reducing the collection efficiency of the ESP

In tangentially-fired boilers, three systems are used to control superheat and reheat steam temperatures as load, excess air, boiler fouling, and other parameters vary. These include:

- <u>Burner Tilt</u> Burner tilt adjusts the position of flames in the furnace. When the
  burners are tilted up, furnace waterwall heat absorption is reduced shifting
  heat to the superheater and reheater sections to increase the main and
  reheat steam temperatures. Under normal operation, the burner tilt is
  controlled automatically to achieve the reheat temperature set point.
- Reheat Attemperation Feedwater is sprayed into the reheat steam for further reheat temperature control. Since the attemperation water bypasses the high pressure turbine stages, it increases heat rate (reduces thermal efficiency). Accordingly, the reheat attemperation is normally used as a backup system.
- <u>Main Steam Attemperation</u> Feedwater is also sprayed into the superheat steam to control its temperature. This provides a means to adjust superheat steam temperature independent of reheat steam temperature, which is controlled primarily by burner tilt.

Fouling of the reheater, caused by SI operation, requires higher burner tilts to maintain reheat temperature. Also, additional superheat steam attemperation is required to control the main steam temperature. Only very low levels of reheat attemperation are required.

### 6.1.2 Slagging

Specific areas of the boiler have the potential for increased slagging under GR-SI operation due to the following:

- The slightly lower primary zone stoichiometric ratios and resulting higher gas temperature in that zone could exacerbate slagging in the lower furnace
- The reducing conditions in the reburning zone could promote slagging in this zone.

Demonstrations of the technology have shown that GR-SI operation does not exacerbate slagging in the furnace. Some buildup of slag in the lower furnace, from the top burner elevation to the OFA ports, can occur, but in not excessive levels as compared to that under baseline operation. Most of the deposits below the nose elevation were small deposits of fine ash particles which most likely consisted of particles with a very low melting point. The small molten particles impinge on the tubes and tend to resolidify as they come in contact with the relatively cooler wall. Little change in the upper furnace slagging was observed.

Since ash fusion temperatures are lower under reducing conditions, areas in the furnace that are deficient in excess air may have increased slag buildup. Sootblowing more frequently or with greater effectiveness (i.e. higher pressure) may be required to reduce slag buildup.

Injection of natural gas and FGR promote formation of slag patterns, i.e. slag accumulations around the nozzles forming "eyebrows" and on the waterwall areas above the natural gas/FGR injectors. Ash accumulation at the port entrances may be attributed to the tempering caused by the injection of gas and FGR and the fuel-rich conditions in the reburning zone. Periodic manual cleaning (nominally weekly) is sufficient to maintain normal GR operation. Ports should be thoroughly cleaned after periods when the GR system is out of service.

## 6.1.3 Convective Pass Fouling

As stated previously in this report, GR results in an increase in flue gas temperature in the upper furnace due to heat input above the primary zone. Steam temperatures, however, are controlled through steam attemperation. SI increases the particulate loading which tends to foul the tubes. However, this is offset by increased frequency of sootblowing. In tangentially-fired boilers, with the burner tilt capability, a cyclical pattern results. At full load, coal burners are initially tilted downward. As tubes begin to foul when SI is introduced, the burners tilt upward to compensate for the reduction in heat absorption. Sootblower operation rectifies the situation and the burners return to the downward position.

Demonstrations of the technology have shown that during GR-SI the heat absorption ratio (HAR) and cleanliness factors can be maintained at near baseline conditions thru frequent use of sootblowing. The HAR for the secondary superheater varied most widely but was maintained. Very little change was noted in the reheater. However, higher flue gas temperatures were observed in the primary superheater due to increased levels of fouling. Steam temperatures were maintained during GR-only operation by using steam attemperation.

# 6.1.4 <u>Auxiliary Power</u>

The major contributors to increases in auxiliary power requirements are the following:

- FGR fan
- SI air fan
- Injector cooling air fans
- Sorbent transport equipment
- Electrostatic precipitator (handling increased particulate)

For boilers less than 100 MWe, the increased power requirement will be 700-1000 kW.

# 6.1.5 <u>Cooling Requirements</u>

Retrofit of a GR-SI system includes installation of several furnace wall openings. Components in these openings that are not water cooled require protection from the hot furnace gases. These include openings for natural gas, OFA, and sorbent. During GR-SI operation these components are cooled by the input gases. Natural gas cools the reburning nozzles, OFA cools the OFA nozzles, and sorbent injection air cools the SI nozzles.

During non GR-SI operation a protective cooling air blanket is directed to and through the GR-SI furnace wall openings to provide the required protection from the hot furnace gases. The protective blanket for the overfire ports uses a small amount of hot secondary air that passes through the OFA control dampers. The dampers have a minimum open position which assures enough air passage to give a velocity of 50 ft/sec or greater through the ports. Cooling fans are used to cool the natural gas and SI injectors.

### 6.2 <u>Durability</u>

A high number of equipment items in the plant are subject to mechanical wastage from using coal as the primary fuel, particularly the process and delivery systems. These include feeders, mills, primary coal/air transport lines, coal nozzles in burners, convective pass, electrostatic precipitator, stack, and bottom ash systems. Surface wastage (erosion) exists on boiler parts that are exposed to particulate from combusted coal and sootblowing. During GR, approximately 10-20% of the coal fuel is replaced by natural gas. The use of gas allows for a corresponding reduction in coal usage, reducing the mechanical wastage on equipment. Also, since natural gas has no particulate or ash residue, the decreased use of sootblowers and lower level of particulate provides a proportional benefit to tube life expectancy.

### 6.2.1 Boiler Tubes

During reburning, a fuel-rich condition is established in the reburning zone. It is well known that fuel-rich conditions can enhance tube wastage due to two mechanisms:

- When fuels containing sulfur are burned under oxygen deficient conditions, some of the sulfur forms reduced sulfur species such as COS and H<sub>2</sub>S. These species react with iron in the tubes via Fe + H<sub>2</sub>S = FeS. The FeS scales off the tube leading to wastage (corrosion).
- Under normal excess air operation, the tubes are protected by a thin oxide layer. Reducing conditions, particularly fluctuating (oxidizing/reducing) conditions, can continuously degrade this protective layer.

Normal rates of tube wastage in coal-fired boilers are in the range of 0.001" to 0.003" per year, while some boilers inherently produce massive tube wastage. During the two field demonstrations described above, a one-year load-following long term test program was conducted. Data obtained following the test period provided an initial indication of the potential for significant tube wastage.

During field demonstrations performed by EER, the boiler tubes were subjected to both destructive and non-destructive testing to determine if GR operation jeopardizes the life of the tubes. Specific areas were targeted where the mechanisms listed above suggest the potential for significant tube wastage. The goal of the testing was first and foremost to determine if there was a significant increase in tubewall wastage from the planned one year of operation. This would be readily apparent using the testing methods prescribed. A secondary goal was to determine the incremental change in tube thickness and project this change to the end of the boiler useful life. Since the nominal tube wastage rates are small, a statistical approach was warranted. It should be noted that extrapolating small tube wastage differences over a large time period can be considered questionable. However, significant levels of tube wastage over a short period can be used to provide long-term predictions.

