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NUMERICAL COMPUTATION OF DYNAMIC STRESS INTENSITY FACTORS BY A LAGRANGIAN FINITE-DIFFERENCE METHOD (THE HEMP CODE)

Abstract

Numerical computation of the dynamic stress intensity factor for a centrally cracked rectangular bar, subjected to uniaxial tensions in plane strain geometry with Heaviside-function time dependence, is carried out using the timedependent Lagrangian finite-difference code, HEMP. Excellent agreement is found with results obtained from other methods. This demonstrates the capability and the reliability of the HEMP code as an extremely useful numerical method for solving dynamic problems in fracture mechanics. Also discussed is the dynamic behavior of the stress intensity factor and crack opening.

Introduction

The knowledge of the stress field in the vicinity of a crack is a prerequisite to the prediction of fracture and failure of the material. In order for the crack to propagate, the stress at the crack tip must exceed some critical value. For brittle materials, the linear elasticity model usually gives a reasonably good description of the macroscopic phenomena of fracture in solids. However, for this model it is well known that regardless of the magnitude of the applied load the stress at the tip of a crack may have a singularity of $O(r^{-1/2})$, where r is the radial distance from the crack tip to the point of observation, 1-4 To overcome the analysis difficulty that an infinite stress assumption causes, the stresses in the neighborhood of the crack tip are

normalized by multiplying them by a function $O(r^{1/2})$, yielding zero at the crack tip. This leads to the definition of the stress intensity factors.³ For dynamic problems, the dynamic stress intensity factor is defined in the same manner as for the static case. The only difference is that the dynamic stress intensity factor is a function of time t.

Recently, considerable research has been done on the determination of the dynamic stress fields around a finite crack in an infinite elastic solid.⁵⁻¹⁵ However, the quasi-numerical methods used are incapable of solving dynamic problems with finite boundaries. On the other hand, the finite element method can be used to solve problems in fracture mechanics with finite boundaries provided

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the loads are static. 16-20Actually. finite element methods can be used for dynamic problems by applying a series of incremental static loads to approximate time variations. 21-23 This approach requires inordinate amounts of computer time for two- and three-dimensional problems when a large number of elements are used, because a large set of simultaneous linear algebraic equations must be solved at each time step. It becomes especially inefficient when large matrices occur as in calculations involving three space dimensions. The work presented here is concerned with the solution of problems involving dynamic fracture with finite boundaries using finite difference methods, which avoid the difficulties of the finite element method.

The main mathematica' tool used is a Lagrangian finite-difference computer program, the HEMP code. The general program can be used to solve problems in three space dimension and time, ²⁴⁻²⁶ In this paper we demonstrate the capability and the reliability of the HEMP code for solving dynamic problems in fracture mechanics by applying it to a simple twodimensional but nontrivial dynamic problem.

For simplicity, a centrally cracked rectangular bar in plane strain geometry is considered. A uniaxial tension with a Heavisidefunction time dependence is applied. The details of the example are given in the next section, and the numerical results are presented graphically in the third section. Finally, in the last section, a detailed discussion of the numerical results is given

Example Problem

Consider a rectangular bar with a centrally located crack shown in Fig. 1. It is loaded dynamically in the axial direction by a uniform tension P(t) with Heaviside-function time dependence (Fig. 2). The boundary conditions given



Fig. 1. Geometry of the bar in the example problem; a = 0.24 cm.



Fig. 2. Load curve assumed in example problem.

correspond to plane strain loading. The bar is assumed to be a linear elastic material with the following characteristics:

Shear modulus G = 0.76923 Mbar, Bulk modulus K = 1.66667 Mbar, Poisson's ratio ν = 0.3, Density ρ = 5 g/cm³.

Hence the longitudinal wave speed is

$$v_{\rm L} = \sqrt{\frac{K + \frac{4}{3}G}{\rho}} \approx 0.734 \text{ cm}/\mu\text{sec}, \qquad (1)$$



Fig. 3. Zoning of the quarter of the bar treated in the calculation.

the transverse wave speed is

$$v_{\rm T} = \sqrt{\frac{G}{\rho}} \approx 0.392 \, {\rm cm}/\mu {\rm sec},$$
 (2)

and the Rayleigh wave speed²⁷ is

$$v_{\rm R} \approx 0.495 v_{\rm L} \approx 0.363 \, {\rm cm}/\mu sec.$$
 (3)

By symmetry it is sufficient to solve the quarter problem, as shown in Fig. 3. A grid of 50 zones/cm was used.

