NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS

WARTIME REPORT

ORIGINALLY ISSUED
February 1945 as
Advance Confidential Report 4J28

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ON CONCAVE WALLS

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WASHINGTON

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ADVANCE CONFIDENTIAL REPORT

INVESTIGATION OF BOUNDARY LAYER TRANSITION
ON CONCAVE WALLS

By H. W. Lempmann

SUMMARY

Transition of the boundary layer from the laminar to the turbulent regime was investigated on the concave side of a plate with a radius of curvature of 2.5 feet. The critical Reynolds number was found to be considerably lower than on a flat plate and on the concave side of a plate with a 20-foot radius of curvature previously investigated. It was furthermore found that, in agreement with the theoretical results of Göttler, \( \frac{R}{\theta} \), here termed the "Görtler parameter," is the proper critical parameter governing boundary layer instability due to concave curvature. The critical parameter at transition was found to have a value of 9.6.

The influence of pressure gradient and of an increased free-stream turbulence level on the position of the transition point on the concave side of the plate of 2.5-foot radius of curvature has been studied. Small variations of the pressure gradient did not alter the value of the critical Göttler parameter. This result is compared with similar measurements on the convex side of a plate of 20-foot radius of curvature. Increased tunnel turbulence lowered the value of the critical parameter to 3.0 at a turbulence level of 0.05 percent to 6.0 at a turbulence level of 0.3 percent.

The investigation confirms the previous result that the mechanism of the breakdown of the laminar boundary layer is essentially different on convex and concave boundaries.

A discussion of the practical applicability of transition measurements is given, and the difference between critical Reynolds number corresponding to laminar instability and critical Reynolds number corresponding to transition is pointed
cut. A definition of transition Reynolds number, based on
the apparent shearing stress caused by the laminar oscillations, is given. In the case of flow along a flat plate,
values of transition Reynolds number are calculated approxi-
mately for different intensities of the initial disturbance.

INTRODUCTION

The larger part of the California Institute of Technology
investigation on the effect of curvature on boundary layer
stability and transition has been previously reported (refer-
ence 1). This previous research led to the conclusion that
the mechanism of the breakdown of the laminar boundary layer
is essentially different on concave and convex walls. The
instability of the laminar boundary layer on convex boundaries
was found to be brought about by plane disturbances, the so-
called Tollmien–Schlichting waves, which were first observed
by Erydén, Schubauer, and Skramstad on a flat plate. The
range of unstable frequencies and the amplification character-
istics of the Tollmien–Schlichting waves were found, within
the experimental scatter, to be the same on the convexly
curved boundaries as had earlier been observed by Schubauer
and Skramstad (reference 2) on the flat plate. It was accord-
ingly concluded that the transition point was essentially un-
aaffected by convex curvature. These measurements extended
over a range of effective curvature $\theta/r$ from $0$ to $10^{-3}$,
where $\theta$ is the momentum thickness of the boundary layer and
$r$ is the radius of curvature of the boundary.

Concave curvature was found to have a pronounced destabi-
lizing effect on the laminar boundary layer, the critical
Reynolds number decreasing with increasing effective curvature.
It was, therefore, concluded that the mechanism of transition
on concave walls was different from that on flat and convex
boundaries. It was furthermore shown that these observations
were in general agreement with theoretical results of Görtler
(reference 3) which predicted a strong dynamic instability due
to three-dimensional disturbances on concave boundaries. The
measurements for the case of concave boundaries were, however,
not so extensive as for the convex case and covered only the
range of effective curvature between $0$ and $0.0001$. One pur-
pose of the investigations reported here is the extension of
the previous measurements to values of effective curvature of
about $0.001$ on a concave boundary.

The influence of the pressure gradient along the laminar
boundary layer on the position of the transition point has long
been known. The measurements which have been carried out before were always confined to flat-plate flow or flow past convex boundaries. Based on Gortler's theoretical results it was suggested in the previous report that the influence of pressure gradient on transition should be less pronounced on concave boundaries. Consequently, measurements of the effect of pressure gradient on the position of the transition point have been carried out and the results are presented in the present report. The effect of an increased free-stream turbulence level on transition was also studied.

The investigations reported here confirm and supplement the results reported previously, and thus conclude the research program on the influence of curvature on boundary-layer transition carried out over a number of years at the California Institute of Technology under the sponsorship and with the financial assistance of the National Advisory Committee for Aeronautics.

The continuous interest, encouragement, and advice of Dr. Th. von Kármán and Dr. C. C. Millikan is gratefully acknowledged. The author also wishes to acknowledge many helpful discussions with Dr. C. C. Lin, and the assistance of Mr. J. Laufer.