Boiler tubes must be of suitable metallurgical quality to withstand the severe conditions encountered in the boiler environment. Thus it was necessary following GR to test the boiler tubes and determine if any changes to the metallurgy had occurred. Tube samples, physically extracted from the boiler both prior to and after GR operation, were tested for microstructures, Rockwell B Hardness and sulfur print. A comparison was then made between the results of the two sets of tube samples to determine if any metallurgical changes occurred, such as overheating, thermal-fatigue cracks, grain-boundary oxidation, corrosion, and pitting.

Following extensive statistical evaluation, it was concluded that:

- No significant tube wastage was found.
- Levels of non-GR tube wastage measured did not compromise the projected life of the boiler.
- No metallurgical degradation was found.

# -6.2.2 Regenerative Air Heater

During demonstrations of the technology, no adverse conditions in the air heater were noted as a direct result of GR-SI operation. However, SI requires an increase in sootblowing which may have a degrading effect on the air heater from the excessive dwell time of the sootblower lance at its innermost position. It is recommended that wear plates be installed at the inboard extent of the sootblower travel to protect the basket elements.

# 6.2.3 <u>Electrostatic Precipitator</u>

The level of particulate produced in the boiler decreases proportionally to the amount of natural gas used during reburning. However, a marked increase occurs during SI. For this reason it was necessary to perform an evaluation of the ESP prior to installation of SI to verify that it had sufficient capacity to handle the increase.

If particulate emissions are found to be excessive, control can be achieved by incorporating a humidification system. This equipment will enhance the effectiveness of the ESP thereby limiting particulate emissions to near baseline levels. By cooling the flue gas and adding moisture, the resistivity of the particles is decreased making the particles more receptive to capture in the ESP. It is necessary to control fouling of the tubes using the sootblowing system in order to maintain a flue gas temperature that is compatible with the humidification system. Fouling will cause the flue gas temperature to rise and reduce the effectiveness of the system.

During demonstrations of the GR-SI technology, no physical damage was found to occur in the precipitators. However, it is recommended that the utility maintain a preventive maintenance program. In addition, the rapping sequence should be optimized to avoid ash buildup.

### 7.0 ECONOMICS

The capital and operating costs for the GR system for  $NO_x$  emissions reduction and the SI system for  $SO_2$  emissions reduction are based on a retrofit of a 300 MWe coal cyclone-fired power plant. The reason for selecting a cyclone-fired unit is that there are not many low cost NOx control options for these type of units. The degree of complexity regarding retrofit costs were factored based on the retrofit costs for the GR-SI demonstrations completed under this DOE contract. Separate capital and operating costs are presented for the GR and the SI systems. These two systems were treated as separate technologies; the only major synergistic effect of the GR system on the SI system is the reduction of  $SO_2$  based on the replacement of sulfur-containing coal with natural gas devoid of sulfur.

### 7.1 Gas Reburning System

#### 7.1.1 GR - Economic Parameters

The capital cost estimates presented summarize major equipment cost, approximate bulk material take-offs, and installation labor to arrive at direct construction costs. Construction indirects are added which include: field supervision, construction overhead and fee, and freight. In addition, costs for detailed engineering, project management, procurement, construction management, start-up, and contingency are included to develop the total installed system cost. All engineering and construction costs are representative of a turn-key contract arrangement. EER considers these estimates to be Class II, Preliminary Estimates. The estimates are expected to be representative of the actual cost -10%/+15%. This is based on the information available at this time which includes preliminary process design and conceptual engineering completed, recent major equipment quotes, bulk material takeoff's and average expected labor rates and productivity.

This section provides the basis for the estimating procedures, along with a list of assumptions used for estimating installation manhours and costs. The cost estimates have been developed using the following sources of information for equipment pricing and for the development of labor costs:

- Richardson's Rapid System 1993 edition of <u>Process Plant Construction</u>
   <u>Estimating Standards</u>
- Questimate Cost Estimating software by Icarus Corp.
- Means Electrical Cost Data 1991 edition
- Vendor Quotations for Major Equipment
- EER's database of previous equipment purchases

Data from all of these sources were summarized using EER cost estimating software. Once the direct costs were determined, costs for field supervision, contractor overhead and fee, freight, engineering, project management, construction management, start-up, and contingency were added to determine the total installed cost.

### 7.1.2 GR - Estimated Process Capital Cost

The design of the GR system included three integrated systems: 1) natural gas injection, 2) FGR, and 3) OFA injection. Natural gas is mixed with FGR at the gas injection nozzles located above the cyclone barrel re-entrant throats. A natural gas header was assumed to exist at the station and a tie-in was made to this supply header to provide the natural gas for the GR system. The tie-in pipe supplied gas to a control and metering station and from this station natural gas was distributed to gas injection nozzles located above the cyclone barrel re-entrant throats. The natural gas valve train, common to all of the injection nozzles, included flow metering and control equipment, and safety shut-off valves.

An FGR system was also included, hot flue gas was drawn from the economizer flue gas outlet with an FGR fan, routing the flue gas to the natural gas injectors. The purpose of using FGR with natural gas injection is to increase the dispersion and mixing of natural gas throughout the furnace Reburn zone. The use of FGR can increase the NOx reduction efficiency of natural gas reburning systems on large furnace boxes and on units with low reburn zone residence times. The FGR system included a multiclone for particulate removal upstream of the FGR fan.

OFA was assumed supplied from the existing hot secondary combustion air windbox. The existing windbox pressure on a cyclone-fired unit is adequate, so booster fans were not required. The installation of natural gas/flue gas injectors and OFA ports requires furnace tubewall modifications. The high windbox pressure of cyclone-fired units, 30" to 75" WC depending on whether the unit is a forced draft or balance draft unit, precludes the necessity of adding booster air blowers for the OFA system. The total cost of equipment and materials for the GR system was estimated at \$1,130,069. The following is a list of equipment/material and costs by area, that make up the total equipment and materials cost for the system.

