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# 8:1 Thermal Cavity Problem

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## Abstract

We present results for the 8:1 thermal cavity problem<sup>(1)</sup> using FIDAP on 3 meshes—each using 3 elements. A brief summary of related results is also included. *Keywords:* Finite elements, FIDAP, thermal convection, CFD

# 1. Introduction

This contribution comes via the rather versatile and general commercial finite element code, FIDAP<sup>(2)</sup>. This code still offers the user a wide selection with respect to element choices, statement of governing equations, (e.g., advective form, divergence form) implicit time integrators (variable-step or fixed step, first-order or second-order), and solution techniques for both the nonlinear and linear sets of equations. We have tested quite a number of these variations on this problem; here we report on an interesting subset and will present the remainder at the conference.

# 2. Methodology

Most of the results were obtained using the classical "plane vanilla" (and less expensive) Galerkin finite element method—no tricks, such as stability-enhancing upwind-related modifications to the advection terms—combined with an 'honest' non-dissipative second-order accurate time integrator: trapezoid rule<sup>(3)</sup> (TR). However, to demonstrate the often-deleterious effects of "stabilizing" modifications, we shall present some SUPG (Streamline-Upwind Petrov-Galerkin) results as well one from a highly-dissipative time integrator: backward Euler (BE).

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In the results to be summarized herein, two types of solvers were employed on the linear systems resulting after the successive substitution (Picard) method was applied to each nonlinear algebraic system: (1) After applying the penalty approximation method to eliminate the pressure (P=  $-\lambda \nabla \mathbf{u}$ ,), the *fully-coupled* ( $\mathbf{u}, \Theta$ ) systems were solved using an efficient form of Gaussian elimination (skyline method  $^{(2,3)}$ ). (2) The segregated solution method <sup>(2,3)</sup> was employed to generate an iterative sequence of smaller (uncoupled) linear systems (for u, v,  $\Theta$  & P, as well as one for a Lagrange multiplier), each of which is solved by an iterative method. The symmetric systems (P and the Lagrange multiplier) were solved using the SSOR-preconditioned CR (conjugate residual) method and the unsymmetrical ones  $(u, v, \Theta)$  via CGS (conjugate gradient squared), preconditioned with diagonal (Jacobi) scaling. Convergence criteria employed were as follows:  $\varepsilon_N = 10^{-7}$  for the outer (Picard) iterations and  $\varepsilon_L = 10^{-4}$  for the linear subsystems. The outer iterations typically converged in 3 - 5 iterations and the linear subsystems required 2-6 via CGS & 20-80 via CR. Sufficient testing assured us that our  $\varepsilon$ 's were sufficiently small—via both relative error and relative residual (Euclidean) norms; i.e.,  $||\Delta x||/||x|| < \varepsilon$  and ||R(x)||/|| $R(x_0) \| < \varepsilon$ , where  $R(x) \equiv Ax-b$ .

# 3. Results

In this 'short' presentation, we will show results from 3 elements on 3 grids—most via TR but with one using BE. The elements used were: (1)  $Q_1Q_0$  (bilinear velocity and temperature, piecewise-constant pressure on Quadrilaterals), (2)  $Q_2P_{.1}$  (biquadratic velocity and temperature, piecewise-linear pressure) and (3)  $Q_2Q_{.1}$ , (same as  $Q_2P_{.1}$  except pressure is piecewise-bilinear). Even though the first and third have some (div-) stability problems <sup>(3)</sup>, they produced excellent results and are still quite useful in general. The  $Q_2P_{.1}$  (9/3) element, while possibly the most popular higher-order element extant (at least

when using quadrilaterals), was often less accurate than  $Q_2Q_{-1}$ . This may be more important in 3D simulations, where neither of these higher-order elements has been adaquately tested/evaluated. Some<sup>(3)</sup> suspect that  $Q_2Q_{-1}$ , even though somewhat unstable, may be the winner in this race.

The results presented in Tables 1-3 are self-explanatory, with the possible need to explain one 'outlyer': The Q<sub>2</sub>P<sub>-1</sub> element performed poorly (low amplitudes) on Mesh 1, but recovered strongly on Mesh 2. A final caveat: All results herein were at very slightly different parameter values; Ra ~  $3.41 \times 10^5$  and Pr ~ 0.709. The correct values will be used prior to the conference, and reported there.