SYMBOLS

- \( x \) distance along surface of plate from leading edge
- \( y \) distance perpendicular from surface
- \( z \) distance along surface perpendicular from leading edge
- \( r \) radius of curvature of the boundary layer
- \( U \) mean velocity at a point in the boundary layer
- \( U_0 \) mean velocity of the free stream
- \( u \) instantaneous \( x \)-component of fluctuation velocity
- \( v \) instantaneous \( y \)-component of fluctuation velocity
- \( w \) instantaneous \( z \)-component of fluctuation velocity
\[ u', v', w' \] root-mean-square values of \( u, v, \) and \( w, \) respectively
\[ \frac{u'}{U_0} \] etc., turbulence level, usually expressed in percent
\( \rho \) density
\( \mu \) viscosity
\[ \nu = \frac{\mu}{\rho} \] kinematic viscosity
\[ q = \frac{1}{2} \rho U_0^2 \] dynamic pressure of the free stream
\[ \eta = \gamma \frac{U_0}{\sqrt{\nu x}} \] Blasius' nondimensional parameter
\( \delta \) boundary layer thickness
\( \delta^* \) boundary layer displacement thickness
\( \theta \) boundary layer momentum thickness
\( \alpha_r \) effective curvature of the boundary layer
\( R \) Reynolds number
\[ R_x, R_\theta, R_\delta^* \] Reynolds numbers based on \( x, \theta \) and \( \delta^* \), respectively, \( R_x = U_0 x / \nu = x - \text{Reynolds number} \), and so forth
\[ R_{x tr}, R_{\theta tr}, R_{\delta^* tr} \] Reynolds numbers at the transition point
\[ R_\theta \sqrt{\frac{\theta}{\alpha_r}} \] Görtler parameter
\[ \tau_A = -\rho \overline{u'v'} \] apparent shear
\[ \tau_L = \mu \frac{du}{dy} \] laminar shear
\[ k = \frac{uv}{u^2} \]
correlation coefficient

\[ \sigma_{fL} \]
laminar skin-friction coefficient

APPARATUS AND METHODS

Wind Tunnel and Test Section

The general layout of the wind tunnel is shown in figure 1. A detailed description can be found in reference 1. The measurements were carried out on the concave side of a smooth glass plate of 2.5-foot radius of curvature. This plate was set in the center of the high (5 ft) and narrow (7.5 in.) test section shown in figure 2. The section consists essentially of two plate-glass plates forming cylinders concentric with the center sheet, on which the measurements are made. These plates are mounted in wooden frames which, by means of a screw adjustment, allow a variation of the width of the channel and of the angle of attack of the center plate. This adjustment makes it possible to alter the pressure gradient along the test section. Owing to the geometry of the section it is, however, difficult to set for a predetermined simple pressure distribution, for example, a linear distribution. The investigations on the effect of pressure gradient on transition were, therefore, rather tedious and the investigated range of pressure gradients limited.

Free-Stream Turbulence Level

The normal free-stream turbulence level of the tunnel was the same as in the previous measurements. The values were

\[ \frac{u'}{U_0} = 0.08 \text{ percent} \]

\[ \frac{v'}{U_0} = \frac{w'}{U_0} = 0.12 \text{ percent} \]

The turbulence level in the test section is mainly controlled by the precision screen "S" (fig. 1) in the pressure chamber. To investigate the position of the transition point at higher
turbulence levels of the free stream, the tunnel turbulence was artificially raised in the following way: Strips of celluloid tape about 1 inch wide and 5 inches long were glued at regular intervals of about 3 inches on the downstream side of the screen S. These strips produced a uniform turbulence level of

\[
\frac{u^i}{U_0} = 0.3 \text{ percent}
\]

in the test section. By removing every second strip, and so forth, the turbulence level could then be lowered to the normal free stream level.

Traversing Mechanism

The traversing mechanism used in this wind tunnel is described in detail in reference 1. The motion of the instrument, for example, hot-wire anemometer, is continuous in the x and y directions and is remotely controlled.

Hot-Wire Anemometer

The mean-speed distribution in the boundary layer was measured with platinum wires of 0.0005-inch diameter and about 5-millimeter length. The turbulence level and the position of the transition point were determined with 0.00024-inch thick and about 2-millimeter long wires. Mean speed was always measured using the constant-resistance method. For measurements of velocity fluctuations the wire was calibrated using the electrical oscillator method (reference 4) and the amplifier properly compensated by means of an inductance-resistance circuit. The amplifier response is flat between less than 6 cycles up to 8000 cycles.

Measurement of the Pressure Distribution

The pressure distribution along the test section was measured by means of a small static tube mounted on the instrument carriage at a distance of about 1 centimeter from the test plate. Thus, the pressure gradient close to the edge of the boundary layer was measured. The irregular "waves" in the distribution (fig. 3) are due to local changes in the radius of curvature of the plate.
Determination of the Transition Point

The transition point was determined by means of the hot-wire anemometer. The hot wire was mounted on the instrument carriage a few thousandths of an inch from the surface of the plate. The carriage was then moved slowly in the direction of the mean flow from the leading edge downstream; the first appearance of the turbulent bursts (see reference 1) on the oscilloscope screen was taken as indication of transition.