Equipment/Materials Description

Equipment/Material Cost

\$125,488

Natural Gas Injection Unit:

**NG Supply Piping** 

NG Valve Train Piping

NG Header Piping

NG Injector Piping

NG Nozzle Flex Hose

**NG** Injection Assembly

NG Vent Piping

NG Instrumentation, Valves & Dampers

**NG Tubewall Modifications** 

NG Injector Cooling Air Piping

7-3

-FGR Unit:

\$548,360

FGR Fan

Hot FGR Duct to Inlet of Fan

FGR Duct from Fan to Boiler

FGR Injector Duct Header

OFA Unit:

\$374,621

**OFA Main Ductwork** 

**OFA Main Duct Expansion Joints** 

**OFA Branch Ductwork** 

**OFA Branch Duct Expansion Joints** 

**OFA Nozzle Ducts** 

**OFA Nozzle Supports** 

**OFA Injection Nozzles** 

**OFA Instrumentation & Dampers** 

**OFA Tubewall Modifications** 

Other:

\$81,600

**Control Modifications** 

**Electrical Work** 

**New Structural Steel** 

The estimated total capital requirement to retrofit a GR system to an existing 300 MWe cyclone-fired unit is \$5,060,000 or a cost of \$16.85/kWe. The breakdown of costs is presented in Table 7-1.

# 7.1.3 GR - Projected Operating and Maintenance Costs

EER conducted analyses to evaluate the fixed and variable (operating) costs of a GR system for a 300 MWe coal cyclone-fired power plant (net heat rate of 10,000 Btu/kWhr before GR); contributing cost factors were as follows:

# TABLE 7-1. 300 MWe CYCLONE-FIRED UNIT GR RETROFIT CAPITAL COST

# **Capital Cost**

Category	\$10 <sup>6</sup>	\$/kWe
Equipment	1.13	3.77
Construction Labor	0.92	3.07
Construction Indirects	0.78	2.60
Other (6%), Freight (2%) & Taxes (5%)	0.15	0.49
Gas Supply <sup>[1]</sup>	0.00	0.00
Gas Metering & Reduction Station	0.45	1.50
Total Process Capital	3.43	11.43
Engineering (10% of process capital)	0.34	1.14
Project Management (8%) /Owners Costs (5%)	0.45	1.49
Project Contingency @ 15%	0.63	2.11
Total Plant Cost	4.85	16.17
Allowance for Funds During Construction [2]	0.00	0.00
Total Plant Investment (TPI)	4.85	16.17
Royalty Fees @ 0.5% of Total Process Capital	0.02	0.06
Startup Costs @ 3% TPI	0.15	0.49
Working Capital @ 0.9% TPI	0.04	0.15
Cost of Construction Downtime (28 days) [3]	0.00	0.00
Total Capital Requirement	5.06	16.85

<sup>[1]</sup> Gas supply availability at site assumed adequate

<sup>[2]</sup> No allowance included based on DOE guideline

<sup>[3]</sup> Assumed downtime to be during scheduled major outage

- 1. Reburning Fuel Cost Differential Since gas costs more than coal on a heating value basis (\$/10<sup>6</sup> Btu), there is a cost related to the amount of gas fired. This was calculated based on the delivered costs of gas and coal and the percentage of gas fired (20.1% of the total heat input). A value of \$1.00/10<sup>6</sup> Btu was used as the differential between the delivered price of natural gas (\$2.47/10<sup>6</sup> Btu) and the delivered price of coal (\$1.47/10<sup>6</sup> Btu).
- 2. <u>Changes in Boiler Efficiency</u> Since the boiler efficiency is lower when using gas as the reburning fuel there needs to be an increase in the amount of fuel fired to make up for the lower efficiency. This increase was based upon a boiler efficiency loss (1.27%) with GR and a composite fuel cost of \$1.67/10<sup>6</sup> Btu.
- Reduced Load on Coal Crushers Since the GR fuel contributes a significant portion of the boiler fuel, there is a corresponding percentage decreased load on the coal crushers. The electricity credit was based on an auxiliary power cost of \$0.02/kWhr.
- 4. <u>Maintenance Items/Spare Parts</u> An allowance of 2% of the total plant investment was used for total maintenance, 40% of this 2% was allocated for maintenance items and spare parts.
- 5. <u>Maintenance Labor</u> An allowance of 2% of the total plant investment was used for total maintenance, 60% of this 2% was allocated for maintenance labor.
- 6. <u>Administration and General Overhead</u> An allowance of 60% of plant labor was added to cover administration and general overhead.
- 7. <u>Local Property Taxes and Insurance</u> An allowance of 3% of total plant investment was used to cover taxes and insurance.
- 8.  $\underline{NO_x}$  Reduction A  $NO_x$  reduction of 67% was assumed at 20.1% of the total heat input as reburn fuel. This is based on a 300 MWe unit having a higher initial  $NO_x$  concentration and longer reburn residence zone time, and therefore a higher  $NO_x$  reduction with less gas than the CWLP unit.

The total annual incremental gross operating cost for the GR system, exclusive of any payback of capital, is estimated at \$4,177,496. If an SO<sub>2</sub> allowance credit is taken based on the reduction of fuel sulfur when firing natural gas, the net operating cost is estimated at \$3,422,703. This SO<sub>2</sub> credit was based on an allowance of \$95/ton (Feb. 1996). The operating cost breakdown for the GR system retrofit to a 300 MWe cyclone-fired unit is shown in Table 7-2.

# TABLE 7-2. 300 MWe CYCLONE-FIRED UNIT, GR OPERATING COST

# Annual Incremental Operating Costs<sup>[1]</sup>

	Annual Use	Cost/Unit	Cost/ Yr
Variable Costs			
Fuel:			
Natural Gas	3,436,898 10 <sup>6</sup> Btu	\$1.00 /10 <sup>6</sup> Btu <sup>[2]</sup>	\$3,436,898
Supplemental Fuel	243,645 10 <sup>6</sup> Btu	\$1.67 /10 <sup>6</sup> Btu <sup>[3]</sup>	\$407,131
Utilities:			
Electricity	3,794 10 <sup>3</sup> kWhr	\$20.00 /10 <sup>3</sup> kWhr	\$75,888
Ash Disposal Credit	(3,389) tons	\$9.29 /ton <sup>[4]</sup>	(\$31,484)
Sub-Total			\$3,888,432
Fixed Costs			
Labor: [5]			
Maintenance (2% of TP	•		\$58,201
Supervision (20% of Ma	aintenance Labor)		\$11,640
Supplies:			
Maintenance (2% of TP	PI x 40%)		\$38,801
Admin. and Gen. Ovhd. (6	60% of total labor)		\$34,920
Local Taxes and Insurance	e @ 3% of TPI		\$145,502
Sub-Total			\$289,064
Total Gross Operating C	cost		\$4,177,496
SO2 Allowance @ \$95/to	n <sup>[6]</sup>		(\$754,793)
Total Net Operating Cos	it .		\$3,422,703

- [1] 65% Capacity factor @ 300 MWe net capacity (10,000 Btu/kWhr heat rate) w/ 20.1% fuel heat input as natural gas
- [2] Natural gas assumed delivered at \$2.47/MM Btu; coal cost at \$1.47/MM Btu
- [3] Extra fuel added to make up for loss in efficiency (1.27%) at same coal/gas ratio as reburn
- [4] Credit for less fly ash, based on 25% carryover, assuming cyclone bottom slag can be disposed of at no cost
- [5] Assumed no added operating labor required to operate the GR system
- [6] February 1996 Allowance Credit Value, reduction based on 4.8 lb SO2/MM Btu for coal w/coal reduction of 19.84%

# \_7.1.4 GR - Summary of Performance and Economics

Based on the developed capital and fixed/variable operating costs, economic projections were made using current dollars which include an inflation rate of 4.0%, and constant dollars which ignore inflation. The factors used in the development of the technology economics are shown in Table 7-3. Table 7-4 shows the performance and cost for a 300 MWe GR System that is retro-fitted to a cyclone-fired boiler. The table reflects the NOx reduction costs based on a 65% capacity factor with 20.1% of the heat input supplied by natural gas at a gas to coal price differential of \$1.00/million Btu. The incremental increase in the levelized cost of power, including capital charges is estimated at 2.81 mills/kWhr in constant dollars and 3.69 mills/kWhr in current dollars.