Figure 1 gives the time history of the temperature at Point 1 for  $Q_2Q_{-1}$  on Mesh 2. Figure 1a focuses on the developing time regime, showing the frequency beating during the early stages that gives way to a single frequency. Figure 1b shows the single frequency behavior at later times in the solution. Figure 2 shows the pattern of temperature variations with respect to the local time average. The dark regions have an instantaneous temperature less than the local mean while in the gray regions it is greater. The arrows track a single disturbance 'bubble' over one oscillation period as it propagates up the hot wall.

In addition to those in the tables, we report briefly a few more results:

 The following elements failed totally (i.e., they went to a steady, non-oscillating flow state).

- (i)  $Q_1Q_0$  on meshes 1 & 2 when streamline upwinding was employed. (Not a surprise, considering that the fluid dynamical *instability* is in the boundary layer *in the flow direction*.)
- (ii)  $Q_1Q_0$  on mesh 1 using mass lumping, caused by poor phase speed accuracy (and inaccurate group velocity).
- (iii)  $Q_2Q_1$  (pressure is bilinear *continuous*) on mesh 1; also not a surprise see Ref. 3.
- (iv) Backward Euler for  $Q_1Q_0$  on Mesh 1 using the same  $\Delta t$  that succeeded for TR (~25 steps/period).
- The(less-expensive) advective form (e.g., u.∇ Θ) was generally more accurate than either the conservation form [e.g., ∇.(uΘ)] or a quadratically-conserving (skewsymmetic) form— an average of the first two.
- Q<sub>1</sub>Q<sub>0</sub> seems to converge 'from above'; e.g.., the amplitudes of the oscillations are too large, whereas all three 'Q<sub>2</sub>' elements seem to mostly converge from below—and, of course, faster.

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		Mesh 1			Mesh 2			Mesh 3			
		Grid resolution: 27x121			Grid resolution: 53x241			Grid resolution: 105x481			
		Steps / period: ~25			Steps / period: ~25			Steps / period: ~25			
		Avg.	Amp.	Period	Avg.	Amp.	Period	Avg.	Amp.	Period	
		05(05	02026	2 4592	059(1	02822	2 4241	05(05	00705	2 4270	
u	$Q_1Q_0$	.05005	.02920	3.4383	.05801	.02822	3.4341	.03003	.02785	2 4250	
	$Q_2P_1$	.05240	.002/1	3.4245	.05/03	.02703	3.4203	.03003	.02774	2 4250	
-	$Q_2Q_{-1}$	.05601	.02597	3.4285	.05088	.02782	3.4201	.05554	.02774	3.4239	
V <sub>1</sub>	$Q_1Q_0$	.46189	.04123	3.4582	.4651	.03969	3.4342	.46163	.03915	3.4279	
	$Q_2P_1$	.46409	.00420	3.4428	.4631	.03877	3.4265	.46251	.03900	3.4259	
	$Q_2Q_{-1}$	.46233	.03658	3.4285	.4627	.03912	3.4261	.46074	.03900	3.4259	
$\Theta_1$	$Q_1Q_0$	.26385	.02291	3.4582	.2664	.02197	3.4341	.26482	.02166	3.4279	
	$Q_2P_{-1}$	.26590	.00221	3.4429	.2658	.02144	3.4265	.26547	.02156	3.4259	
	$Q_2Q_{-1}$	.26590	.02025	3.4286	.2651	.02162	3.4261	.26468	.02158	3.4259	
$\epsilon_{12}$	$Q_1Q_0$	0			0	-		0			
	$Q_2P_1$	0			0	-		0			
	$Q_2Q_{-1}$	0	-	-	0	-	-	0	-	-	
$ \Psi_1 $	$Q_1Q_0$	07293	.00369	3.4582	07397	.00360	3.4341	07450	.00356	3.4276	
	$Q_2P_{-1}$	07337	.00035	3.4414	07398	.00353	3.4264	07444	.00354	3.4259	
	$Q_2Q_{-1}$	07218	.00311	3.4286	07409	.00355	3.4261	07439	.00355	3.4259	
ω <sub>1</sub>	$Q_1Q_0$	-2.2379	.5764	3.4581	-2.3428	.5399	3.4341	-2.4144	.5388	3.4279	
	Q <sub>2</sub> P <sub>-1</sub>	-2.4106	.0536	3.4414	-2.4240	.5431	3.4266	-2.4498	.5408	3.4259	
	$Q_2Q_{-1}$	-2.2513	.5132	3.4285	-2.4190	.5465	3.4257	-2.4455	.5405	3.4259	
$\Delta P_{14}$	$Q_1Q_0$	00152	.01043	3.4582	00193	.01045	3.4340	00326	.01033	3.4280	
	$Q_2P_{-1}$	00182	.00113	3.4412	00125	.01040	3.4264	00219	.01034	3.4262	
	$Q_2Q_{-1}$	00135	.00987	3.4285	00200	.01048	3.4259	00203	.01034	3.4259	
$\Delta P_{51}$	$Q_1Q_0$	5337	.01119	3.4582	5332	.01150	3.4343	5323	.01146	3.4280	
	$Q_2P_{-1}$	5342	.00131	3.4414	5348	.01149	3.4266	5348	.01146	3.4262	
	$Q_2Q_{-1}$	5360	.01086	3.4286	5338	.01157	3.4260	5349	.01146	3.4260	
$\Delta P_{35}$	$Q_1Q_0$	.5362	.00505	3.4581	.5354	.00513	3.4341	.5357	.00510	3.4283	
	$Q_2P_{-1}$	.5360	.00060	3.4422	.5360	.00513	3.4266	.5370	.00511	3.4261	
	$Q_2Q_{-1}$	.5373	.00491	3.4286	.5358	.00517	3.4261	.5370	.00510	3.4257	