MEASUREMENTS AND RESULTS

Mean Speed Distribution

The test-section walls were adjusted to give a pressure gradient as close to zero as possible. The best result which could be obtained is shown as "A" in figure 3. The pressure gradient is seen to be close to zero over the first 30 centimeters of the test plate. Since transition was found to occur at distances from the leading edge of less than 60 centimeters, this pressure gradient 

\( A \) (fig. 3) could, with sufficient approximation, be taken as "zero" gradient. The velocity distribution in the laminar boundary layer was then measured. The result is shown in figure 4. It is seen that the profile follows, rather closely, the Blasius flat-plate profile. There exists a slight systematic deviation from the Blasius profile especially in the range between \( \eta = 3 \) and \( \eta = 6 \). The measured values of \( \frac{u}{U_0} \) are slightly lower than the ones given by the Blasius distribution.

The main purpose of the measurements was to establish whether computations of the momentum thickness \( \delta \), the Reynolds number based on \( \delta \), and so forth could be computed from the Blasius solution. For computation of these data applied to transition measurements, the slight deviation seen in figure 4 is immaterial and consequently the values of \( \delta \), \( \delta^* \), and so forth, used later on, have been computed from the Blasius formula, that is,

\[
\delta = \frac{2}{3} \sqrt{\frac{\nu x}{U_0}}
\]

\[
\delta^* = 1.73 \sqrt{\frac{\nu x}{U_0}}
\]
Effect of Curvature on the Position of the Transition Point

Transition was measured with the hot-wire anemometer. By varying the velocity of the free stream, the range of effective curvature from \(-0.5 \times 10^{-4}\) to \(-1.1 \times 10^{-4}\) could be investigated. The results of the tests are shown in figure 5 where the critical Reynolds number \(R_{x\text{tr}}\) is plotted versus \(\frac{\theta}{r}\). It is seen that \(R_{x\text{tr}}\) falls from about \(\times 10^5\) at \(\frac{\theta}{r} = -5 \times 10^{-4}\) to \(\times 10^5\) at \(\frac{\theta}{r} = -11 \times 10^{-4}\).

This result agrees very well with previous measurements (reference 1). Figure 6 shows the effect of curvature on transition for effective curvature ranging from \(-0.001\) (concave) to \(+0.001\) (convex). Here \(R_{\text{tr}}\) is plotted versus \(\frac{\theta}{r}\). It is seen that the decrease of the critical number found on the concave side of the 20-foot plate continues with higher effective curvature.

According to Görtler (reference 3) \(R_{\theta} \sqrt{\frac{\theta}{r}}\) is the characteristic parameter for the three-dimensional instability on concave walls. If the value of this parameter at transition is plotted versus effective curvature (fig. 7a), it is seen that within the rather large scatter the value of \(\left(\frac{R_{\theta} \sqrt{\frac{\theta}{r}}}{\text{tr}}\right)\) does not systematically vary with \(\frac{\theta}{r}\). The average value of \(\left(\frac{R_{\theta} \sqrt{\frac{\theta}{r}}}{\text{tr}}\right)\) found is about 9.0. The value of the parameter as measured here is somewhat larger than the value of 7.3 found previously on the concave side of the plate of 20-foot radius. There exists a possible explanation for this difference in the value of the Görtler parameter at transition which can best be seen from a plot such as figure 8.

Instead of plotting the Görtler parameter as a function of effective curvature \(R_{x\text{tr}}\) is plotted as a function of \(\frac{R_{\theta}}{\text{tr}}\). Since \(R_{x} \approx R_{\theta}^2\), \(R_{\theta} \sqrt{\frac{\theta}{r}}\) = constant corresponds to straight lines through the origin in this plot. The measured values of \(R_{x\text{tr}}\) for the two test sections of 20-foot and 2.5-foot radius are presented in figure 8, and in addition, the line \(R_{\theta} = 9.4 \times 10^5\) corresponding to transition on a
flat plate in the wind tunnel is shown. Assume that transition due to Görtler vortices occurs for all concave curvatures at the value of $R_\theta \sqrt{\frac{\theta}{r}}$ found on the plate of 2.5-foot radius. All transition measurements should thus fall on the straight line $(R_\theta \sqrt{\frac{\theta}{r}})_{tr} = 9.0$ (fig. 8). Transition on the flat plate, that is, transition caused by the two-dimensional perturbation, was found to occur at $R_{\theta tr} = 9.4 \times 10^{-2}$. Since the Tollmien-Schlichting wave is but little affected by small curvature, it can be expected that transition caused by this disturbance on slightly concave plates occurs at about the same $R_\theta$. Hence, starting from the flat plate, all transition measurements should follow the straight line $R_\theta = 9.4 \times 10^3$ in figure 8. At $\frac{R_\theta}{\theta} = -11 \times 10^3$ the line $(R_\theta \sqrt{\frac{\theta}{r}})_{tr} = 9.0$ and $R_\theta = 9.4 \times 10^{-2}$ intersect; that is, at effective curvatures larger than $-11 \times 10^3$ transition is caused by the three-dimensional disturbance. At smaller effective curvature the Tollmien-Schlichting wave is more unstable and thus is responsible for transition. It appears that this is not the case, but that a more or less continuous change from the transition due to Görtler vortices to transition caused by Tollmien-Schlichting waves takes place. Since the laminar boundary layer on concave walls is unstable with respect to small perturbations of both kinds, such a continuous change is possible and probable. Both types of perturbation are excited by the external free-stream turbulence, traveling downstream, that is, toward higher values of $R_\theta$, both the waves or the vortices of a given wavelength increase in amplitude when they pass through their respective instability region in the $R_\theta$-$\theta$ diagram (see reference 1). A transfer of energy from the one type of perturbation to the other is therefore quite possible and such a transfer becomes unimportant only if one of these perturbations is considerably more unstable than the other, that is, in the limiting cases of small and large effective curvature. A more exact analysis of such an energy transfer is impossible without a detailed knowledge of the amplification of both types of perturbation and furthermore of the excitation due to free-stream turbulence.
Effect of the Pressure Gradient on Transition