If an  $SO_2$  credit is applied based on fuel sulfur reduction when firing natural gas, the net incremental increase in the levelized cost of power is estimated at 2.37 mills/kWhr in constant dollars and 3.11 mills/kWhr in current dollars. The levelized cost of  $NO_x$  removal is estimated at \$646/ton and \$847/ton for current and constant dollar projections, respectively. If an  $SO_2$  credit is applied based on fuel sulfur reduction, the net levelized cost of  $NO_x$  removal is estimated at \$544/ton and \$713/ton for current and constant dollar projections, respectively.

Based on the levelized cost (in constant dollars) for reducing nitrogen oxides, excluding  $SO_2$  credits, the capital charge component made up around 13% of the total cost of  $NO_x$  reduction. The fixed operation and maintenance costs represented only 6%, and the variable cost made up the rest of the cost for removing  $NO_x$ . The cost of  $NO_x$  removal shows that the variable operating cost is the greatest cost component, making up some 81% of the NOx reduction. Further, the most significant component of the variable operating cost is the cost of natural gas.

TABLE 7-3. ECONOMIC FACTORS

Item	Units	Value
Cost of debt	%	8.5
Inflation rate	%	4.0
Construction period	mos.	9
Remaining life of power plant	-	15
Year for cost presented in this report	-	1996
Royalty allowance based on total process capital	%	0.5
Capital charge factor - current dollars	-	0.160
Capital charge factor - constant dollars	-	0.124
O&M cost levelization factor - current dollars	-	1.314
O&M cost levelization factor - constant dollars	<del>-</del>	1.000
Power plant size	MWe (net)	300
Power plant type	cyclone	
Power plant capacity factor	%	65
Sales tax rate	%	5.0
Cost of freight	%	2.0
Engineering/home office fees of total process capital	%	18.0

# TABLE 7-4. GR RETROFIT COST AND PERFORMANCE SUMMARY

### Summary of Data

### **Power Plant Attributes**

	Units	Value
Plant capacity, net	MWe	300
Power produced, net	10 <sup>9</sup> kWhr/yr	1.71
Capacity factor	%	65
Plant life	yr	15
Coal feed	tons/yr	683,280
Sulfur in Coal	wt%	3.0

### **Emissions Control Data**

	Units	NOx
Removal efficiency	%	67.0
Emissions standard	lb/10 <sup>6</sup> Btu	0.86
Emissions without controls	lb/10 <sup>6</sup> Btu	1.30
Emissions with controls	lb/10 <sup>6</sup> Btu	0.43
Amount reduced	tons/yr	7,439

#### **Levelized Cost of Power**

	Current Dollars		<b>Constant Dollars</b>	
	Factor	Mills/kWhr	Factor	Mills/kWhr
Capital Charge	0.160	0.47	0.124	0.37
Fixed O&M	1.314	0.22	1.000	0.17
Variable Operating Cost	1.314	2.99	1.000	2.28
Total Cost		3.69		2.81
SO <sub>2</sub> Credits	1.314	(0.58)	1.000	(0.44)
Total Cost w/SO <sub>2</sub> Credits		3.11		2.37

### Levelized Cost - NOx Removal Basis

	\$/ton			\$/ton	
	Factor	removed	Factor	removed	
Capital Charge	0.160	109	0.124	84	
Fixed O&M	1.314	51	1.000	39	
Variable Operating Cost	1.314	687	1.000	523	
Total Cost		847		646	
SO <sub>2</sub> Credits	1.314	(133)	1.000	(101)	
Total Cost w/SO 2 Credits		713		544	

Basis: 67% NOx reduction assumed based on larger unit with longer Reburn zone residence time than CWLP 33 MWe cyclone unit (58% NOx reduction w/20% gas heat input as Reburn fuel).

# \_7.1.5 GR - Effect of Variables on Economics

The cost of  $NO_x$  reduction was analyzed and certain variables were then selected to perform sensitivity analyses. The variables chosen were natural gas price, the capacity factor and the unit size, the effects of these variables are shown Table 7-5.

TABLE 7-5. COST OF NO<sub>x</sub> REMOVAL

Natural Gas Price, \$/10 <sup>6</sup> Btu*	\$0.50	\$1.00	\$1.50			
NOx Reduction Cost, \$/ton**	\$310	\$544	\$779			
Capacity Factor	50%	55%	60%	65%	70%	75%
NOx Reduction Cost, \$/ton**	\$584	\$569	\$555	\$544	\$535	\$527
Unit Size, MWe	100	300	450	600	750	900
NOx Reduction Cost, \$/ton**	\$652	\$544	\$521	\$507	\$497	\$490

#### Note: Base case variables in bold

For the sensitivity analysis, only the variable being analyzed was changed from the base case, all other variables were kept the same. The cost of natural gas is clearly the driving force for the economics of the GR System. A 0.50 swing in the price of natural gas has the effect of changing the cost of  $NO_x$  reduction by over 0.50 over 0.50 swing in capacity factor has a relatively small effect on the cost of 0.50 removal. A 0.50 swing in capacity factor results in a 0.50 removal. A 0.50 swing in capacity factor results in a 0.50 removal.

To examine the power plant size effect on NOx reduction costs, the capital cost of the 300 MWe unit was used as a base. A scale up factor of 0.75 was used to extrapolate the capital cost for smaller and larger units. This factor was based on a combination of 50% of the equipment being increased in size using a 0.6 scaleup factor and 50% of the equipment being duplicated using a 0.9 scale up factor. The size effect for units larger than 300 MWe has only a slight effect on the cost of  $NO_x$  reduction. For a 900 MWe unit

<sup>\*</sup> Differential price compared to coal

<sup>\*\*</sup> Constant dollar basis, includes \$95/ton SO<sub>2</sub> credit

the cost of NO<sub>x</sub> reduction is only\_\$54/ton less than that for the 300 MWe unit. With smaller units the effect is more dramatic, for a 100 MWe unit the cost of NOx reduction would be \$108/ton more than that for the 300 MWe unit.