Table 1; Point 1 Data

		Mesh 1			Mesh 2			Mesh 2		
		Grid resolution: 27x121			Grid resolution: 53x241			Grid resolution: 105x481		
		Gr. 1 1 05			a			St. ( 1 1 25		
		Steps / period: ~25			Steps / period: ~25			Steps / period: ~25		
		Avg.	Amp.	Period	Avg.	Amp.	Period	Avg.	Amp.	Period
-Nu <sub>x=0</sub>	$Q_1Q_0$	4.5661	3.88e-3	3.4582	4.5796	3.70e-3	3.4343	4.5821	3.63e-3	3.4279
	$Q_2P_{-1}$	4.6318	4.30e-4	3.4410	4.5893	3.62e-3	3.4265	4.5825	3.61e-3	3.4259
	$Q_2Q_{-1}$	4.6328	3.70e-3	3.4286	4.5893	3.65e-3	3.4260	4.5821	3.61e-3	3.4258
-Nu <sub>x=W</sub>	$Q_1Q_0$	4.5661	3.88e-3	3.4582	4.5796	3.70e-3	3.4343	4.5821	3.63e-3	3.4263
	$Q_2P_{-1}$	4.6318	4.39e-4	3.4421	4.5888	3.62e-3	3.4265	4.5825	3.61e-3	3.4259
	$Q_2Q_{-1}$	4.6328	3.70e-3	3.4286	4.5893	3.65e-3	3.4261	4.5821	3.61e-3	3.4258

Table 2; Nusselt Numbers

	Avg.	Amp.	Period	Δυσ	<u>, 25</u>	1	bieps / p	0110 <b>u</b> . 25	
		1		Avg.	Amp.	Period	Avg.	Amp.	Period
$P_1 Q_0$ $P_2 P_{-1}$	.2396 .2393	2.21e-5 2.99e-6	3.4582 3.4336	.2396 .2397	1.78e-5 1.69e-5	3.4335 3.4265	.2397 .2397	1.73e-5 1.71e-5	3.4271 3.4254
$P_2Q_{-1}$	.2396	1.817e-5	3.4286	.2397	1.69e-5	3.4250	.2397	Amp. 1.73e-5 1.71e-5 1.70e-5 .00161 .00161	3.4271
$P_{1}$	3.0188	.000155	3.4419	3.0180	.00160	3.4275	3.0179	.00161	3.4258
	$\frac{Q_{-1}}{Q_0}$ $\frac{Q_0}{P_{-1}}$ $\frac{Q_{-1}}{Q_{-1}}$	$\begin{array}{c cccc} Q_{-1} & .2395 \\ Q_{0} & 2.8728 \\ P_{-1} & 3.0188 \\ Q_{-1} & 3.0171 \end{array}$	$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$				

Table 3; Mean Quantities

Figure 1. The temperature at Point 1. (a) the early stages of flow development showing the frequency beating. (b) the latter stages of the solution, after stabilization, showing single frequency behavior.

Figure 2. Patterns of the instantaneous temperature variation from the local time averaged mean; time interval between plots is (approximately) 1/6 of one period. In dark regions the temperature is less than the local mean, in gray regions it is greater.



Figure 1a



Figure 1b



Figure 2

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