It is well known that a pressure gradient along a laminar boundary layer strongly influences the position of the transition point. The so-called low-drag or laminar flow airfoil represents an application of this result. All investigations on the influence of the pressure gradient have previously, however, been confined to measurements on flat or convexly curved boundaries, that is, to boundary layer flows, where the Tollmien–Schlichting instability is predominant. Based on the theoretical analysis of Görtler (reference 3) it was suggested previously (reference 1) that the influence of the pressure gradient on transition should be less pronounced on concave walls where the boundary-layer instability should be due to the three-dimensional vortices.

To investigate this question, measurements of the influence of pressure gradient on transition were carried out on the convex side of the plate of 30-foot radius and on the concave side of the plate of 2.5-foot radius. Thus, a typical Tollmien–Schlichting and Görtler case could be compared. The pressure distributions for which transition was investigated are shown in figures 3 and 9, respectively. Before a discussion and comparison of the measurements can be presented, the question of the proper parameter for the pressure gradient and the proper critical number for transition must be discussed.

It is sometimes useful to present transition data in the form of a critical Reynolds number $R_x$ based on the distance from the leading edge, that is, from the stagnation point. However, $R_x$ certainly is not the proper physical parameter, and it is preferable to base the critical number on some measure of the local boundary layer thickness. The momentum thickness appears as the best choice since it is closely connected with the shearing stress at the wall and thus with the slope of the velocity profile at the wall. In this connection the instability of a velocity profile with respect to Tollmien–Schlichting waves is known to depend very much on the slope (and curvature) of the profile in the neighborhood of the wall. Görtler found that for the three-dimensional disturbance also momentum thickness is the most suitable length for the critical parameter. Thus it appears reasonable to plot for flat and convex surfaces the critical number $Re_{tr}$ and for the concave surfaces $(R_0 \sqrt{\frac{\partial}{\partial x}}_{tr})$ as functions of some effective pressure gradient. To represent the pressure gradient, that is, $\frac{1}{q} \frac{dp}{dx}$ in dimensionless form, a length is again needed.
The effective force acting on an element of the boundary layer is the force due to the pressure gradient, \( \frac{\delta \, dp}{q \, dx} \).

The boundary layer thickness \( \delta \) entering here has more kinematic than dynamic character and thus the displacement thickness \( \delta^* \) appears as the logical choice. This can also be seen from von Kármán's momentum equation of the boundary layer

\[
\frac{T_0}{q} = \frac{1}{q} \frac{d(f \theta q)}{dx} - \frac{\delta^* \, dp}{q \, dx}
\]

where also the combination \( \frac{\delta^* \, dp}{q \, dx} \) appears.

The results of the measurements are, therefore, represented in the form \( R_e \, \text{tr} \) and \( \left( R_e \, \sqrt{\delta^* \over r} \right) \), respectively, versus \( \frac{\delta^* \, dp}{q \, dx} \). It is, of course, not to be expected that a complicated phenomenon, such as the laminar instability for flow with pressure gradient, can be described by one critical parameter alone. Under the influence of the pressure gradient, the velocity profile of the layer will, in fact, change continuously from the stagnation point up to eventual separation.