Numerical Results

Due to the inability of any finitedifference method to represent the extremely steep stress gradients that occur in the neighborhood of a crack tip, a subroutine for extrapolation of the mode I dynamic stress intensity factor $k_1(t)$ from the stress field in the vicinity of the crack tip is used here. The procedure uses the stress field equations in the neighborhood of the crack tip (r = 0), assuming an iminite elastic medium,

$$\sigma_{\rm im}(t) \approx \frac{k_1(t)}{(2\pi r)^{1/2}} f_{\rm im}(\theta), \qquad (4)$$

where

$$\begin{split} \mathbf{f}_{\mathbf{xx}}(\theta) &= \left(1 + \sin\frac{\theta}{2}\sin\frac{3\theta}{2}\right)\cos\frac{\theta}{2}, \\ \mathbf{f}_{\mathbf{yy}}(\theta) &= \left(1 - \sin\frac{\theta}{2}\sin\frac{3\theta}{2}\right)\cos\frac{\theta}{2}, \\ \mathbf{f}_{\mathbf{xy}}(\theta) &= \mathbf{f}_{\mathbf{yx}}(\theta) = \sin\frac{\theta}{2}\cos\frac{\theta}{2}\cos\frac{3\theta}{2}. \end{split}$$

From (4) we define for every time step

$$k_{1}^{*}(n\Delta t) = \frac{(2\pi r_{jk})^{1/2}}{f_{im}(\theta_{jk})} \sigma_{im}(n\Delta t).$$
 (5)

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r_{ik}. Our particular extrapolation path along the straight line x = 0.01 cm is the closest to the best path (see Appendix). Nexi, these discrete data are fitted by least-squares to a straight line,

$$k_1''(n\Delta t) = k_1(n\Delta t) + \beta(n\Delta t)r, \qquad (6)$$

where $k_{1}(n\Delta t)$ is the approximate value of the mode I dynamic stress intensity factor at time $n\Delta t$. The error for this kind of straight-line extrapolation is $O(r^2)$ (see Appendix). Note that the data for two to three zones nearest to the crack tip are discarded in the least-squares curve fitting. Extrapolations along lines in several directions were also made.





The final values agreed to within 5%. A typical plot of (5) and (6) is shown in Fig. 4. The normalized mode I dynamic stress intensity factor $\tilde{k}_1(t) = k_1(t) / P(t)(\pi a)^{1/2}$ is plotted against t in Fig. 5. Also, the geometry of the crack opening for several time steps is shown in Fig. 6. A σ_{xx} contour plot is given in Fig. 7. The above subroutine has been used successfully by others. 17,28



Fig. 7.

A typical σ_{xx} contour (retraced from computer plot).

Discussion

If there were no crack in the rectangular bar, the stress field in the bar due to the dynamic loading P(t) of Fig. 2 would be a standing wave centered around the appropriate static value, However, in the presence of the crack there are many oscillations in the \tilde{k}_1 (t)-vs-t curve. These oscillations are due to scattering

phenomena from the crack tip and the boundary surfaces.

On the time axis of Fig. 5, I, denotes the numerically computed (HEMP code) time needed for the incident longitudinal wave to travel to the crack from the bar ends. Now, the following times are computed from the wave velocities, Eqs. (1),

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(2), and (3), and the true geometric distances: $(R_1 - I_1)$ denotes the time needed for the Rayleigh wave, generated by the initial incident waves, to travel between the two crack tips; $(P_1 - I_1)$ denotes the time needed for the scattered longitudinal wave, generated by the initial incident waves, to tra from a crack tip to the nearest boundary surface of the bar and back to the same tip; similarly, $(S_1 - I_1)$ corresponds to the scattered transverse wave; I, denotes the time needed for the incident longitudinal waves to travel the length of the bar, reflect from the boundary surface on the opposite side, and then travel back to the crack; analogous definitions are given to $(R_2 - I_2)$, $(P_2 - I_2)$, and (S2 - I2) respectively, generated by these secondly incident longitudinal waves. Hence, with the help of these time marks one can identify the oscillations in the $\widetilde{k}_1(t)\text{-}vs\text{-}t$ curve as being caused by the cancellation and the reinforcement of the incident waves by various scattered waves. The σ_{xx} -vs-t curve (Fig. 8) at



Fig. 8. The σ_{XX} -vs-t curve at (x, y) = (0.01 cm, 0.37 cm).

(x, y) = (0,01 cm, 0.37 cm) has the same general behavior as the \tilde{k}_1 -vs-t curve except for the extra peak at $t \approx 3 \mu \text{sec}$ due to the scattered longitudinal wave traveling directly from the nearest crack tip to the point (0.01 cm, 0.37 cm).