### 7.1.6 GR - Comparison with other NO, Control Technologies

Methods for controlling  $NO_x$  from coal-fired utility boilers include combustion modifications and post combustion treatment of the flue gas. Combustion modifications involve operating the primary combustion zone under fuel rich conditions (and therefore reduced temperatures), cooling the flame at a higher rate, and dilution of the flame to reduce adiabatic flame temperatures. Gas residence times in the high temperature zone as well as excess air levels are reduced, inhibiting the formation of fuel and thermal  $NO_x$ .

The combustion modification techniques that can be applied depends on the type of boiler and method of firing the fuel. Low  $NO_x$  burner technology with OFA has been successfully applied to wall and tangentially fired pulverized coal units. Low  $NO_x$  burner technology, however, cannot be applied to cyclone units due to the configuration of the cyclone furnaces. The importance of OFA as it relates to staging the combustion process has been determined in testing of low  $NO_x$  burner retrofits and demonstrations. This information has promoted the addition of OFA to conventional firing systems as a stand alone alternative to low  $NO_x$  burners for utilities requiring moderate reductions.

OFA systems may be "close-coupled" to the existing burner assemblies on tangentially fired units, or separated higher into the furnace on both tangentially and wall fired designs for deeper staging and increased NO<sub>x</sub> reductions. Staged combustion with OFA also cannot be applied to cyclone-fired units with high sulfur coal feedstocks. Industry experience indicates that this combustion modification technique for high sulfur feedstocks results in high levels of corrosion in the cyclone barrels.

Post combustion techniques include reburning, selective catalytic reduction (SCR), and selective non-catalytic reduction (SNCR). The reburning process using natural gas as the reburning fuel is described elsewhere in this manual. Coal and coal water slurry (CWS) have also been proposed as reburning fuels. In Table 7-6 below, a relative comparison is made between the cost, design, and operating factors associated with the three reburning fuels.

Table 7-6. Reburning Fuel Comparison

	Natural Gas	Coal	cws
Reburning Fuel Cost	Highest	None	Low
Capital Cost	Lowest	Highest	Low
SO <sub>2</sub> Reduction	Yes	None	Varies
Injector Size	Small	Large	Small
Auxiliary Power	Low	High	Moderate
Residence Time Requirement	Low	Moderate	Moderate

Natural gas is the most expensive reburning fuel, with the differential above coal averaging \$1.00 to \$1.50/106 Btu. Coal reburning involves no differential fuel cost since the total heat input to the unit does not change. The cost of CWS is site specific depending on the cost and the availability of the coal fines used to formulate the slurry. CWS may be produced by wet milling the primary coal (~\$4/ton), using the minus 100 mesh froth cell product from coal cleaning plants, or recovering coal fines from coal preparation plant ponds with advanced coal cleaning techniques (delivered cost could be less than primary coal cost or higher depending on ownership of resource, quality of impounded coal and distance from power plant). Other fuels, like fuel oil and Orimulsion®, a Venezuelan bitumen-water emulsion can also be used as effective reburning fuels.

If gas is available at the power plant, GR offers the lowest capital cost investment since there are no fuel preparation or handling equipment requirements. Coal reburning will require the addition of coal handling and milling equipment, milling equipment upgrading, or storage and handling equipment for coal fines produced elsewhere. Reburning with slurry requires CWS feeding equipment, added air compression for CWS atomization, and either onsite CWS storage or CWS formulation equipment for delivered coal fines filter cake.

Since natural gas contains no sulfur, GR offers an additional SO<sub>2</sub> reduction over that provided by SI or other processes since gas replaces coal containing sulfur. For normal GR applications gas would replace coal and SO<sub>2</sub> would be reduced some 20%.

Auxiliary power requirements for GR are relatively lower since fuel handling and preparation equipment is not necessary as it is with reburning using coal or CWS as the reburn fuels. Demonstrations of GR with FGR have shown that, with most furnace designs and adequate natural gas pressures available, the FGR may not be necessary to promote adequate mixing of the natural gas with the furnace gases. In such a case, the FGR fan can be eliminated, further reducing the auxiliary power requirements.

Consideration of the furnace geometry and available residence time may be critical in the selection of the reburning fuel. Natural gas requires the shortest residence time for the reburning process since the fuel "particle" size is at the molecular level. Coal, having larger particle sizes will require longer residence times.

Selective catalytic and non-catalytic reduction are post combustion treatment methods. In the selective catalytic reduction process, ammonia vapor and preheated air are mixed and injected into the flue gas at the boiler exit. The optimum temperature window for this process is 550 to 750°F. Flue gas at this temperature is generally available upstream of the unit's air heater. A catalytic converter is installed in the duct work at this location. NO<sub>x</sub> is reduced by the process to diatomic nitrogen in the converter. SCR systems are better suited for installation downstream of a hot side precipitator since dust buildup and catalyst fouling are reduced. On systems installed upstream of a cold side precipitator, the catalyst

mesh size must be increased to reduce dust build-up and catalyst fouling. The larger mesh size dictates a larger converter to provide the necessary surface area.

Ammonia slip (un-reacted NH<sub>3</sub>) is a major operating consideration with SCR systems. As the catalyst is expended, ammonia slip increases. Ammonia passing through the converter forms ammonium sulfate in particulate form which may foul equipment downstream such as air heaters, draft fans, or precipitators. Sulfates may also form in the catalyst pores to deactivate the catalyst if the flue gas temperature drops below 500°F. Un-reacted ammonia may also be adsorbed by the fly ash and increase the leachability of metals in the ash, affecting the salability of the fly ash.

In the SNCR process, ammonia or urea based reagents are injected into the upper furnace at locations where flue gas temperatures range from 1600 to 2000°F. With this process the required high activation energy is provided by the temperature of the flue gas, and a converter with catalyst is not necessary.

An independent study completed for the U.S. EPA (Contract No. 68-D2-0168) "Investigation of Performance and Cost of  $NO_x$  Controls as Applied to Group 2 Boilers", compared the costs of competing  $NO_x$  control technologies. The costs for various  $NO_x$  reduction systems applied to cyclone-fired units were developed as part of this study. The values for GR system are slightly different than those developed by EER due to different cost assumptions. Although actual costs vary, the relative costs for the technologies evaluated should be valid. In Table 7-7, the cost of Gas Reburning, Coal Reburning, Selective Non-Catalytic Reduction (SNCR) and Selective Catalytic Reduction (SCR), based on \$/kWe and \$/ton of  $NO_x$  removed are shown for 400 MWe cyclone-fired units. As shown in the table, the cyclone-fired boiler  $NO_x$  control technologies show a cost per ton of  $NO_x$  removed that ranges from approximately \$490 to \$690. Based on this comparison Coal Reburning is the least expensive and SNCR the most expensive with GR and SCR being near equal when the price differential between natural gas and the primary coal is \$1.00 /106 Btu. The  $NO_x$  reduction for SCR assumed for this study was low, only 50%.