The instability zones for small perturbations and the amplification characteristics thus vary continuously downstream. Transition occurs if perturbations are amplified to a sufficient degree. Since the total amplification is thus an integral effect depending on all instability zones passed by the perturbation on its way from its origin downstream to the transition point, it seems hardly possible to give a representation involving only one critical number such as \( R_e \). Only where the instability zones are little affected by a change in the profile as, for example, according to Görtler (reference 3), in the case of flow past concave boundaries, can it be expected that it will be possible to represent the instability character by only one critical parameter, in this case \( R_e \, \sqrt{\delta^* \over r} \). Except for pressure gradient "C" on the convex side of the plate of 20-foot radius, the momentum thickness and displacement thickness were again computed using the Blasius solution. The influence of these comparatively small gradients on the boundary-layer thickness was found to be negligible within the scatter of the transition measurements. In the case of gradient C in figure 9, the momentum thickness and displacement thickness were computed from the solution of the laminar boundary-layer equations for the case of a linear decrease of the free-stream velocity. The experimentally determined
pressure distribution was approximated by a parabola, and 
the momentum and displacement thicknesses were determined 
from the tabulated solution (reference 5). It is seen from 
figure 10 that for the case of convex curvature $R_{tr}$ de-
creases with positive and increases with negative pressure 
gradient as expected. The largest decrease with positive 
pressure gradients occurs close to zero pressure gradient. 
Whether or not this is true for negative gradients remains 
undecided since no measurements at larger negative values of 
$\Delta^* \frac{dp}{q \, dx}$ could be carried out due to the limited Reynolds number 
of the test plates. The rapid change of $R_{tr}$ in the neigh-
borhood of zero pressure gradient and the comparatively small 
change at larger positive values of $\Delta^* \frac{dp}{q \, dx}$ is interesting. 
Thus it is indicated that the Blasius solution is singular 
in the sense that a slight variation of the profile from the 
Blasius solution does cause large changes in the stability 
character. This is not unreasonable since the Blasius solu-
tion is indeed a singular solution with $\frac{\partial^2 u}{\partial y^2} = 0$ at $y = 0$.

A theoretical analysis of the instability character of pro-
files where the pressure gradient is not zero but is small 
will be required for a physical explanation of this pressure-
gradient effect.

The absolute values given in figure 10 for $R_{tr}$ as a 
function of $\Delta^* \frac{dp}{q \, dx}$ are certainly not universally applicable 
since they are strongly influenced by the wind-tunnel charac-
teristics, that is, turbulence level, turbulence spectrum, 
and so forth (see also reference 1). The relative variation 
of the critical Reynolds number at transition with pressure 
gradients, however, is believed to be quite general. Conse-
quently, it is to be expected that in practical applications, 
such as the laminar flow wing, and so forth. the movement of 
the transition point when the pressure gradient is altered 
is most pronounced in the neighborhood of zero gradient.

Figure 11 shows the effect of pressure gradient on 
transition on the concave side of the plate of 2.5-foot 
radius. It is seen that the critical parameter for transition 
of the boundary layer decreases with increasing pres-
sure gradient. This effect hardly exceeds the scatter of 
the points, however, and is not believed to be actually 
systematic. A comparison of the values of the critical pa-
rameter for transition at positive gradients with the range
of values at zero pressure gradient shows that the apparent
decrease at positive pressure gradient lies within the ex-
perimental error. The comparatively large scatter of the
points is due to the inherent difficulty of transition meas-
urements on a plate with a small radius of curvature and
also to errors in the determination of the boundary layer
thickness which enters both parameters \( R_x \sqrt{\vartheta \rho} \) and \( \delta^* \frac{dp}{dx} \).
The measurements were taken at small values of \( R_x \) because
of early transition, and thus \( \vartheta \) and \( \delta^* \) vary comparatively
rapidly with \( x \). An error in \( x_{tr} \) thus produces larger
errors in \( \theta_{tr} \) and \( \delta^*_{tr} \) than appears in the case of the
convex plate where \( R_{x, tr} \) was five to ten times larger. The
present investigation accordingly indicates that the influence
of pressure gradient on transition at concave surfaces is
either zero or at most very small.

**Effect of Free Stream Turbulence on Transition**

That free-stream turbulence has a strong influence on
the position of the transition point is well established.
The first systematic investigation of this effect was made
by Schubauer and Skramstad (reference 2), who measured the
effect of varied free stream turbulence level on transition
of the boundary layer on a flat plate.

In the present investigation, transition was measured
on the concave side of the plate of \( 2 \text{ ft} \) foot radius for
three turbulence levels of the free stream. The results
are given in figure 7. It is seen that an increased tur-
bulence level leads to earlier transition. The critical number
\( \left( R_x \sqrt{\vartheta \rho} \right)_{tr} \) decreases from 9.0 with \( \frac{u'}{U_0} = 0.06 \) percent to
6.0 with \( \frac{u'}{U_0} = 0.3 \) percent. The magnitude of the turbulence
effect is of the same order as in the case of a flat plate
(reference 2). Thus it appears that the three-dimensional
disturbances, like the Tollmien-Schlichting waves, are ex-
cited by turbulence in the free stream. A more detailed
discussion of the turbulence effect meets, as in the case
of the Tollmien-Schlichting waves, with the difficulty that
the phenomenon depends on the spectrum (or on the scale) of
tunnel turbulence, and furthermore on the amplification char-
acteristics of the perturbation along the path traveled by
the perturbation.
LAMINAR INSTABILITY AND TRANSITION

The preservation of laminar flow up to very large Reynolds numbers is of considerable practical interest. Experimental investigations of transition from the laminar to the turbulent regime have, therefore, been undertaken with a double purpose: To give an understanding of the mechanism of transition or at least of the breakdown of laminar motion, and to permit prediction of the occurrence of transition for a given practical case such as a certain airfoil. The first part of this problem has found a complete solution: Recent experiments (reference 1 and especially reference 2) confirmed, in general, the results of earlier theoretical investigations of Heisenberg, Tollmien, Schlichting, Görtler, and others. On the basis of this experimental and theoretical evidence, and also of earlier work of Taylor on the flow between rotating cylinders, there appears to be no doubt that laminar flow always becomes unstable for a sufficiently large Reynolds number. This critical Reynolds number will depend on the form of the laminar motion, for example, laminar boundary-layer flow, laminar pipe flow, and so forth, and on the type of disturbance, for example, two-dimensional waves or three-dimensional vortices. Instability in the sense used here means that for all Reynolds numbers above the critical one there will exist a range of, say, wavelengths or frequencies such that a disturbance of the proper frequency or wavelength will increase in amplitude.