The normalized static stress intensity factor for the same problem but with static load of 4×10^{-3} Mbar at both ends of the bar can be estimated with reasonable accuracy, if the maxima and minima of $\tilde{k}_1(t)$ are not significantly altered by the dynamic oscillations. We believe that this is the case for our example. The simplest way to obtain \tilde{k}_1 (static) is

$$\widetilde{k}_{1}(\text{static}) = \frac{1}{2} [\text{Max } \widetilde{k}_{1}(t) - \text{Min } \widetilde{k}_{1}(t)]$$
$$- [\text{Min } \widetilde{k}_{1}(t)] H (-\text{Min } \widetilde{k}_{1}(t)).$$

where H is the unit Heaviside function, and Max f(:) and Min f(t) are the first global maximum and the first global minimum of f(t) respectively. Hence from Fig. 5 and the above formula, \tilde{k}_1 (static) = 1.095, which is about 5.5% larger than the more accurate theoretical solution,²⁹ \tilde{k}_1 (static) = 1.038. Since the dynamic \tilde{k}_1 has already incorporated an extrapolation, a more accurate method to obtain the static \tilde{k}_1 would be to use the calculated σ_{xx} (t) a few zones away from the tip. Thus we obtain k_1 (static) as follows: First obtain σ_{xx} (static) from Fig. 8.

$$\sigma_{XX}(\text{static}) = \frac{1}{2} [\text{Max } \sigma_{XX}(t) - \text{Min } \sigma_{XX}(t)]$$
$$- [\text{Min } \sigma_{XX}(t)] \cdot \text{H}(-\text{Min } \sigma_{XX}(t))$$
$$= 5.70 \times 10^{-3} \text{ Mbar.}$$

This stress value corresponds to t_s = 9.22 μ sec. It is seen in Fig. 5 that for

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t = t_s, \tilde{k}_1 (static) = 1.028, which is about 1% less than the value given in Ref. 29. Thus we see that both the dynamic and static values of the stress intensity factor can be calculated with the timedependent HEMP code.

It is important to notice that the dynamic stress intensity factors for a finite crack in an infinite elastic solid are found to be 15-30% greater than the analogous static value.^{13,15} These phenomena are attributed to the scattered Rayleigh waves. If there were only one incident plane wave in our example, then due to the superposition principle in linear elasticity theory,

$$[Max \tilde{k}_{1}(t)]_{one wave}$$
$$= \frac{1}{2}[Max \tilde{k}_{1}(t)]_{two waves} = 1.35,$$

which is about 30% larger than the analogous static value. But for the two-wave case, the overshooting is about 160%. This shows that in a practical situation the overshooting of the dynamic stress intensity factor is attributed mainly to



Fig. 9. The u-vs-t curve at the middle of the crack surface.

the particular geometry and the manner of loading.

The geometry of the crack opening and the opening and the closing of crack surfaces as functions of t are the natural products of the HEMP code calculation. They are plotted in Figs. 6 and 9, respectively. In general, the crack opens or closes when the crack tip experiences tension or compression, respectively.

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Appendix Error Analysis

In the neighborhood of the crack tip, assuming an infinite elastic medium,

$$\sigma_{xx}(r,\theta) = k_1 f_{-1/2}(\theta) r^{-1/2} + f_0(\theta) + f_{1/2}(\theta) r^{1/2} + f_1(\theta) r$$

+ $f_{3/2}(\theta) r^{3/2} + f_2(\theta) r^2 + O(r^{5/2}),$ (A-1)

where

$$f_{-1/2}(\theta) \neq 0$$
, for $0 \le |\theta| < \pi$.

It is well known³⁰ that the angular functions $f_n(\theta)$, n = 0, 1, 2, ..., have the special property

$$f_n(0) = 0, n = 0, 1, 2, \dots$$
 (A-2)

If we define

$$k_{1}^{*}(\mathbf{r},\theta) = \frac{\mathbf{r}^{1/2}}{f_{-1/2}(\theta)} \sigma_{\mathbf{x}\mathbf{x}}(\mathbf{r},\theta),$$
 (A-3)

then (A-1) and (A-2) yield

$$k_{1}^{(i)}(\mathbf{r},\theta) = k_{1} + [f_{-1/2}(\theta)]^{-1} \left[f_{0}(\theta) \mathbf{r}^{1/2} + f_{1/2}(\theta) \mathbf{r} + f_{1}(\theta) \mathbf{r}^{3/2} + f_{3/2}(\theta) \mathbf{r}^{2} + f_{2}(\theta) \mathbf{r}^{5/2} + O(\mathbf{r}^{3}) \right].$$
(A-4)

It is obvious that the best way to obtain higher accuracy is to choose the extrapolation path so that most of the f's vanish. In this particular case, the extrapolation path is along $\theta = 0$. From (A-2) and (A-4),

$$k_{1}^{*}(\mathbf{r}, \mathbf{0}) = k_{1} + [f_{-1/2}(\mathbf{0})]^{-1} \left[f_{1/2}(\mathbf{0})\mathbf{r} + f_{3/2}(\mathbf{0})\mathbf{r}^{2} + O(\mathbf{r}^{3}) \right].$$
(A-5)

Hence a straight-line fit of $k_1^*(r, 0)$ will have an error $O(r^2)$, and a quadratic fit of $k_1^*(r, 0)$ will have an error $O(r^3)$.

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