TABLE 7-7. 400 MWe CYCLONE-FIRED NO<sub>x</sub> CONTROL COMPARISON

Technology	NOx Reduced	Capital Cost \$/kWe	NO <sub>x</sub> Removed⁴ \$/ton
Gas Reburning <sup>1</sup>	60	15.2	590
Coal Reburning	50	52.7	490
SNCR <sup>2</sup>	35	7.3	690
SCR <sup>3</sup>	50	40.0	575

- (1) Natural Gas @ \$2.47/106 Btu and Coal @ \$1.47/106 Btu
- (2) 50% Urea solution @ \$0.50/gal
- (3) Anhydrous Ammonia @ \$162/ton & SCR catalyst replacement (3 yr life) @ \$350/ft3
- (4) Base levelized costs

However, SCR systems have achieved 85%+ reductions with increased catalyst volume. For NO<sub>x</sub> reduction beyond what is possible by a particular technology, it is possible to combine some technologies for deeper reduction. For an example, Advanced GR is currently being marketed involving the simultaneous application of GR and SNCR. Overall NO<sub>x</sub> reduction is expected to be in the range of 75 to 90 percent. GR has also been demonstrated with low NO<sub>x</sub> burners, achieving an overall NO<sub>x</sub> reduction of 75 percent.

### 7.3 SI System

The SI system was designed to store, meter, and convey micronized hydrated lime (sorbent) to the injection nozzles in the upper furnace of the 300 MWe cyclone-fired Unit. The SI system is comprised of the following major components: sorbent storage silo, weigh hopper, rotary valve feeder, screw pump, air transport blower, conveying line, sorbent splitter, SI air fan, and furnace injection nozzles.

### 7.3.1 SI - Economic Parameters

The economic parameters were the same as that used for GR (See Section 7.1.1 and Table 7-3 in Section 7.1.4).

# 7.2.2 SI - Estimated Process Capital Cost

The total cost of equipment and materials for the SI system was estimated at \$883,278. The following is a list of equipment/material and costs by area, that make up the total equipment and materials cost for the system.

Equipment/Materials Description

Equipment/Material Cost

Sorbent Storage Unit:

\$134,710

Sorbent silo Weigh hopper Silo vent filter

Sorbent Feeding Unit:

\$232,774

Rotary valves Sorbent screw pumps Sorbent transport blowers Dust control unit

Sorbent Transport:

\$361,686

Piping, hoses, valves, splitters Sorbent equipment air compressors Air dryer Valves and controls

Sorbent Injection Unit:

\$154,108

Coaxial jet sorbent injectors, C.S./S.S. SI air fan SI nozzle cooling fan Instruments/controls

The estimated total capital requirement to retrofit an SI system to an existing 300 MWe cyclone-fired unit is \$3,860,000 or a cost of \$12.86/kWe. A breakdown of the capital cost is presented in Table 7-8.

# TABLE 7-8. 300 MWe CYCLONE-FIRED UNIT SI RETROFIT CAPITAL COST

# **Capital Cost**

Category	\$10 <sup>6</sup>	\$/kWe
Equipment	0.88	2.94
Construction Labor	0.72	2.40
Construction Indirects	0.61	2.03
Other (6%), Freight (2%) & Taxes (5%)	0.11	0.38
Total Process Capital	2.33	7.75
Engineering (10% of process capital	0.23	0.78
Project Management (8%) /Owners Costs (5%)	0.30	1.01
Project Contingency @ 15%	0.43	1.43
Total Plant Cost	3.29	10.97
Allowance for Funds During Construction [1]	0.00	0.00
Total Plant Investment (TPI)	3.29	10.97
Royalty Fees @ 0.5% of Total Process Capital	0.01	0.04
Startup Costs @ 3% TPI	0.10	0.33
Working Capital @ 0.9% TPI & 14 days supply Ca(OH)2	0.46	1.52
Cost of Construction Downtime (21 days) <sup>[2]</sup>	0.00	0.00
Total Capital Requirement	3.86	12.86

<sup>[1]</sup> No allowance included based on DOE guideline

<sup>[2]</sup> Assumed downtime to be during scheduled major outage

# -7.2.3 SI - Projected Operating Maintenance Cost

EER evaluated the fixed and variable operating costs of an SI system for a 300 MWe coal cyclone-fired power plant; contributing cost factors were as follows:

- 1. <u>Cost of Hydrated Lime Sorbent</u> The purchased price for hydrated lime was based on the costs incurred for the CCT demonstrations, \$83/ton.
- 2. <u>Sootblowing</u> The frequency of sootblowing was increased for a power plant with an SI system.
- 3. <u>Ash Disposal</u> An increase of ash disposal results from the addition of sorbent to the boiler furnace.
- 4. <u>Auxiliary Power</u> The power increases due to the added air blower and air compressor horsepower requirement
- 5. <u>Maintenance Items/Spare Parts</u> An allowance of 3% of the total plant investment was used for total maintenance, 40% of this 3% was allocated for maintenance items and spare parts.
- 6. <u>Maintenance Labor</u> An allowance of 3% of the total plant investment was used for total maintenance, 60% of this 3% was allocated for maintenance labor.
- 7. <u>Administration and General Overhead</u> An allowance of 60% of plant labor was added to cover administration and general overhead.
- 8. <u>Local Property Taxes and Insurance</u> An allowance of 3% of total plant investment was used to cover taxes and insurance.

The total annual incremental gross operating cost for the SI system, exclusive of any payback of capital, is estimated at \$8,610,679. The operating cost breakdown for the SI system retrofit for a 300 MWe cyclone-fired unit is presented in Table 7-9.

# 7.2.4 SI - Summary of Performance and Economics

Based on the developed capital and operating costs, economic projections were made using current dollars, which include an inflation rate of 4.0%, and constant dollars which

# TABLE 7-9. 300 MWe CYCLONE-FIRED UNIT SI OPERATING COST

# Annual Incremental Operating Costs<sup>[1]</sup>

	Annual Use	Cost/Unit	Cost/ Yr
Variable Costs	-		
Raw Material: Hydrated Lime <sup>[2]</sup>	87,327 tons	\$83.00 /ton	\$7,248,101
Utilities: Electricity	10,480 10 <sup>3</sup> kVVhr	\$20.00 /10 <sup>3</sup> kWhr	\$209,597
Ash Disposal	97,759 tons	\$9.29 /ton	\$908,180
Sub-Total			\$8,365,878
Fixed Costs			
Labor: <sup>[3]</sup> Maintenance ( 3% of TPI x 60%) Supervision ( 20% of Maintenance I	Labor)		\$59,226 \$11,845
Supplies: Maintenance (3% of TPI x 40%)			\$39,484
Admin. and Gen. Ovhd. (60% of total	labor)		\$35,536
Local Taxes and Insurance @ 3% of	TPI		\$98,710
Sub-Total			\$244,801
Total Operating Cost			\$8,610,679

<sup>[1] 65%</sup> Capacity factor @ 300 MWe net capacity (10,000 Btu/kWhr heat rate)

<sup>[2] 95%</sup> Ca(OH)2 with Ca/S ratio of 1.75

<sup>[3]</sup> No incremental operating labor

ignore inflation. Table 7-10 shows the performance and cost for an SI System that is retrofitted to a 300 MWe cyclone-fired boiler. The incremental increase in the levelized cost of power is estimated at 5.32 mills/kWhr in constant dollars and 6.98 mills/kWhr in current dollars. The levelized cost of SO<sub>2</sub> removal is estimated at \$490/ton and \$643/ton for constant and current dollar projections, respectively.