Thus the mechanism of laminar instability is clear and the critical Reynolds number thus defined can be predicted. However, for practical purposes, it is desirable to predict a different Reynolds number; namely, the Reynolds number at which the flow becomes turbulent. There is a distinct difference between these two Reynolds numbers; if the first Reynolds number $R_1$, that is, the critical number in the sense of the small perturbation theory, is reached, amplification of certain disturbances begins. The second, or "practical" critical Reynolds number $R_2$ means that amplification of disturbances has already taken place and to such an extent that complete breakdown of the laminar motion occurs.

The critical Reynolds number $R_1$ is, in general, so low and the difference between $R_1$ and $R_2$ so large that for practical applications the prediction of $R_1$ only is of little use. For example, the laminar boundary layer along a flat plate in the absence of a pressure gradient becomes theoretically unstable at a Reynolds number based on the
distance from the leading edge, $R_x = 6 \times 10^4$. Experiments in low-turbulence tunnels (references 1 and 2) confirmed this number but showed also that actual transition occurred only at Reynolds numbers from $2 \times 10^6$ to $2.8 \times 10^6$. Hence, the larger part of the boundary layer is in most practical cases in an unstable state, that is, in a condition similar to that of a supercooled liquid or a supersaturated vapor. The Reynolds number $R_1$ corresponds in this analogy to the melting or condensation temperature $T_1$. The Reynolds number $R_2$ corresponds to the temperature $T_2$ at which under given experimental conditions the substance actually solidifies or condenses. Consequently $R_1$ is a definite number for a certain type of laminar motion, corresponding to the definite melting temperature of a certain substance. On the other hand, $R_2$ depends not only upon the type of laminar motion but also upon the initial disturbances present in the laminar layer and therefore on the experimental setup, for example, the free stream turbulence level. The amplification of these initial disturbances in the unstable region can be computed from the small perturbation analysis, provided the magnitude of these disturbances does not exceed the range of this linearized theory. Let $a(R_x)$ denote the ratio of the amplitude of a given oscillation at a Reynolds number $R_x$ to that of the same oscillation just upstream of the instability zone. The value $a(R_x)$ depends, of course, on the frequency of the oscillation (see, for example, references 1 and 2). It is now possible to compare $a(R_x)$ for various frequencies and obtain the maximum possible amplification $a(R_x)_{\text{max}}$ at a given Reynolds number $R_x$. Such a computation was carried out by Schlichting (reference 1) for flat-plate flow. Schlichting found in this case

$$a(R_x)_{\text{max}} = 0.55\,e^{0.555\times10^{-5}R_x} \quad (1)$$

Comparing his result with measurements of Hansen and Gebers, Schlichting noted that at transition the "most dangerous" frequencies had been amplified four to nine times. However, if the values of $R_x$ found in recent experimental work are inserted into equation (1), $a(R_x)$ becomes of the order of $10^5$. This point was emphasized by G. I. Taylor in 1938 (reference 8) and at that time was considered as strong evidence against the validity of the small-perturbation theory. This apparent difficulty in applying equation (1) is mainly caused by the lack of a clear definition of $R_2$, the Reynolds number of transition; thus, in order to bridge the gap between $R_1$ and $R_2$, it is first necessary to define $R_2$ in such a way that it can be related to the small perturbation theory and hence to $R_1$. 
The following definition appears to be reasonable:

\[ R_z \] is the Reynolds number at which the apparent shear
\[ \tau_A = -\rho \overline{u'v'} \] due to amplified boundary layer oscillations
at any point in the boundary layer becomes equal to the
laminar shear \[ \tau_L = \mu \frac{du}{dy} \] in the boundary layer." Beginning
with \( R_z \), certain disturbances will be amplified. The per-
turbation components \( u \) and \( v \) are correlated and con-
sequently give rise to an apparent shear. The magnitude of
this apparent shear increases as the amplitude of the oscil-
lations increases. When this apparent shear becomes of the
same order as the laminar shear of the undisturbed mean mo-
tion the mean profile can evidently no longer remain unaltered.
Hence, \( R_z \) defined in this way can be expected to be reason-
ably close to the Reynolds number at which the oscillation has
an effect on the mean flow sufficient to modify radically the
mean velocity profile.

The shear which exists in the undisturbed laminar motion
\( \tau_L \) can be computed as a function of \( R_x \) for any given case.
The apparent shear caused by amplified boundary layer oscil-
lations \( \tau_A \), is more difficult to compute since \( \tau_A \) depends
upon the initial disturbance in the laminar layer and the
amplification characteristics.

It is, however, possible to bring \( \tau_A \) into a form
where its general behavior as a function of \( R_x \) becomes
more evident and where in certain cases, at least the order
of magnitude of \( \tau_A \) and thus of \( R_z \) can be obtained by
crude approximations.

If the correlation coefficient \( k = \frac{\overline{u'v'}}{u'v'} \) and the
fluctuation levels \( \frac{u'}{U_0} \) and \( \frac{v'}{U_0} \) are introduced, \( \tau_A \) becomes

\[ \tau_A = -\rho \overline{u'v'} = -\rho k \frac{u'}{U_0} \frac{v'}{U_0} U_0^2 \]

It is furthermore convenient to introduce a factor \( b \) by
writing

\[ b \frac{u'}{U_0} = \frac{v'}{U_0} \]

and thus
\[
\frac{T_A}{\frac{1}{2} \rho U_o^2} = -2 k b \left( \frac{u'_i}{U_o} \right)^2
\]  

(2)

Using the amplification factor \( a(R) \), \( \frac{u'_i}{U_o} \) can be related to its initial value, \( \left( \frac{u'_i}{U_o} \right)_i \), just upstream of the amplification zone:

\[
\frac{T_A}{\frac{1}{2} \rho U_o^2} = -2 k b \left( \frac{u'_i}{U_o} \right)^2 \left[ a(R) \right]^2
\]  

(3)

Equation (3) is still quite general, \( T_A \) will, of course, vary across the boundary layer and its magnitude will also depend upon the oscillation frequencies and the Reynolds number. However, only the maximum value which \( T_A \) can reach at a certain Reynolds number is necessary for the present discussion, since an attempt was made to find the Reynolds number at which \( T_A \) first becomes equal to \( T_L \). Consequently,

\[
\left( \frac{T_A}{\frac{1}{2} \rho U_o^2} \right)_{\text{max}} = -2 \left\{ k b \left( \frac{u'_i}{U_o} \right)^2 \left[ a(R) \right]^2 \right\}_{\text{max}}
\]  

(4)

as function of the Reynolds number \( R_x \). Now, the distribution of \( T_A \) across the boundary layer has a maximum close to the so-called "critical layer" which is usually near the wall. Hence, for the transition criterion \( T_L \) shall be replaced, approximately, by its value at the wall. Thus, in terms of the usual laminar skin friction coefficient,

\[
\frac{T_L}{\frac{1}{2} \rho U_o^2} = c_{fL}
\]

and the ratio \( \frac{T_A \text{ max}}{T_L} \) becomes:
\[
\frac{T_{a \text{ max}}}{T_L} = \frac{2}{c_{fL}} \left\{ k \left[ \frac{u_1}{U_0} \right]^n \left[ a(R) \right]^m \right\}_{max} \tag{5}
\]

\[R_x \text{ is then determined by}
\[
- \frac{2}{c_{fL}} \left\{ k \left[ \frac{u_1}{U_0} \right]^n \left[ a(R) \right]^m \right\}_{max} = 1 \tag{6}
\]

**Application to Flat Plate Flow**

The results obtained by Schlichting (references 6 and 7) can now be used to evaluate equation (5) or (6) numerically. Equation (1)* gives \(a(R)\) max. The correlation coefficient \(k\) has been computed by Schlichting (reference 7) at one point on the lower branch and one point on the upper branch of the instability limit curve. Comparing the results at these two points it appears that \(k_{max}\) differs but little and thus the variation of \(k\) with Reynolds number is apparently very small. Since \(c_{fL}\) and especially \(a(R_x)\) vary rapidly with \(R_x\), it is sufficient to replace \(k(R)\) in equations (5) and (6) by a constant value. The maximum value \(k = -0.13\) found by Schlichting is thus used in equation (5). The same argument apparently holds for \(b\); in the neighborhood of the critical layer, \(b\) is of the order 0.1. This value also agrees with measurements (reference 1). The skin friction coefficient for the Blasius layer is given by

\[
c_{fL} = 0.664 R_x^{-\frac{1}{2}}
\]

Thus equation (5) finally becomes:

---

*The use of Schlichting's amplification function in the present analysis is not strictly correct; the oscillations are, of course, no longer "small perturbations" in the sense of the linear theory if \(T_A\) becomes of the order of \(T_L\). The error involved is, however, probably not very large and the order of magnitude of \(R_x\), which is of main concern here, will be unaffected.*
\[ \frac{T_A \max}{\tau_L} = 0.016 \left( \frac{u_1'}{U_0'} \right)^2 \sqrt{R_x} e^{1.1x\tau_0} R_x \]  \hspace{1cm} (7)

Figure 12 gives \( \frac{T_A \max}{\tau_L} \) plotted against \( R_x \) for different values of the initial fluctuation level. In considering the magnitude of the initial disturbance, it should be kept in mind that \( \left( \frac{u_1'}{U_0'} \right) \) is the fluctuation level with the most dangerous, that is, most highly amplified, frequency. Thus \( \left( \frac{u_1'}{U_0'} \right) \) is only a fraction of the total fluctuation level in the laminar layer upstream of the amplification zone.