#### 7.2.5 SI - Effect of Variables on Economics

The cost of SO<sub>2</sub> reduction was analyzed and certain variables were then selected to perform sensitivity analyses. Based on the levelized cost (constant dollars) for reducing sulfur dioxide emissions, the capital charge component made up only 5.3% of the total cost. The fixed operation and maintenance costs represented only 2.6%, and the variable cost (92.1%) made up the rest of the cost for removing SO<sub>2</sub>. The hydrated lime accounted for some 87% of the variable costs.

The variables chosen were the cost of hydrated lime, the capacity factor, the unit size and the percentage of sorbent utilization. For the sensitivity analysis, only the variable being analyzed was changed with all other variables being kept the same. The hydrated lime and how well it is utilized is the driving force for the economics of the SI System. The effect of the variables analyzed, on the cost of SO<sub>2</sub> removal, is shown in Table 7-11.

A \$20/ton swing in the price of hydrated lime has the effect of changing the cost of  $SO_2$  removal by some \$95/ton. The effect of capacity factor has a relatively small effect on the cost of  $SO_2$  removal. A 10% swing in capacity factor results in a \$5-\$7/ton variation in the cost of  $SO_2$  removal. The capital cost of the 300 MWe unit was used as a base and a scale up factor of 0.7 was used to extrapolate the capital cost for larger units. This factor was based on a combination of some 70% of the equipment being increased in size using a 0.6 scaleup factor and some 30% of the equipment being duplicated using a 0.9 scale up factor.

### TABLE 7-10. SI RETROFIT COST AND PERFORMANCE SUMMARY

# Summary of Data

#### **Power Plant Attributes**

	Units	Value
Plant capacity, net	MMe	300
Power produced, net	10 <sup>9</sup> kWhr/yr	1.71
Capacity factor	%	65
Plant life	yr	15
Coal feed	tons/yr	683,280
Sulfur in ∞al	wt%	3.0

### **Emissions Control Data**

	Units	SO <sub>2</sub>
Removal efficiency	%	45
Emissions standard	lb/10 <sup>6</sup> Btu	1.20
Emissions without controls	lb/10 <sup>6</sup> Btu	4.80
Emissions with SI control	lb/10 <sup>6</sup> Btu	2.64
Amount reduced	tons/yr	18,654

### Levelized Cost of Power

	Current Dollars		Consta	nt Dollars
	Factor	Mills/kWhr	Factor	Mills/kWhr
Capital Charge	0.160	0.36	0.124	0.28
Fixed O&M	1.314	0.19	1.000	0.14
Variable Operating Cost	1.314	6.44	1.000	4.90
Total Cost		6.98		5.32

# Levelized Cost - SO<sub>2</sub> Removal Basis

	\$/ton			\$/ton	
	Factor	removed	Factor	removed	
Capital Charge	0.160	33	0.124	26	
Fixed O&M	1.314	20	1.000	15	
Variable Operating Cost	1.314	589	1.000	448	
Total Cost		643		490	

Basis: 300 MWe unit size and 65% onstream capacity factor

26% calcium utilization @ Ca/S ratio of 1.75 and \$83/ton price for 95% Ca(OH)2 hydrated lime

TABLE 7-11. COST OF SO<sub>2</sub> REMOVAL\*

Hydrated Lime, \$/ton	63	73	83	93	103	
SO <sub>2</sub> Removal Cost, \$/ton	395	442	490	537	584	
Capacity Factor,%	50	55	60	65	70	75
SO <sub>2</sub> Removal Cost, \$/ton	501	497	493	490	487	485
Unit Size, MWe	100	300	500	700	900	
SO <sub>2</sub> Removal Cost, \$/ton	530	490	480	476	473	
Sorbent Utilization,%	26	36	46			
SO <sub>2</sub> Removal Cost, \$/ton	490	354	277			

Note: Base case variables in bold

The unit size has a moderate effect on the cost of SO<sub>2</sub> removal. For a 900 MWe unit, the cost of SO<sub>2</sub> removal is only some \$17/ton less than that for the 300 MWe unit. For smaller units the effect is greater; for a 100 MWe unit the cost of SO<sub>2</sub> removal would be some \$40/ton greater than that for a 300 MWe unit. This is an economy of scale effect with the reduction in cost starting to taper off at the 300 MWe size.

The sorbent utilization dramatically impacts the cost of  $SO_2$  removal because it affects the consumption rate of sorbent. For this economic analysis a Ca/S ratio of 1.75 was used with an overall calcium utilization of 26%. An increase in sorbent utilization from 26% to 46% would result in a drop of  $SO_2$  removal cost from \$490/ton to \$277/ton.

# 7.1.6 SI - Comparison with other SO<sub>2</sub> Control Technologies

The primary methods of coal-fired utilities used in complying with the 1990 Clean Air Act Amendments (CAAA) include switching from high sulfur Eastern coal to low sulfur Western coal, purchase of SO<sub>2</sub> allowance credits and stack gas scrubbing.

<sup>\*</sup> Constant Dollar Basis

### Fuel Switching

Fuel switching from high sulfur to low sulfur coals has seen increased interest since the passage of the 1990 CAAA, especially with the promotion very low sulfur coals from the Powder River Basin (PRB) of Wyoming. PRB coals are ranked as sub-bituminous and generally have a sulfur content of one half percent, but also have higher ash and moisture contents and lower heating values. In considering coal switching to low sulfur coals, fuel costs (including transportation) and operational or performance impacts are of major concern.

PRB coal (as mined) has been very competitively priced as compared to eastern and Midwestern bituminous coals, with transportation costs weighing heavily in the feasibility determination. Transportation of PRB coals to the Midwestern and Eastern U.S. has been exclusively by rail. Utilities switching to PRB coal are required to contract with one or more rail operating companies for transportation, and therefore become subject to future rate hikes or labor strikes which would interrupt delivery.

Delivery of design coal to an existing facility may have been by truck or river barge, and the plant may not have rail access or rail unloading facilities. Addition of rail spurs, car unloading equipment, handling equipment, and coal storage piles must be considered in the feasibility of the fuel switch. Utilities may also consider purchasing unit trains and yard locomotives as apposed to long term contracting for these services.

The moisture and ash content and lower heating value of PRB coal can have a significant impact on plant performance, and modifications may be required to accommodate the fuel. Milling capacity should be reviewed in terms of drying the higher moisture coal and increasing the mass through the pulverizers in order to avoid a unit derate. Spare mills may be lost due to the milling requirements, and modifications may be required to provide additional mills or additional heat input to the mills for drying of the coal. Since the western fuel has a higher moisture content, heat loss from the unit via the flue gas will increase as

will the required firing rate. Due to the increased ash content, impacts on furnace slagging and convective section fouling must be addressed in terms of gas temperature profile changes and sootblower coverage. Increased ash loading will also affect the particulate collection device (ESP or baghouse) as well and ash handling systems and disposal.

### SO<sub>2</sub> Allowance Credits

Allowance credits may be purchased by the utility, emissions being set by the U.S. EPA and determined as SO<sub>2</sub> emissions emitted in a base year between 1985 and 1987. The current NSPS also requires that 70 to 90 percent plus SO<sub>2</sub> must be removed based on the sulfur content of the coal. Surplus allowance credits may result from fuel switching (low sulfur coal, oil, or gas), or over-scrubbing by any of the flue gas desulfurization technologies. The cost of credits is set by the supply and demand for such credits, being heavily influenced by the cost of delivery of PRB coal and control technology costs.

### Flue Gas Desulfurization (FGD)

Many FGD processes have been developed, differing in the chemical reagents used in the process and the resulting end products or waste. Most processes return an unusable or "throw-away" product, while a small portion provide marketable sulfur or sulfate based products. Recovery processes require sophisticated chemical processing plants and the capital investment may be unattractive. These processes return reagent quality products and are not to be confused with FGD systems producing sludges that are disposed of sold, for example, as concrete admixtures. Discussion of all of the available processes is well beyond the scope of this manual, but the processes can be classified into three distinct categories; wet scrubbing, spray drying, or dry injection.

The wet scrubbing process involves exposure of the flue gas to sprays of an alkaline slurry mixture of water with crushed limestone or slaked lime. Other alkaline reagents such as sodium carbonate have also been used. The spraying of the alkaline slurry and absorption

of the sulfur-dioxides in the flue gas occurs in a variety of absorber designs including venturi scrubbers, static packing scrubbers, moving bed scrubbers, tray tower absorbers, and spray tower absorbers. In the wet scrubbing process, only a portion of the water in the spray is evaporated as the flue gas is cooled to the adiabatic saturation temperature. The evaporation that occurs in the scrubbing process results in water vapor being discharged to the atmosphere, requiring make up water to be continually added to the process. Reheaters are often provided at the scrubber outlet to prevent condensation and corrosion problems in the downstream duct work, draft fans, and stack. End products are retained in the liquid scrubber effluent, and continuously removed by clarification and vacuum filtration producing the discharge known as "filter cake". Wet systems are located downstream of a particulate collection device to minimize fly ash contamination of the process and resulting end product.

In spray drying systems, flue gas is contacted with very finely atomized alkaline or slurry solutions. Evaporation of the water and absorption of the sulfur dioxide in the flue gas occurs almost simultaneously. The resulting end products are completely dry and can be collected by conventional particulate control devices such as precipitators and baghouses, and transported by conventional fly ash handling equipment. Spray drying and residence time in the flue gas prior to collection provides the initial contact between the atomized reactive slurry and the acid gas contaminants. The particulate control device is an integral part of the spray drying process providing additional contact time between the reactant and the flue gas. Spray drying processes are therefore better suited for application with baghouses rather than ESP's since the flue gas must pass through the cake of reactants on the bags. Dry scrubbing offers the advantages of minimal water consumption and elimination of reheat requirements since the flue gas is not reduced to the adiabatic saturation temperature as with wet scrubbing. However, dry scrubbing is usually economically limited to coals with low to medium sulfur contents since more expensive reagents such as lime or sodium carbonate are required.

As described in this manual, SI involves pneumatic injection of dry sorbents such as hydrated lime or limestone into the furnace flue gases at temperatures of 2000 to 2250°F. There are no on-site processing equipment requirements for the raw sorbent material since local suppliers can make adjustments in their production for the grind requirements of limestone and in the hydrating process for lime. SI also does not require slaking or slurry preparation equipment, or expensive absorber or spray towers.

The process equipment for SI includes silo storage, feed, transport, and injection equipment, and the capital cost is a fraction of wet or dry scrubbing systems. SI, however, does not offer the reduction capabilities of these competing technologies. SO<sub>2</sub> capture with SI ranges from approximately 30 to 50 percent depending on the sulbur dioxide level in the flue gas (proportional to the sulfur in the coal), the type and feedrate of sorbent to be used, and the available residence time at desired temperatures in the boiler furnace. Higher removal efficiencies and sorbent utilization is realized with more reactive and expensive sorbents such as hydrated lime, which accounts for a significant portion of the operating costs.

SI also requires balance of plant modifications due to increased fouling of radiant and convective sections of the boiler, as well as impacts on particulate control devices, ash handling equipment, and ash disposal procedures. A possible benefit of DSI is the significant increase of the calcium content of the fly ash, improving the salability of the ash as a concrete admixture. If the product cannot be sold, disposal requirements may increase the SI operating costs.

With the relatively lower capital costs and higher operating costs as compared to the wet or dry scrubbing systems, SI might be suited for application to aging plants with low capacity factors or relatively short operating lives. In order to reach full compliance while using SI, utilities may consider allowance purchasing, or over-scrubbing on newer units in their system. The SO<sub>2</sub> removal cost of SI is compared to wet scrubbers, coal switching and the purchase of SO<sub>2</sub> allowance credits in Table 7-12.

TABLE 7-12. SI COMPARISON WITH OTHER SO<sub>2</sub> CONTROL OPTIONS

Technology	SO₂ Removal	SO₂ Removed \$/ton
SI	30-50	490¹
SO <sub>2</sub> Scrubbers	90-95	322 <sup>2</sup>
Switch to Low Sulfur Coal	65-85	113³
SO <sub>2</sub> Allowance Credits	0-100	110⁴

- (1) Nominal cost for 300 MWe unit based on 26% calcium utilization and hydrated lime at \$83/ton
- (2) Power Generation publication, 4/4/97, based on average cost for 80 scrubbers (1996 survey)
- (3) Power Generation publication, 4/4/97, average of 112 units that switched to low sulfur Western coal
- (4) Power Generation publication, 4/4/97, average selling price in 1996

With current economics, the SI technology will probably not be cost competitive with other available utility options for base and intermediate load units unless the SI calcium utilization can be increased and the cost of the lime based reagent decreased.

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