Two facts are evident from figure 12.

(1) The slopes of the curves \( \frac{T_A \max}{\tau_L} \) versus \( R_x \) are very large in the neighborhood of \( \frac{T_A \max}{\tau_L} \). Hence \( R_x \) does not depend too much upon the exact value of \( \frac{T_A \max}{\tau_L} \) which is chosen to define transition.

(2) Quantity \( R_x \gg R_1 \) even with considerable initial fluctuation levels. (For example, with \( \left( \frac{u_1'}{U_0'} \right) = 1 \) percent, \( R_x = 10 R_1 \).)

The next step would logically involve relating the quantities entering equation (7) to factors known to influence transition. Especially the relation between \( \left( \frac{u_1'}{U_0'} \right) \) and the free stream turbulence level is very important.

At the present time only a few general considerations can be given in this connection and the main effects classified. It is hoped to obtain more complete relations in the future.

(1) Effect of free-stream turbulence.— Turbulence of the
free stream will affect the quantity \( \left( \frac{u^i}{U_0} \right)_i \) only. The effect will depend upon the free stream level \( \left( \frac{u^i}{U_0} \right)_f \) and upon the frequency distribution or upon the scale of turbulence. The relation between \( \left( \frac{u^i}{U_0} \right)_i \) and \( \left( \frac{u^i}{U_0} \right)_f \) is not simple and requires further study.

(2) Effect of pressure gradient. — The pressure gradient will affect at least two quantities in equation (3): namely, \( c_{fL} \) and the amplification function \( a(R) \). An adverse, that is, positive gradient causes a more rapid decrease of \( c_{fL} \) with Reynolds number to \( c_{fL} = 0 \) at separation, and as Schubauer and Skramstad (reference 2) found, a larger amplification \( a(R) \).

(3) Effect of roughness. — Small roughness elements, like free stream turbulence, will mainly affect \( \left( \frac{u^i}{U_0} \right)_i \). The value of \( \left( \frac{u^i}{U_0} \right)_i \) corresponding to a given roughness element will depend upon the element's height and also upon the width, since the latter influences the frequency distribution of the disturbance caused by the presence of the element.

CONCLUDING REMARKS

The results of measurements of boundary layer transition on the concave side of a plate with a 2.5-foot radius of curvature confirm the previous result that the mechanism leading to the breakdown of laminar flow differs essentially at convex and concave boundaries. Transition on convexly curved surfaces is due to the two-dimensional Tollmien-Schlichting type of instability as is the case with a flat plate. Transition on concave boundaries appears to result from the three-dimensional Görtler type of instability, and is thus related to the instability occurring in the flow between rotating cylinders.
No influence of pressure gradient on transition on the concave wall was found. The influence of a positive pressure-gradient on transition on convex boundaries is most pronounced in the neighborhood of zero pressure gradient. Free-stream turbulence appears to have about the same effect on the position of the transition point in concave and convex boundary layer flow.

The mean velocity profiles in the boundary layer of a concave plate with an effective radius of curvature around $10^{-3}$ was found to be very close to the Blasius flat-plate distribution.

A discussion of the breakdown mechanism of laminar flow shows the difference between the Reynolds number $R_{1}$ corresponding to the beginning of instability, in the sense of a small perturbation theory, and the Reynolds number $R_{2}$ corresponding to transition. The former number is well defined and can be predicted for a given case with good accuracy. The latter can be defined by the condition that at $R_{2}$ the maximum apparent shear due to the laminar boundary layer oscillations becomes equal to the laminar shearing stress at the wall for the undisturbed mean motion. A rough estimate of $R_{2}$ for the case of flat plate flow shows that even with considerable initial disturbances $R_{2}$ is much larger than $R_{1}$.

California Institute of Technology,
Pasadena, Calif., October 29, 1944.
REFERENCES


Figure 1: Flow diagram of the boundary layer research tunnel.
FIGURE 2. Test plate with 2.5 foot radius of curvature.
Figure 5. - Reynolds number of transition against the effective curvature on the concave side of the plate with 2.5-foot radius of curvature.

Figure 6. - Effect of curvature on transition. The arrows point to the value of $Re_{tr}$ found for the flat plate.
Figure 7. - Transition measurements on the concave side of the plate with 2.5-foot radius of curvature at three different turbulence levels of the free stream.

Figure 8. - Transition measurements on the concave sides of the plates of 2.5 and 20-foot radius of curvature.
Figure 9. - Pressure gradients on the convex side of the plate of 20-foot radius for which transition was investigated.

Figure 10. - Effect of the pressure gradient on transition for flow past the convex side of the plate with 20-foot radius of curvature.
Figure 11.— Effect of the pressure gradient on transition on the concave side of the plate with 2.9-foot radius of curvature.

Figure 12.— Transition criterion for flow along a flat plate with different intensities of the initial disturbance.